

FINITE ELEMENT ANALYSIS PROGRAM (FEAP)  
FOR CONDUCTION HEAT TRANSFER

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# NAVAL POSTGRADUATE SCHOOL

## Monterey, California



# THESIS

FINITE ELEMENT ANALYSIS PROGRAM (FEAP)

FOR CONDUCTION HEAT TRANSFER

by

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Finite Element Analysis Program (FEAP)  
for Conduction Heat Transfer

by

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## ABSTRACT

The Finite Element Analysis Program (FEAP) was expanded to solve linear and nonlinear, two and three dimensional heat conduction problems. The usual types of boundary conditions, including radiation, may be specified. A wide range of two- and three-time level schemes for the solution of time dependent problems is available in the program and a discussion of those most commonly used is presented. The algorithms for the solution of typical practical problems are described and several numerical examples are presented. The results are compared with the available analytical solutions. A listing of this expanded version of FEAP and the corresponding user's instructions are provided.



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## 1. INTRODUCTION

This thesis describes an application of the Finite Element method to Heat Transfer analysis through the use of a computer program. This program was designed to solve linear and nonlinear, steady and unsteady, two and three dimensional heat conduction problems involving temperature dependent thermophysical properties and complicated radiation/convection boundary conditions.

The Finite Element Analysis Program (FEAP), programmed by Professor R. L. Taylor in the Department of Civil Engineering of the University of California, Berkeley, served as the point of departure for the present code. It has now been expanded with two additional modules for Heat Transfer analysis and time integration of first order equations, respectively, but the original characteristics of a research and educational tool in which the various modules can be changed or added to as desired, were maintained.

Although all equations are retained in core, the program can handle realistic engineering problems with several hundred unknowns in most computer systems.

While the algorithms to solve steady heat transfer problems are well studied and defined, more research has to be done on the solution of unsteady problems. The Zienkiewicz two- and three-time level schemes were integrated in the program in such a way that a wide range of



choices of time integration algorithms is available to the user. A limited study was performed in order to determine the characteristics of the most used members of those families of numerical schemes.

The FEAP code is written in FORTRAN IV language and this version was constructed and tested on an IBM 360/67 computer system with OS/67 release 18. Other systems and installations will require modifications.



## II. STATEMENT OF OBJECTIVES

### A. EQUATION OF CONDUCTION. INITIAL AND BOUNDARY CONDITIONS

The problem considered for solution is thoroughly developed by Arpaci in Ref. 1 and is mathematically described in the region  $\Omega$  by the equation

$$\rho \cdot c \frac{\partial T}{\partial t} = \bar{\nabla} \cdot (\bar{K} \bar{\nabla} T) + Q \quad (1)$$

subjected to boundary conditions

$$T = T_w \quad \text{on} \quad \Gamma_1 \quad (2)$$

and

$$(\bar{K} \bar{\nabla} T) \cdot \bar{n} + q + q_c + q_r = 0 \quad \text{on} \quad \Gamma_2 \quad (3)$$

and initial condition

$$\lim_{t \rightarrow 0} T = T_0 \quad (4)$$

$\bar{\nabla}$  is the gradient operator,  $\Gamma_1$  and  $\Gamma_2$  are mutually exclusive parts of the boundary of the region  $\Omega$ ,  $T$  is the temperature and  $t$  is the time. The thermal capacity  $\rho c$ , the thermal conductivity  $\bar{K}$  and the rate of internal heat generation  $Q$  are thermophysical properties dependent on temperature.

In equation (3)  $\bar{n}$  is an outward unit vector normal to the boundary surface while  $q$ ,  $q_c$ ,  $q_r$  represent respectively the imposed heat flux and the rates of heat flow per unit area due to convection and radiation defined as

$$q_c = h_c (T - T_{ac}) \quad (5)$$

and



$$q_r = F\sigma (T^4 - T_{ar}^4) = h_r (T - T_{ar}) \quad (6)$$

In (5)  $h_c$  is the temperature dependent convective heat transfer coefficient. In (6) the parameter  $h_r$  is defined by the expression

$h_r = F\sigma (T^2 + T_{ar}^2) (T + T_{ar})$ , where  $F$  is the radiant exchange factor and  $\sigma$  the Stefan-Boltzman constant.  $T_{ac}$  and  $T_{ar}$  are the equilibrium temperatures for which, respectively, no convection and radiation occurs.

## B. NUMERICAL SOLUTION. DISCRETIZATION BY THE GALERKIN METHOD. MATRIX FORMULATION

The spacewise discretization of equation (1) using cartesian coordinates can be accomplished by Galerkin's principle as shown by Zienkiewicz in Ref. 2 and Lew in Ref. 3.

Let the unknown function  $T$  be approximated, throughout the solution domain at any time  $t$ , by the relationship

$$T = \sum_{i=1}^n N_i(x,y,z) T_i(t) = \langle N \rangle \{T\} \quad (7)$$

where  $N_i$  are the usual shape functions defined piecewise element by element,  $T_i$  being the nodal parameters.

The result is

$$[K] \{T\} + [C] \{\dot{T}\} + \{F\} = \{0\} \quad (8)$$

where  $[K]$  and  $[C]$  are symmetric matrices defined, on the element level, as

$$[K]^e = \int_{\Omega} e \left( \left\langle \frac{\delta N}{\delta x} \right\rangle^T \left\langle \frac{\delta N}{\delta x} \right\rangle k_x + \left\langle \frac{\delta N}{\delta y} \right\rangle^T \left\langle \frac{\delta N}{\delta y} \right\rangle k_y + \left\langle \frac{\delta N}{\delta z} \right\rangle^T \left\langle \frac{\delta N}{\delta z} \right\rangle k_z \right) d\Omega + \int_{\Gamma_2} e (h_c + h_r) \langle N \rangle^T \langle N \rangle d\Gamma \quad (9)$$



$$[C]^e = \int_{\Omega^e} \rho c \langle N \rangle^T \langle N \rangle d\Omega \quad (10)$$

The load vector  $F$  is

$$\{F\}^e = - \int_{\Omega^e} \langle N \rangle^T Q d\Omega + \int_{\Gamma_2^e} \langle N \rangle^T (q - h_c T_{ac} - h_r T_{ar}) d\Gamma \quad (11)$$

where the surface integrals are performed only over the surfaces where the prescribed boundary condition applies.

It must be noted that the set of equations (8) is nonlinear since the matrices  $[K]$ ,  $[C]$  and vector  $\{F\}$  are dependent on  $T$ .

The system of equations (8) may now be solved by any automated numerical technique. Code FEAP was written for this purpose and is described in the rest of this thesis.



### iii. FINITE ELEMENT ANALYSIS PROGRAM

The Finite Element Analysis Program (FEAP) was programmed and published by Taylor in Ref. 4 where it is well explained. The reader should consult this reference for details about it.

This thesis is an expansion of the original program towards the solution of Heat Conduction problems and as such only the main features of the basic program will be referred in this section. A more detailed explanation is reserved for the new subroutines added to the program. The complete user's instructions for FEAP are given in Appendix A and they complement this section.

#### A. MAIN PROGRAM

FEAP can be separated into two basic parts:

- a) Data input module and preprocessor
- b) Solution and output modules

The data input module must transmit sufficient information to the other modules so that each problem can be solved. All the input data is stored in a single array (integer array M) which is partitioned to store all the data arrays, as well as the global arrays, e.g., stiffness, mass, load, etc.

The total capacity of the program is controlled by the dimension of the array M in the blank common of the main



program and the corresponding value of the variable MAX.

The partition of  $M$  is performed in the control routine (subroutine PCONTR) and the data used for it is supplied by the user in the title and control information cards (first two cards for every run of the program).

Then the nodal coordinates, element connections, material properties, nodal loading and boundary conditions may be input using the macro control statements. They are interpreted by subroutine PMESH and each control is a function independent of the others.

To clarify what was said till this point, the discussion of a simple example may be useful. Consider the mesh presented in<sup>1</sup>Figure 1 and assume that quadratic rectangular elements as the one defined in Figure 2 are used. This example is completely solved in Appendix B and will be used throughout this text.

The sector of a circular ring was divided in six elements and the 33 nodes were numbered. The order of numbering is not crucial but in order to improve the profile of non-zero coefficients the following general rule should be used.

The numbering should be such as to minimize the nodal difference for each element (maximum node number minus minimum node number).

The user can now proceed to the preparation of data for the program. The first step consists of specifying the problem title and the control information. The latter is

<sup>1</sup>Figures are grouped at the end pp 47-79



clearly 33 nodes, six elements, one material set, two spatial dimensions and eight nodes per element. If this is a Heat Transfer problem we have one unknown per node (temperature).

The program expects now the data cards for mesh description. Any analysis require at least

- a) coordinate data which follows the macro command COOR
- b) connectivity table which follows the macro command

ELEM

- c) material data which follows the macro command MATE

In addition, most analysis will require specification of nodal boundary restraint conditions, macro BOUN, and the corresponding nodal force or displacement values, macro FORC. The term displacement refers to the value of the unknown for each specific problem. In a Heat Transfer problem the unknown is temperature while in an Elasticity analysis it is a geometrical displacement.

The TEMP macro command (temperature data) shall not be used in a Heat Transfer analysis to specify nodal temperature. It may be used to specify auxiliary nodal quantities.

If in our problem we assume the line defined by nodes 31, 32 and 33 at the temperature of 20, the line defined by nodes 1, 2 and 3 at 820 and insulated elsewhere, the macro control statements and correspondent data cards will be

COOR

1 5 .1



31		.3	
4	5.116666667		
29	.283333333		
2	5 .1	3.	
32	.3	3.	
3	5 .1	6.	
33	.3	6.	
5	5.116666667	6.	
30	.283333333	6.	

(blank card)

POLA

1	33	1
---	----	---

(blank card)

ELEM

1	1	1	6	8	3	4	7	5	2	5
---	---	---	---	---	---	---	---	---	---	---

(blank card)

BOUN

1	1	-1
3		-1
31	1	-1
33		-1

(blank card)

FORC

1	1	820.
3		820.
31	1	20.
33		20.



(blank card)

MATE

1 2

10. 10. 8000. 250. 3

END

In spite of the fact that the MATE command must be included in this part of the data preparation, its discussion is left to the section where the element modules are treated.

After completing the mesh data input, we are ready to initiate the problem solution.

FEAP has modules for variable algorithm capabilities and which, if necessary, can be modified or expanded. The basic aspect of the variable algorithm program is a macro instruction language which can be used to construct modules for specific algorithms as needed. This language is interpreted by subroutine PMACR and the complete list of macro instructions available in this version of FEAP is given in Appendix A.3.

Here, as an example, we will discuss in detail the algorithm for the solution of linear steady state problems using the original explanation given in Ref. 4.

To use the macro programming commands, the user only needs to learn the mnemonics of the language. If one wishes to form the global stiffness matrix the program instruction TANG is used (TANG is the mnemonic for a symmetric tangent stiffness matrix and for nonlinear elements would form and



assemble into the global stiffness the element tangent stiffness computed about the current displacement state; for linear elements this is just the linear stiffness matrix). For a problem with an unsymmetric tangent stiffness the macro command UTAN is used.

If one wishes to form the right hand side of the equations modified for specified displacements one uses the program instruction FORM. The resulting equations are solved using the instruction SOLV.

Printed output can be obtained using the macro command DISP.

The above instructions are sufficient to solve linear steady state problems, that is, the macro instructions

TANG ( or UTAN )

FORM

SOLV

DISP

are precisely the required instructions to solve any linear steady state problem.

If the same block of instructions is repeated twice or more, considerable effort is wasted in preparing the macro instruction data. To rectify this, looping commands are introduced as the instruction pair

LOOP        n

·

·

·

NEXT

which indicates that looping over all instructions between



LOOP and NEXT will occur n times.

Many other classes of problems can be solved using the macro instruction list given in Appendix A. With the above short explanation and the discussion presented in section IV for Heat Transfer problems, the reader will be able to construct his own algorithms using the macro programming language.

## B. FINITE ELEMENT SOLUTION MODULES

Most of FEAP is common to a wide range of problems, but some of the computations are linked to the type of problem to be solved. It is clear that if one wishes to solve an Elasticity problem, the calculation of the stiffness matrix will be different from that used for a Heat Transfer problem. It is also clear that differences also arise if one is using triangular elements instead of quadratic isoparametric elements as a way of describing the continuum.

The program was designed such that all computations associated with any type of element are contained in an element subroutine called ELMTnn where nn is between 01 and 05 in this version of FEAP. Each element type to be used is specified as part of the material property data, following macro control MATE.

Since the objective of this work is the solution of Heat Transfer problems, the element modules discussed here are the two and three dimensional heat transfer modules named ELMT02 and ELMT03.



## 1. Two Dimensional Heat Transfer Element

Subroutine ELMT02 is accessed using the number two in column ten of each material number card.

According to the value of the variable ISW, defined in subroutine PMACR, a specific function is performed. If ISW=1 it reads and prints the material property data and line boundary conditions. The stiffness and mass matrices are computed if ISW=3 and ISW=5 respectively. The load vector due to internal forces and boundary conditions is calculated when ISW=6.

The material properties may be input as constants or as a table of temperature and property values. If any other way of defining the property values is required by a specific problem, subroutines PKX, PKY, PROC or PQ which calculate respectively the conductivity in x and y directions, the thermal capacity and the heat generation per unit volume, can be easily modified or substituted.

The shape functions used by this heat transfer module are calculated by subroutines SHAPE and SHAP2. These routines are capable of constructing shape functions for a three-node triangle, a four-node linear quadrilateral, an eight-node quadratic serendipity quadrilateral, a nine-node quadratic Lagrangian quadrilateral or any combination in-between.

A four- to nine-node two dimensional element is shown in Figure 3. As shown by Bathe and Wilson in Ref. 5 the temperature within the element is expressed at any time



in the local coordinate system  $s, t$  in terms of the nodal temperatures  $T_i$  by

$$T(s,t) = \sum_i N_i(s,t) T_i$$

where

$$N_1 = (1-s)(1-t)/4 - (N_5+N_8)/2 - N_9/4$$

$$N_2 = (1+s)(1-t)/4 - (N_5+N_6)/2 - N_9/4$$

$$N_3 = (1+s)(1+t)/4 - (N_7+N_6)/2 - N_9/4$$

$$N_4 = (1-s)(1+t)/4 - (N_7+N_8)/2 - N_9/4$$

$$N_5 = (1-s^2)(1-t)/2$$

$$N_6 = (1+s)(1-t^2)/2$$

$$N_7 = (1-s^2)(1+t)/2$$

$$N_8 = (1-s)(1-t^2)/2$$

$$N_9 = (1-s^2)(1-t^2)$$

If any of the nodes from five to nine are omitted the corresponding value of  $N$  is zero.

The connectivity table must follow the numbering convention shown in Figure 3, otherwise wrong results will be obtained.

The maximum number of nodes per element will depend on the type of elements to be used and may be from three to nine. The eight-node serendipity shape functions occur if the ninth node number is omitted. The four-node quadrilateral shape functions are computed if only the first four nodal connections are non zero and the three-node triangle if the first three nodal connections are non zero. Finally, if a mid-side nodal connection is omitted the edge is linear.



The line boundary conditions, closely related to the type of element used, are defined in the cards following the macro MATE. The line subjected to a specified boundary condition is identified using the local coordinates. A line is numbered 1 or 2 if it is perpendicular to the axis  $s$  or  $t$ , respectively. Those numbers are positive or negative according to which direction of the axis it is perpendicular. Thus, as an example, the line defined by the nodes 2, 6 and 3 is numbered 1 while the line defined by nodes 1, 5 and 2 is numbered -2.

The boundary conditions are treated in subroutine BCOND2 and four types are considered:

- a) specified flux
- b) convection with constant heat transfer coefficient
- c) convection with temperature dependent heat transfer coefficient
- d) radiation

If more complex boundary conditions are required subroutine BCOND2 may be easily modified.

The numerical integration necessary to evaluate the matrices and vectors in equation (6) are performed using the Gauss quadrature formula; the abscissas and weight coefficients are calculated in subroutine PGAUSS. The user may choose the number of points per direction used in the integration from one to six, but the default value is four points.

To illustrate the above, we assume that our sector



used as an example is part of the cross section of a hollow cylinder made of a material with conductivity in x and y directions of 10, specific heat 250 and density 8000. If the outside surface is subjected to convection to an ambient atmosphere at 20 degrees with a constant heat transfer coefficient of 200, the cards following the macro card MATE are

MATE

```

      1      2
10.      10.      8000.      250.      3
      6      2      1 200.      20.
```

It must be noted that a plane analysis is specified and the integration will be performed using three points per direction. The reader must also note that this version of the problem is slightly different from the one given when the mesh data preparation was discussed. In the previous problem the line now subjected to a convection boundary condition was at a specified constant temperature of 20. If this second problem is to be solved the cards corresponding to nodes 31-33 in the BOUN and FORC sets must be omitted.

Following is a list of subroutines included in the two dimensional heat transfer module:

ELMT02 : main routine; forms the necessary arrays for the solution.

SHAPE : calculates the shape functions for the triangle and the linear quadrilateral elements.

SHAP2 : adds the quadratic terms and the center node



to the shape functions.

PGAUSS : determine the abscissae and weight coefficients for the numerical integration.

BCOND2 : adds to the arrays calculated in ELMT02 the contribution from the specified boundary conditions.

PKX : determine the temperature dependent conductivity in the x direction.

PKY : determine the temperature dependent conductivity in the y direction.

PROC : determine the temperature dependent thermal capacity.

PQ : determine the temperature dependent heat generation per unit volume.

CONV : determine the temperature dependent heat transfer coefficient.

TABLE : called by PKX, PKY, PQ, PROC and CONV; calculates the correspondent coefficient to a given temperature by linear interpolation between two consecutive entries in a table.

JACBB2 : calculates the jacobian determinant when an integration over a line is necessary.

## 2. Three Dimensional Heat Transfer Element

This module is called ELMT03 and is accessed using the number three in column ten of each material number card.

This element is the generalization of ELMT02 for three dimensional space and little remains to be said about its use.



The material properties are read in a similar way as the two dimensional case, but the conductivity in the z direction must be added. This module uses the same subroutines as ELMT02 to read and calculate the temperature dependent properties.

Subroutines SHAP3D and SHAP3 may construct interpolation functions for any combination between a eight-node linear brick and a 21-node Lagrangian brick.

An eight- to twenty one-node three dimensional solid element is shown in Figure 4. The natural coordinates (r, s and t) of the eight corner nodes are ( $\pm 1$ ,  $\pm 1$ ,  $\pm 1$ ), of the twelve mid-edge nodes are (0,  $\pm 1$ ,  $\pm 1$ ), ( $\pm 1$ , 0,  $\pm 1$ ) and ( $\pm 1$ ,  $\pm 1$ , 0) and of the center node are (0, 0, 0).

The temperature within the element is defined in terms of the nodal temperatures  $T_i$  at any time by [Ref. 5]

$$T(r,s,t) = \sum N_i(r,s,t)T_i$$

If we define

$$\begin{aligned} G(\beta, \beta_i) &= .5(1+\beta_i\beta) \quad , \quad \text{for } \beta_i = \pm 1 \\ &= 1-\beta^2 \quad \quad \quad \text{for } \beta_i = 0 \end{aligned}$$

and

$$g_i = G(r, r_i)G(s, s_i)G(t, t_i)$$

where  $r_i$ ,  $s_i$  and  $t_i$  are the natural coordinates of the element nodal points, the interpolation functions  $N_i$  are

$$\begin{aligned} N_1 &= g_1 - (N_9 + N_{12} + N_{17})/2 - N_{21}/8 \\ N_2 &= g_2 - (N_9 + N_{10} + N_{18})/2 - N_{21}/8 \\ N_3 &= g_3 - (N_{10} + N_{11} + N_{19})/2 - N_{21}/8 \\ N_4 &= g_4 - (N_{11} + N_{12} + N_{20})/2 - N_{21}/8 \end{aligned}$$



$$\begin{aligned}
N_5 &= g_5 - (N_{13}+N_{16}+N_{17})/2 - N_{21}/8 \\
N_6 &= g_6 - (N_{13}+N_{14}+N_{18})/2 - N_{21}/8 \\
N_7 &= g_7 - (N_{14}+N_{15}+N_{19})/2 - N_{21}/8 \\
N_8 &= g_8 - (N_{15}+N_{16}+N_{20})/2 - N_{21}/8 \\
N_j &= g_j - N_{21}/4 \quad \text{for } j=9,\dots,20 \\
N_{21} &= g_{21}
\end{aligned}$$

If any of the nodes from nine to twenty one are omitted the corresponding value of  $g$  is zero.

The numbering convention used in the definition of the shape functions and shown in Figure 4 must be followed by the user when the connectivity table is built and the surface boundary conditions are specified. The surfaces are numbered  $\pm 1$ ,  $\pm 2$ ,  $\pm 3$  as they are perpendicular to the positive or negative directions of the axis  $r$ ,  $s$ ,  $t$ , respectively.

As the numerical integration routine is the same as for ELMT02, the user may choose from one to six points per direction, the default value being four points.

Following is a list of subroutines included in the three dimensional heat transfer module:

ELMT03 : main routine; forms the matrices necessary for the solution.

SHAP3D : calculates the shape functions for the eight node linear brick.

SHAP3 : adds the quadratic terms to the shape functions and the center node for the Lagrangian brick.

BCOND3 : adds to the matrices the contribution from



the specified boundary conditions.

PKZ : determine the conductivity in the z direction

JACBB3 : calculates the jacobian determinant when an integration over a surface is necessary.

### C. TIME INTEGRATION MODULE

The first order ordinary differential equation solver is accessed when the macro command ODE1 is utilized and uses the Zienkiewicz two- and three-time level schemes. The details of these algorithms are treated in section IV.

This module was designed for the solution of first order equations but can, with little programming effort, be expanded in order to include higher order algorithms and solve higher order differential equations.

In order to economize computer memory space at some cost of execution time, this module only uses the space previously reserved for the mesh construction. As the mesh data is not needed during the execution of a time step integration, all the corresponding arrays are kept in file 9 during that period. Once the next displacement vector is calculated and the current displacement vector is saved in file 10, all the information in file 9 is retrieved and the mesh data restored.

File 10 is used as a working space during the time integration and the current displacement vector is saved in it for possible future use. This vector is retrieved if the algorithm used in the next time step integration is a three point scheme.



Besides the time step size change using the appropriate macro instructions (macro TOL), an optional automatic time step adjustment was incorporated in the program. The norm of the difference between the displacements vectors at two consecutive times is computed at each step. If the norm is less than a predetermined value  $\Delta T_{\max}$ , the time step size is doubled before going to the next step, whereas if the norm is greater than a prespecified value  $\Delta T_{\min}$ , the time step size is halved and the calculation for that time step is repeated until the norm is acceptable. The magnitudes of the maximum and minimum values of the norm are problem dependent and must be chosen by the user. If they are not specified no time step adjustment will be performed. The recalculation of the displacement vector is always done using the two point scheme, even when a three point scheme was utilized for the first calculation.

The macro command ODE1 used to access this module must be followed by a second macro command in order to determine the function to be performed. The secondary macro instructions are INIT, LINE or QUAD.

The couple ODE1 INIT reads the integration constants theta, beta and gamma, the parameters for the automatic time step adjustment and the initial displacement vector. This data must follow the macro program (see user's instructions). No time integration is performed by this instruction.

The couple ODE1 LINE performs the two point scheme



and the current displacement vector is substituted by the new calculated displacement vector.

The couple ODE1 QUAD has a function similar to ODE1 LINE but uses the three point scheme.

Once the execution of an ODE1 macro command is complete, the mesh data is restored and the displacement at time  $t$  replaced by the displacement at time  $t + \Delta t$ , when the secondary macro command is LINE or QUAD. If INIT is used, the displacement vector becomes the given initial displacement vector.

The subroutines included in this module are:

PODE1 : control routine; reserves working space and calls the appropriate subroutines.

INIT : reads the constants theta, beta and gamma and the maximum and minimum values for the norm used in the time step adjustment.

PLINE : performs the two point integration scheme.

PQUAD : performs the three point integration scheme.

TERM : performs the automatic time step adjustment.



#### IV. APPLICATION OF FEAP TO HEAT TRANSFER PROBLEMS

The program FEAP requires from the user the knowledge of the algorithm to be used in the solution of the problem to be treated.

In this section the algorithms used in the solution of heat conduction problems are discussed.

##### A. STEADY STATE PROBLEMS

###### 1. Algorithms for Linear Problems with Macro Program

In this case equation (8) becomes

$$[K] \{T\} + \{F\} = \{0\} \quad (12)$$

The algorithm used to solve (12) is described by

$$[K] \{v\} = -\{F\} - [K] \{T^0\} \quad (13)$$

$$\{T\} = \{T^0\} + \{v\} \quad (14)$$

where  $\{T^0\}$  are the specified nodal temperatures.

The macro instruction TANG builds the left hand side of (13) while FORM forms the vector in the right hand side. The instruction SOLV solves the system (13) and performs the step defined by (14).

Consequently the simplest macro program used to solve this problem is

```
TANG
FORM
SOLV
DISP
```



## 2. Algorithms for Nonlinear Problems with Macro Program

Equation (12) still applies to this case but now  $[K]$  and  $\{F\}$  may be dependent on the nodal temperatures  $\{T\}$ . The well known Newton-Raphson iteration may be used and the algorithm is described as

$$[K(T^i)] \{v^i\} = -\{F(T^i)\} - [K(T^i)] \{T^i\} = \{R^i\} \quad (15)$$

$$\{T^{i+1}\} = \{T^i\} + \{v^i\} \quad (16)$$

where  $\{T^i\}$  are the nodal temperatures at the  $i$ th iteration. The algorithm defined by (15) and (16) is the one used for the linear problem repeated several times. Then the macro program may be

```

LOOP      n
TANG
FORM
SOLV
DISP
NEXT
DISP

```

where  $n$  are the user's guess of the number of iterations necessary to obtain equilibrium. However, the program has an internal check on the value of the vector norm  $||R^i||$ , where

$$||R|| = \left( \sum_{k=1}^n |R_k^i|^2 \right)^{1/2}$$

Whenever

$$||R^i|| < \text{TOL} \times \max_j ||R^j|| \quad (j=1, \dots, i)$$

where  $\text{TOL}$  is a predefined tolerance ( $10^{-9}$  is the default



value), the iteration ceases and a skip to the macro command immediately following the first NEXT occurs. Usually one begins with zero as the initial guess of the displacement vector; however, any other vector may be used.

## B. TIME DEPENDENT PROBLEMS

### 1. Two and Three Point Recurrence Schemes for First Order Equation

The set of differential equations

$$[K]\{T\} + [C]\{\dot{T}\} + \{F\} = \{0\} \quad (8)$$

may be solved using one of the many recursive schemes described in the literature. From them, the two and three time level schemes presented by Zienkiewicz [Ref.4] offer a wide range of choices for the solution of linear and nonlinear problems.

The recurrence relation for the two-time level scheme can be written as

$$\left(\frac{1}{\Delta t}[C] + \theta[K]\right)\{T_{n+1}\} + \left(-\frac{1}{\Delta t}[C] + (1-\theta)[K]\right)\{T_n\} + \{\bar{F}\} = \{0\} \quad , \quad 0 \leq \theta \leq 1 \quad (17)$$

where the subscripts  $n$  and  $n+1$  denote evaluation at time  $t$  and  $t+\Delta t$ . The vector  $\{\bar{F}\}$  is defined as

$$\{\bar{F}\} = \{F_{n+1}\} + (1-\theta)\{F_n\} \quad (18)$$

The choice of the parameter  $\theta$  defines the particular scheme to be used and the reader will recognize a well known series of finite difference formulas with a modification of using a weighted loading term  $\{\bar{F}\}$ .

Consider the system of decoupled equations in terms of the modal participation variables  $T_i$



$$c_i \dot{T}_i + k_i T_i + f_i = 0 \quad (19)$$

for free response, i.e.,  $f_i=0$ , expression (17) becomes

$$\left(\frac{1}{\Delta t} c_i + k_i \theta\right) (T_i)_{n+1} + \left(-\frac{1}{\Delta t} c_i + k_i (1-\theta)\right) (T_i)_n = 0 \quad (20)$$

Substituting the relation

$$(T_i)_{n+1} = \lambda (T_i)_n \quad (21)$$

in (20), the resulting characteristic equation of the recurrent scheme solved for  $\lambda$  gives

$$\lambda = [1 - k_i (1-\theta) \Delta t / c_i] / [1 + k_i \theta \Delta t / c_i]$$

The scheme will be unconditionally stable if

$$|\lambda| < 1$$

for any value of  $p_i$  where

$$p_i = (k_i / c_i) \Delta t \quad (22)$$

This condition is satisfied for

$$\theta \geq 1/2$$

On the other hand if

$$0 < \theta < 1/2$$

stability is conditional requiring

$$p_i < 2/(1-2\theta)$$

Figure 5 shows how  $\lambda$  varies with  $p_i$  for the schemes discussed later and how it compares with the exact value of

$$\lambda = \exp(-p_i)$$

The schemes considered are

- a) Crank-Nicolson with  $\theta=1/2$
- b)  $\theta=2/3$  proposed by Zienkiewicz [Ref. 4]
- c)  $\theta=3/4$  proposed by the author
- d)  $\theta=0.878$  proposed by Liniger in Ref. 6.

The Zienkiewicz three-time level scheme is defined



as

$$(\gamma[C] + \beta \Delta t[K])\{T_{n+1}\} + ((1-2\gamma)[C] + (1/2-2\beta+\gamma)\Delta t[K])\{T_n\} + (- (1-\gamma)[C] + (1/2+\beta-\gamma)\Delta t[K])\{T_{n-1}\} + \Delta t\{F\} = \{0\} \quad (23)$$

where the subscripts  $n$ ,  $n+1$  and  $n-1$  denote evaluation at time  $t$ ,  $t+\Delta t$  and  $t-\Delta t$  respectively.

The vector  $\{\bar{F}\}$  is defined as

$$\{\bar{F}\} = \beta\{F_{n+1}\} + (1/2-2\beta+\gamma)\{F_n\} + (1/2+\beta-\gamma)\{F_{n-1}\} \quad (24)$$

and the parameters  $\gamma$  and  $\beta$  define the particular scheme to be used.

Considering again the system of decoupled equations (19) and using the relations (21) and (22) one finds that the characteristic equation for the three-time level scheme is

$$(\gamma+\beta p_i) \lambda^2 + [(1-2\gamma) + (1/2-2\beta+\gamma)p_i]\lambda + [-(1-\gamma) + (1/2+\beta-\gamma)p_i] = 0 \quad (25)$$

Writing

$$g = [1+(1/2+\gamma)p_i]/[\gamma+\beta p_i]$$

$$h = [-1+(1/2-\gamma)p_i]/[\gamma+\beta p_i]$$

the roots of (25) may be written as

$$\lambda_{1,2} = (2-g)/2 \pm [(2-g)^2 - 4(1-h)]^{1/2}/2$$

These roots will be complex if the quantity under the square root is negative. Then the modulus of  $\lambda$  is

$$|\lambda| = (1-h)^{1/2}$$

The scheme is stable if  $|\lambda| < 1$  for any  $p_i$  and Wood in Ref. 7 shows that this condition is satisfied if

$$\gamma > 1/2 \quad \text{and} \quad \beta > \gamma/2$$



Equation (25) has two roots:  $\lambda_2$ , the principal root which is the approximation to the exact

$$\lambda = \exp(-p_i)$$

and the spurious root  $\lambda_2$ . For relative stability one must have

$$|\lambda_1| < |\lambda_2|$$

Also a negative root or complex roots can produce oscillation.

In Figure 6a-f the real roots  $\lambda_1$  and  $\lambda_2$  and the modulus  $|\lambda|$  for the complex roots are plotted against  $p_i$  for the schemes:

- a)  $\beta=1/3$ ,  $\gamma=1/2$  proposed by Lees in Ref. 8
- b)  $\beta=3/4$ ,  $\gamma=1$  proposed by Hogge in Ref.9
- c)  $\beta=0.646$ ,  $\gamma=1.2184$  proposed by Wood [Ref. 7]
- d)  $\beta=4/5$ ,  $\gamma=3/2$  proposed by Zienkiewicz [Ref. 4]
- e)  $\beta=9/10$ ,  $\gamma=3/2$  proposed by the author
- f) Fully implicit algorithm,  $\beta=1$ ,  $\gamma=3/2$  [Ref. 7].

From Figure 6a one may predict a very strong oscillatory behaviour for the Lees algorithm a).

In order to study the various schemes, the physical problem represented by the nondimensionalized linear heat conduction equation

$$\frac{\delta T}{\delta t} = \frac{\delta^2 T}{\delta x^2}$$

was solved in a bar of length 4 and width 1 subjected to the boundary conditions

$$\begin{array}{lll} T=1 & \text{at} & x=0 \\ \frac{\delta T}{\delta x}=0 & \text{at} & x=4 \end{array}$$



and initial condition

$$T=0 \quad \text{at} \quad t=0$$

i.e., consider a step change in the surface temperature. This problem was also studied by Wood and Lewis in Ref. 10.

The problem was discretized in space using ten equal length two dimensional quadratic elements through the length of the bar. This type of elements was chosen because it was reported by Wood and Lewis as giving the worst numerical results.

In order to avoid the discontinuity in the loading term caused by the step change of the surface temperature at the initial time, the problem was first solved starting from time  $t=\Delta t$ , where  $\Delta t$  is the time step size. The value of the temperature distribution at  $t=\Delta t$  is provided by the exact analytical solution. This procedure also provides the necessary two starting vectors for the three-time level schemes.

The temperature of the nodes at  $x=1$  is used as reference value to compare the various schemes.

In Figure 7a-d the results obtained from the various two-time level schemes with  $\Delta t=2$  for the temperature at  $x=1$  are compared with the analytical solution. For this ideal starting conditions the Crank-Nicolson,  $\theta=1/2$ , proves to be the most accurate algorithm. When the step size was increased to 10, all the schemes performed well as shown in Figure 8a-d.

The corresponding results for the three-time level



schemes are presented in Figure 9a-f and Figure 10a-f for  $\Delta t=2$  and  $\Delta t=10$ , respectively.

Strong oscillations were observed with the Lees algorithm a).

For  $\Delta t=2$  the schemes c), d) and e) performed well. For the larger step size of 10 the safest schemes are e) and f).

The step change in the loading term has been reported [Ref. 10] as producing relative instability when some of the schemes are used, particularly with the Crank-Nicolson algorithm. The noise may be suppressed reducing the magnitude of the time step but this is impracticable in most problems. Other techniques may be used to reduce the amplitude of the oscillations as those discussed by Wood and Lewis [Ref. 10] and Gresho and Lee in Ref. 11.

The step change in the loading term in the problem in study is due to the variation of the surface temperature at the initial time  $t=0$ . That discontinuity in the force term is illustrated in Figure 11a. When the time dimension is discretized it has been common practice to calculate the numerical solution at the nodes  $t=0$ ,  $t=\Delta t$ , etc. When a three-time level scheme is used the node at  $t=-\Delta t$  is considered assuming all conditions steady before  $t=0$ . The force values used by this method are indicated by the circles in Figure 11a. This procedure transfers to the numerical solution the uncertainty in the value of the force



term at time  $t=0$  introduced by the mathematical discontinuity assumed in the analytical solution. This problem is avoided if one uses the procedure illustrated in Figure 11b. The nodes considered for the discretization are those at  $t=-\Delta t/2$ ,  $t=\Delta t/2$ ,  $t=3\Delta t/2$ , etc. The discontinuity at  $t=0$  is smoothed by the interpolation of the loading term inherent to the particular scheme being used. The values of the force term for every node are well defined and no uncertainty is associated with any of them. For the three-time level schemes the node at  $t=-3\Delta t/2$  is used assuming all conditions steady before  $t=0$ .

The problem studied was solved using both starting procedures. The results obtained with the first procedure, start at  $t=0$ , using the two- and three-time level schemes are shown in Figures 12a-d and 13a-f, respectively. The second procedure, start at  $t=-\Delta t/2$ , gives the results shown in Figures 14a-d and 15a-f for the two- and three-time level schemes, respectively.

One may conclude that, in this simple problem, when the procedure illustrated in Figure 11b is followed, the step change in the loading term do not deteriorate the accuracy of the numerical results obtained with the schemes tested, with the exception of the Crank-Nicolson algorithm. This scheme shows an oscillatory behaviour not observed when the step change in the force term is not present.

The starting procedure of Figure 11b also proves to be an efficient way of starting the time integration with



the three-time level schemes.

## 2. Algorithms for Time Dependent Problems with Macro Program

A simplified form of the two-time level scheme is available in FEAP and is described by

$$\left(\frac{1}{\Delta t}[C_n] + \theta[K_n]\right)\{T_{n+1}\} + \left(-\frac{1}{\Delta t}[C_n] + (1+\theta)[K_n]\right)\{T_n\} + \Delta t\{F\} = \{0\} \quad (26)$$

where the subscripts  $n$  and  $n+1$  denote evaluation at time  $t$  and  $t+\Delta t$ , respectively.

The three-time level scheme programmed in FEAP is described by the expression

$$\begin{aligned} &(\gamma[C_n] + \beta \Delta t[K_n])\{T_{n+1}\} + ((1-2\gamma)[C_n] + (1/2-2\beta+\gamma)\Delta t[K_n])\{T_n\} \\ &+ (-(1-\gamma)[C_n] + (1/2+\beta-\gamma)\Delta t[K_n])\{T_{n-1}\} \\ &+ \{F_n\} = \{0\} \end{aligned} \quad (27)$$

where the subscripts  $n$ ,  $n+1$  and  $n-1$  denote evaluation at times  $t$ ,  $t+\Delta t$  and  $t-\Delta t$  respectively.

It must be noted that in (26) and (27) the vector  $\{F_n\}$  is an approximation of the interpolated force vector  $\{\bar{F}\}$ .

If the force vector is strongly dependent on time, this approximation may produce wrong results.

In problems where the stiffness matrix is constant and the force term results from the specified boundary displacements or where the specified boundary forces are only time dependent, the interpolation of the force term may be easily accomplished since the force/displacement boundary values can be changed at any time using the macro commands MESH or PROP. This procedure was found particularly useful



when a step change in boundary displacements or forces is applied at the initial time.

The first order ordinary differential equation solver module is accessed using the macro command ODE1 followed by one of the following macro commands:

INIT to specify the initial displacement vector  
and the integration constants

LINE to perform the two-time level algorithm

QUAD to perform the three-time level algorithm

Since the matrices  $[K]$  and  $[C]$  and the vector  $\{F\}$  must be formed before the time integration, the macro commands TANG, CMAS or LMAS and FORM are closely associated with the use of ODE1. It should be noted that the macro FORM forms the vector

$$\{R(T_n)\} = -\{F_n\} - [K_n] \{T_n\}$$

which is always dependent on the current displacement. Therefore the macro command FORM must be used every time step and precede ODE1.

To solve a fully nonlinear problem the following macro program may be used:

```
DT          Δt
ODE1 INIT
LOOP        n
TANG
CMAS ( or LMAS )
FORM
ODE1 LINE ( or QUAD )
```



TIME  
DISP  
NEXT

The macro command program must be followed by the data cards necessary to specify the integration constants  $\theta$ ,  $\gamma$  and  $\beta$  and the initial displacement vector.

If the matrices  $[K]$  and/or  $[C]$  are not temperature or time dependent then the instructions TANG and/or CMAS or LMAS must be placed outside the loop.

For the simplest case of a linear problem, the macro program may be

```
DT           $\Delta t$ 
ODE1 INIT
TANG
CMAS ( or LMAS )
LOOP        n
FORM
ODE1 LINE ( or QUAD )
TIME
DISP
NEXT
```



## V. NUMERICAL EXAMPLES

### A. RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

Consider the hollow cylinder of infinite length already discussed in section III. Consider the case in which the inside wall is maintained at a constant temperature and heat is lost by convection through the outer wall. This problem was considered for solution by Lew [Ref. 3].

The geometrical and thermophysical characteristics are:

Inside radius ( $r_{in}$ )	0.1 m
Outside radius ( $r_{out}$ )	0.3 m
Thermal conductivity ( $k$ )	10 W/m °C
Outside ambient temperature	20 °C
Heat transfer coefficient	200 W/m <sup>2</sup> °C
Inside wall temperature ( $T_{in}$ )	820 °C

The finite element model of this problem was completely discussed before and is shown in Figure 1. The complete computer output is given in Appendix B. It must be noted that the element arc width used is arbitrary since the problem is symmetric and no heat is conducted perpendicular to the radius.

The comparison between the analytical and FEAP solutions is shown in Figure 15. In this case the values provided by the finite element model used are very accurate and no further mesh refinement is necessary.



## B. NONLINEAR STEADY STATE HEAT CONDUCTION IN A SLAB WITH TEMPERATURE DEPENDENT CONDUCTIVITY

A liquid is boiled by a flat electric heater plate of thickness  $2L$ . The internal energy  $Q$  generated electrically may be assumed to be uniform. The boiling temperature of the liquid, corresponding to a specific pressure, is  $T_w$ .

In the finite element analysis for the temperature distribution in the heater, ten equal length quadratic elements were used to represent the unit cross section through the half plate thickness  $L$ , since the problem is symmetric. The extremity of the resulting slab is assumed at the temperature  $T_w$  while the other is insulated.

The thermal conductivity is assumed temperature dependent according to the expression

$$k = a(1+bT)$$

where  $a$  and  $b$  are constants.

Assuming the following values for the parameters in a consistent system of units

$$T_w = 0.0$$

$$a = 0.5$$

$$L = 1.0$$

$$Q = 1.0$$

the problem was solved for  $b$  equal 0.0, 1.0 and 2.0.

The numerical results are compared with the analytical solutions in Figure 16 and one may conclude that FEAP provides exact solutions in this problem.



### C. TRANSIENT TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

The same hollow cylinder of infinite length treated before is considered again but the case solved is the one with both outside and inside walls maintained at constant temperatures. Thermal diffusivity,  $\alpha = k / \rho c$  is specified as  $0.00005 \text{ m}^2/\text{sec}$ . Initially the cylinder is at an ambient temperature  $T_0$  of  $20^\circ\text{C}$ ; suddenly the temperature of the inside wall is raised to  $820^\circ\text{C}$  while the outside wall is maintained at  $20^\circ\text{C}$ . All other conditions are the same as for the steady state problem.

In order to obtain the unsteady solution the number of elements in the finite element model was doubled, thus twelve radial quadratic elements were used. The element arc width chosen is the same  $6^\circ$  as before.

The two time-level scheme with  $\theta = 2/3$  was used for the time integration. In order to compare the results the dimensionless temperature  $T^*$ , radius  $r^*$  and Fourier number  $Fo$  were used. They are defined as

$$T^* = (T - T_0) / (T_{in} - T_0)$$

$$r^* = r / r_{in}$$

and

$$Fo = \alpha t / r_{in}^2$$

A time step size  $\Delta t$  of 2 seconds was chosen initially. After 10 sec,  $\Delta t$  was increased to 10 sec and to 50 sec after 200sec. After 1000 sec,  $\Delta t$  was increased to 100 sec.

Figure 17 shows the evolution of the temperature of the



center interior points of the cylinder and Figure 18 compares the analytical values with the FEAP solution for the radial temperature distribution for two different times.

#### D. SLAB WITH RADIATION-CONVECTION BOUNDARY CONDITIONS

Consider a plane slab of thickness  $L$ , with constant thermophysical properties ( $k = \rho c = 1$ ), subjected to simultaneous radiation and convection at  $x=0$  and insulated at  $x=L$  ( $0 < x < L$ ). The slab is initially at a uniform temperature  $T_0$  and it was decided to consider zero ambient temperatures for convection and radiation.

It was also decided to consider the Biot number

$$Bi = h_c L / k = 1$$

where  $h_c$  is the convection heat transfer coefficient and

$$R = \epsilon \sigma T_0^3 L / k = 4$$

where  $\epsilon$  is the emissivity and  $\sigma$  the Stefan-Boltzman constant.

The same mesh as in Example B was employed for the finite element solution. The time integration was performed by the three-time level scheme with  $\gamma = 1/2$  and  $\beta = 1/3$ . A variable time step size was chosen. A value of  $\Delta t = 0.001$  was used initially. After  $t = 0.02$ ,  $\Delta t$  was increased to 0.0025. After  $t = 0.2$ ,  $\Delta t$  was increased to 0.01. After  $t = 0.4$ ,  $\Delta t$  was increased to 0.025. After  $t = 1.0$ ,  $\Delta t$  was increased to 0.05.

The history of the ratios  $T_w/T_0$  and  $T_m/T_0$  are plotted in Figure 19 against the Fourier number

$$Fo = \alpha t / L^2$$



$T_w$  is the surface temperature at  $x=0$  and  $T_m$  is the surface temperature at  $x=L$ .

The finite element results are compared with those given by Haji-Sheick and Sparrow in Ref. 12 and obtained from a Monte Carlo method.



## VI. CONCLUSIONS AND RECOMMENDATIONS

The computer program in this thesis provides an accurate and reliable means for solving a variety of Heat Transfer problems. The use of this program and efforts to increase its versatility are highly encouraged.

The capabilities of the program, while quite significant, can still be improved. The present version is designed for an "in-core" solution technique, which restricts the problem size to within the computer core-size. The capacity to handle large problems may be increased through the use of some "out of core" technique for solving the system of equations and even for building the mesh data. In that case the size of the problems treated would be restricted only by the availability of external storage devices.

In order to check the mesh input and to easily interpret the output, a graphics module must be included in the program.

Although the time integration module seems to provide reliable results in linear and some nonlinear problems, it should be subjected to further research in order to test it with different types of nonlinear analysis.



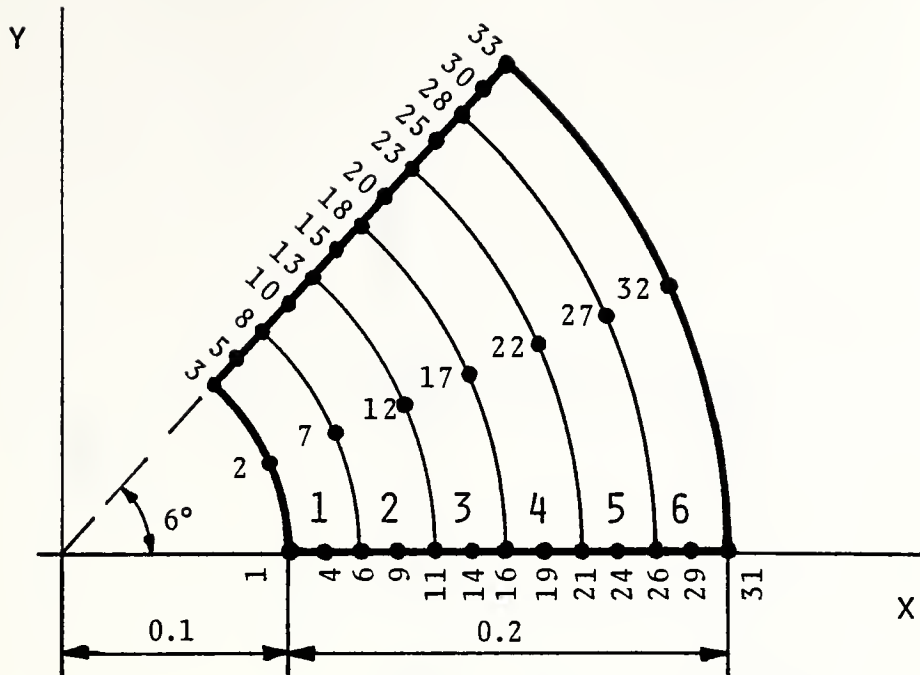


Figure 1. Mesh for Hollow Cylinder Problem

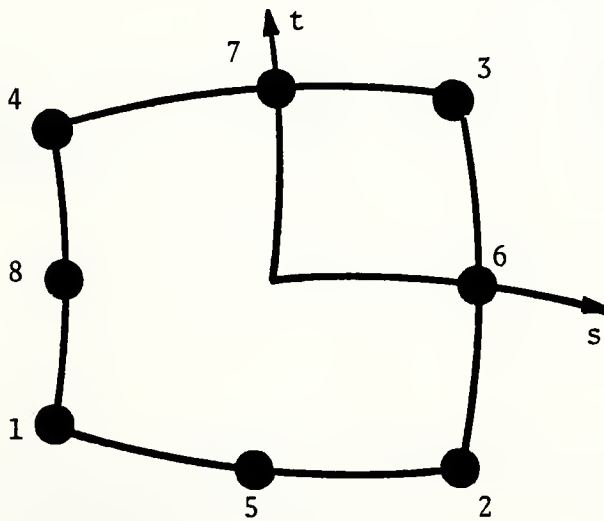


Figure 2. Eight-Node Serendipity Quadrilateral



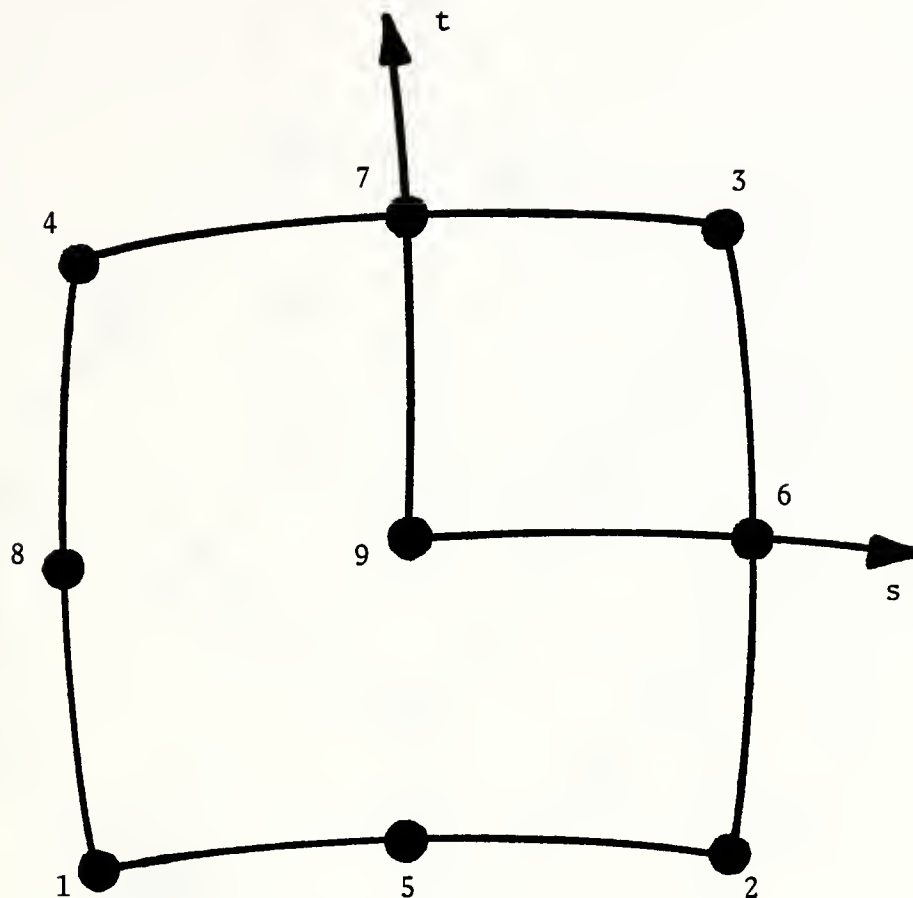


Figure 3. Local Node Number Sequence for a Quadratic Lagrangian Element



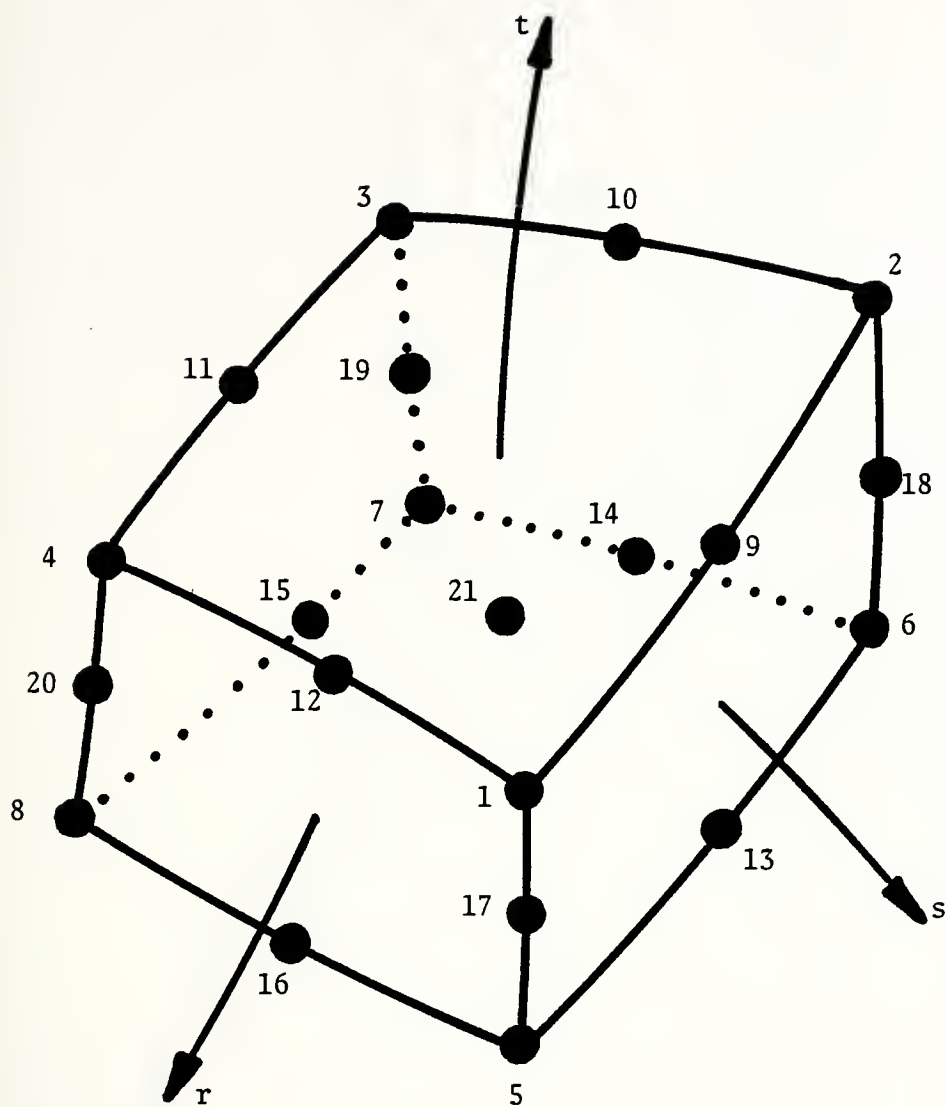


Figure 4. Local Node Number Sequence for a 21-Node Three Dimensional Element



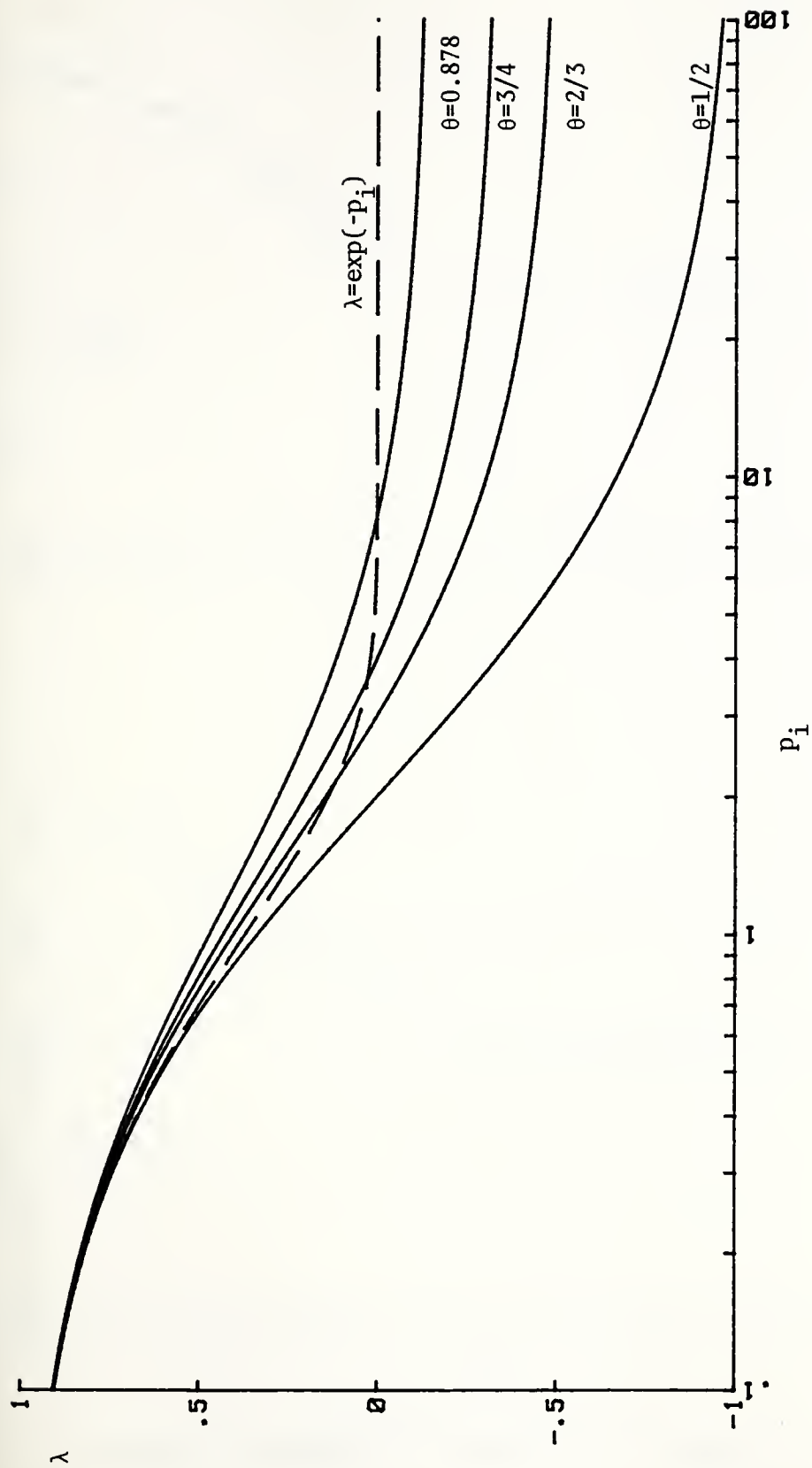
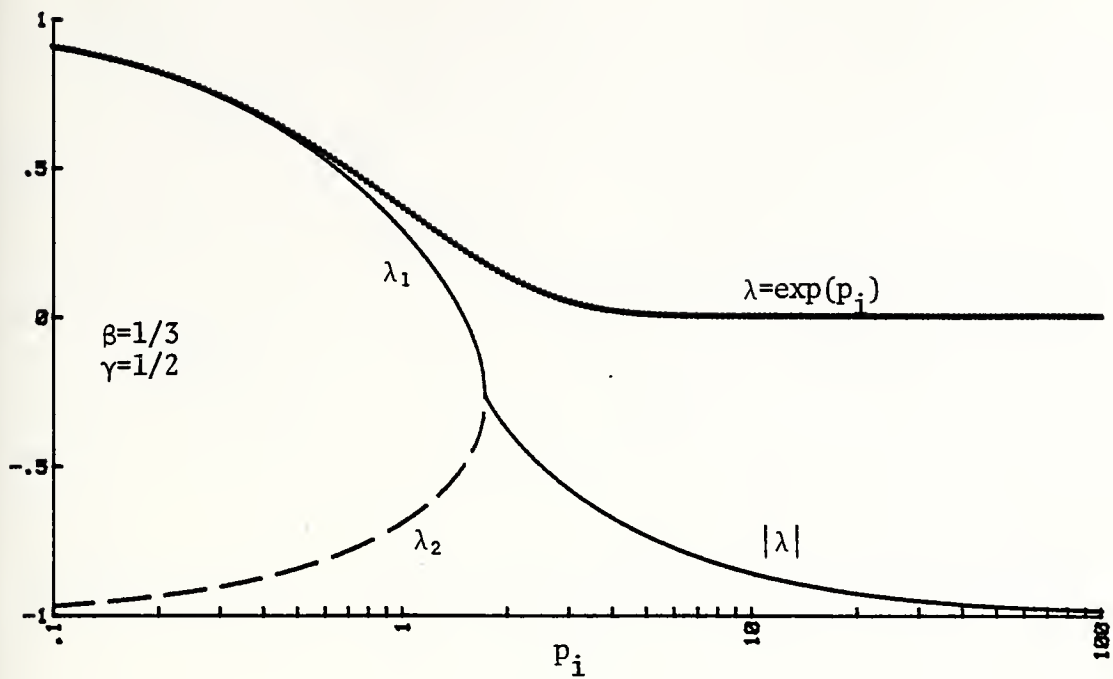
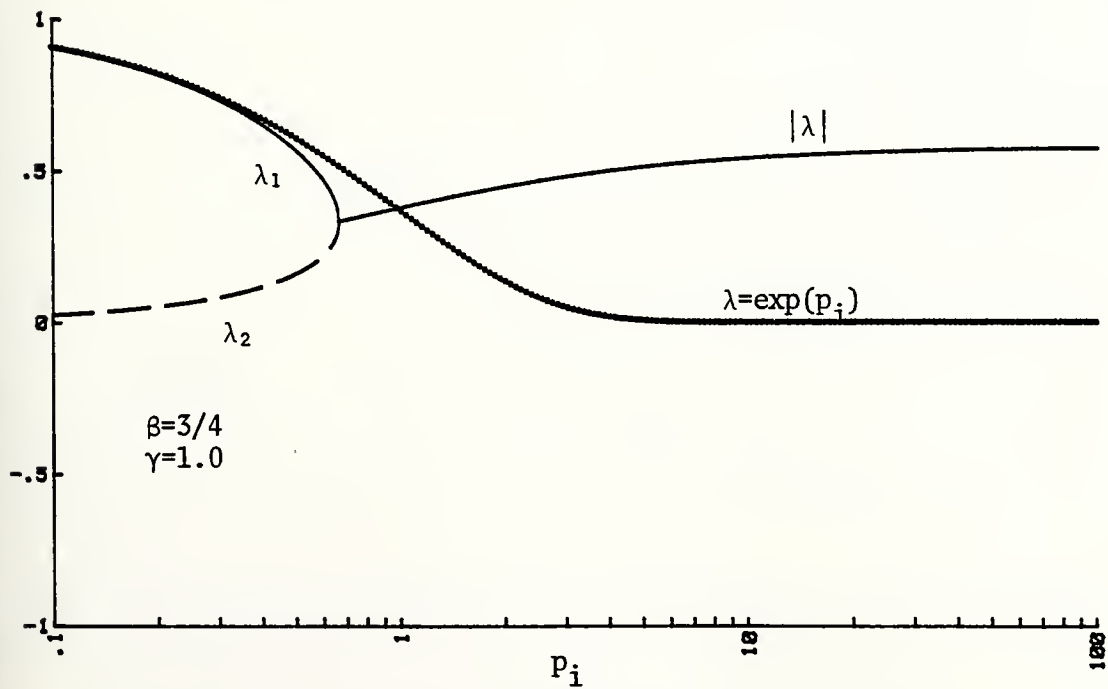


Figure 5. Zienkiewicz Two-Time Level Scheme





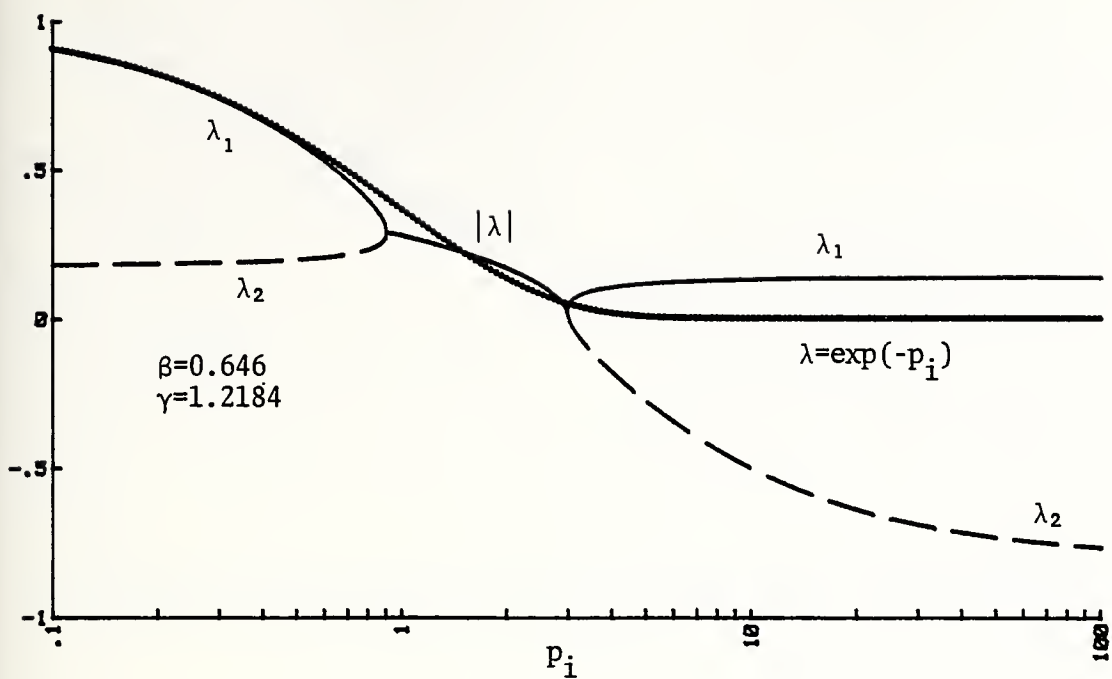
(a)



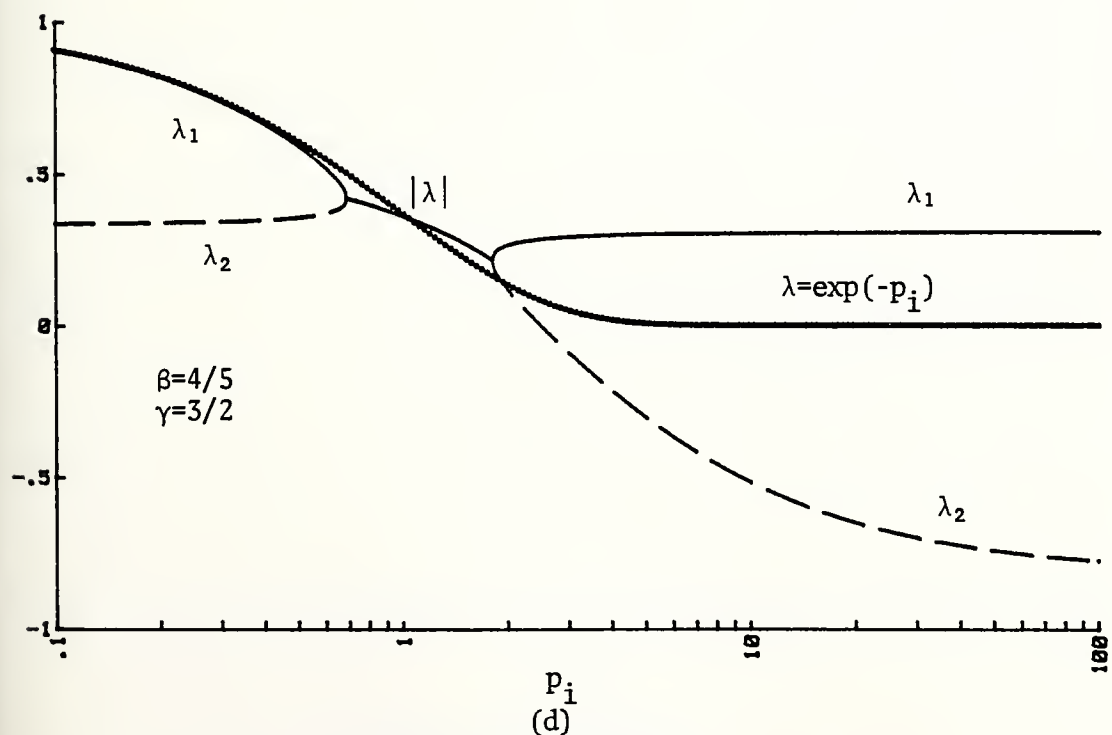
(b)

Figure 6. Zienkiewicz Three-Time Level Schemes





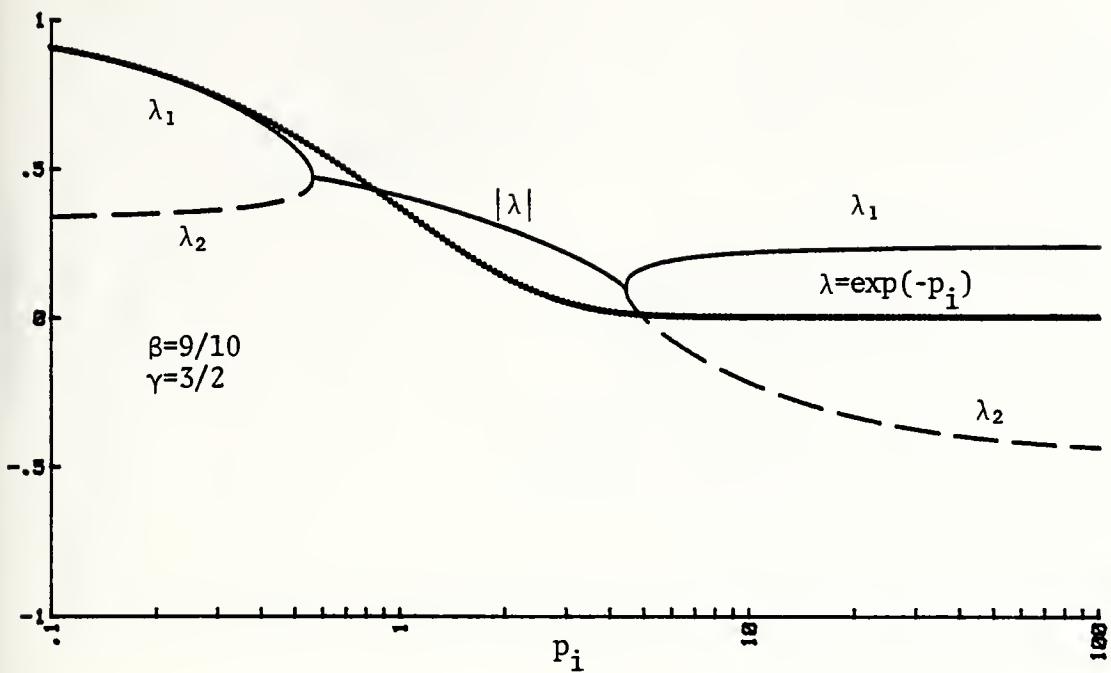
(c)



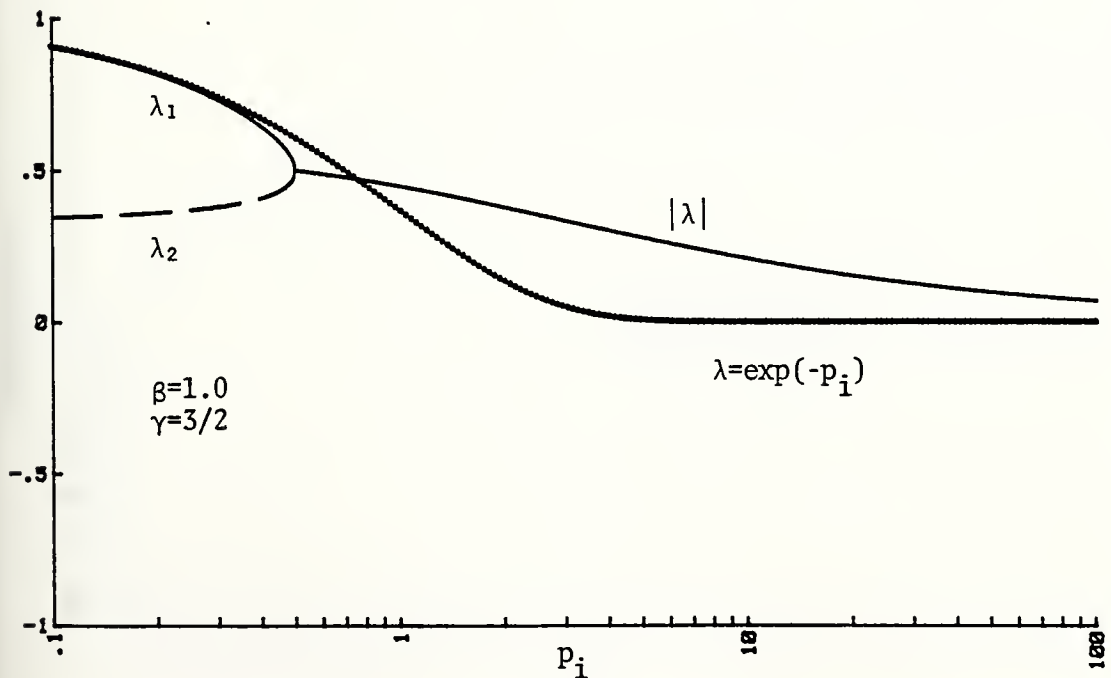
(d)

Figure 6. (continued)





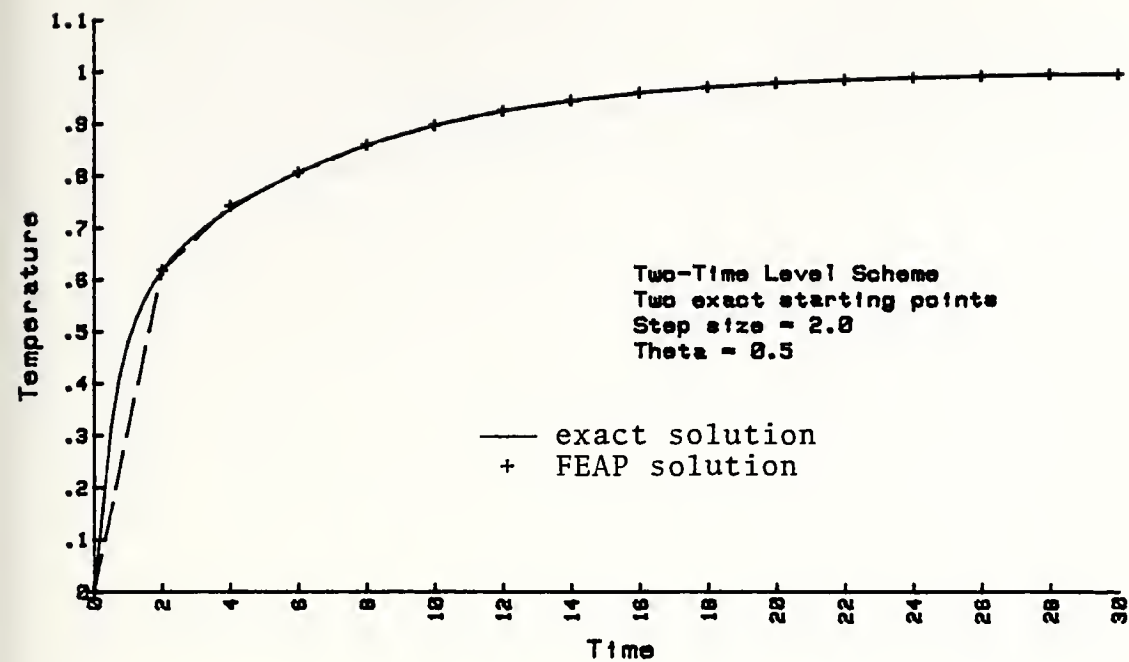
(e)



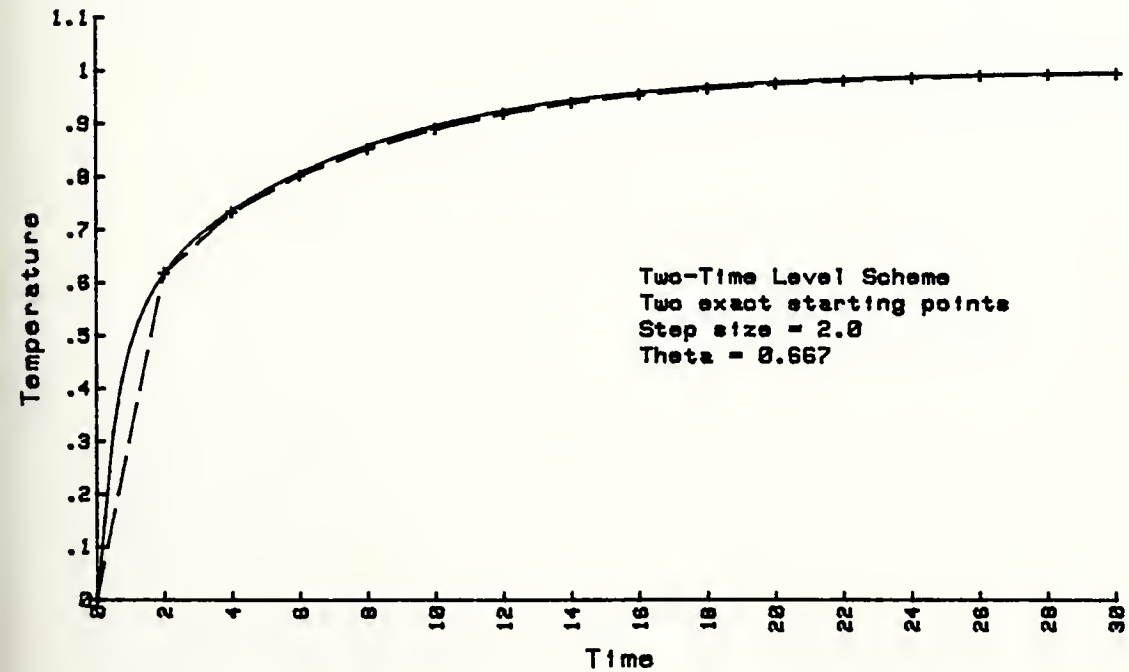
(f)

Figure 6. (continued)





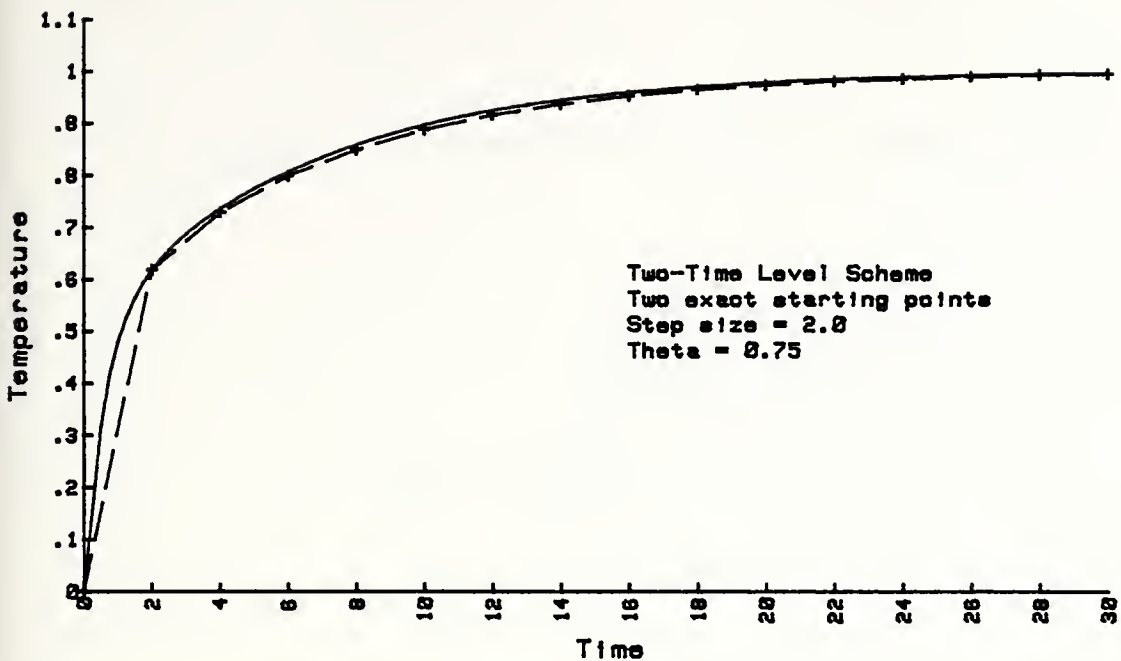
(a)



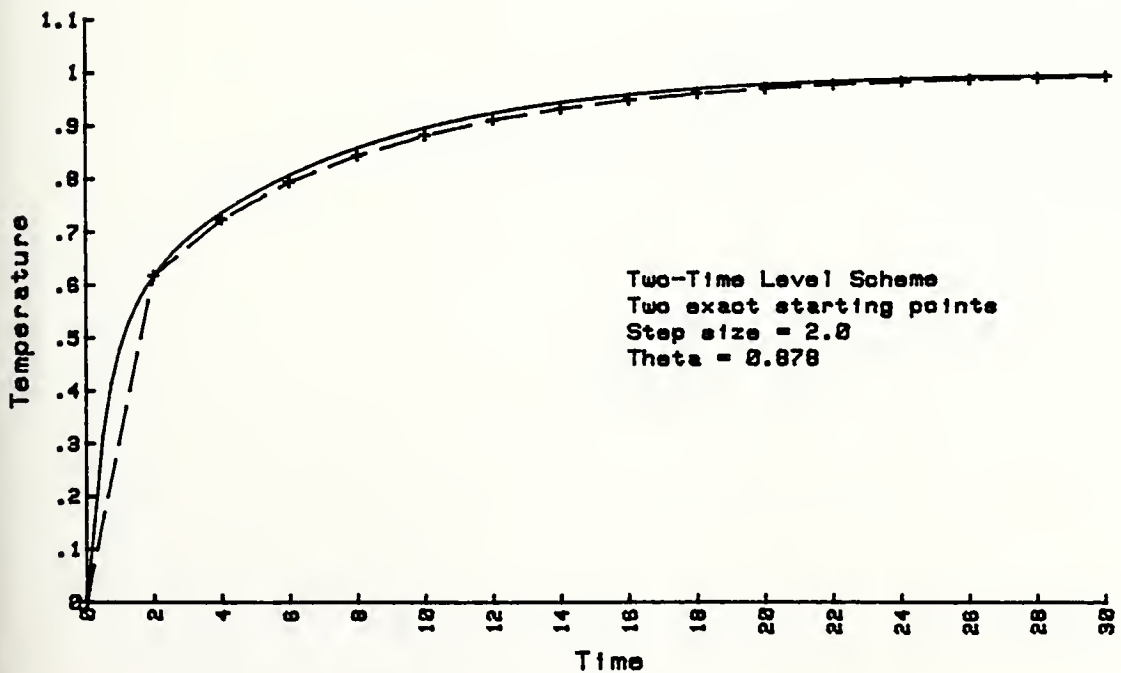
(b)

Figure 7.





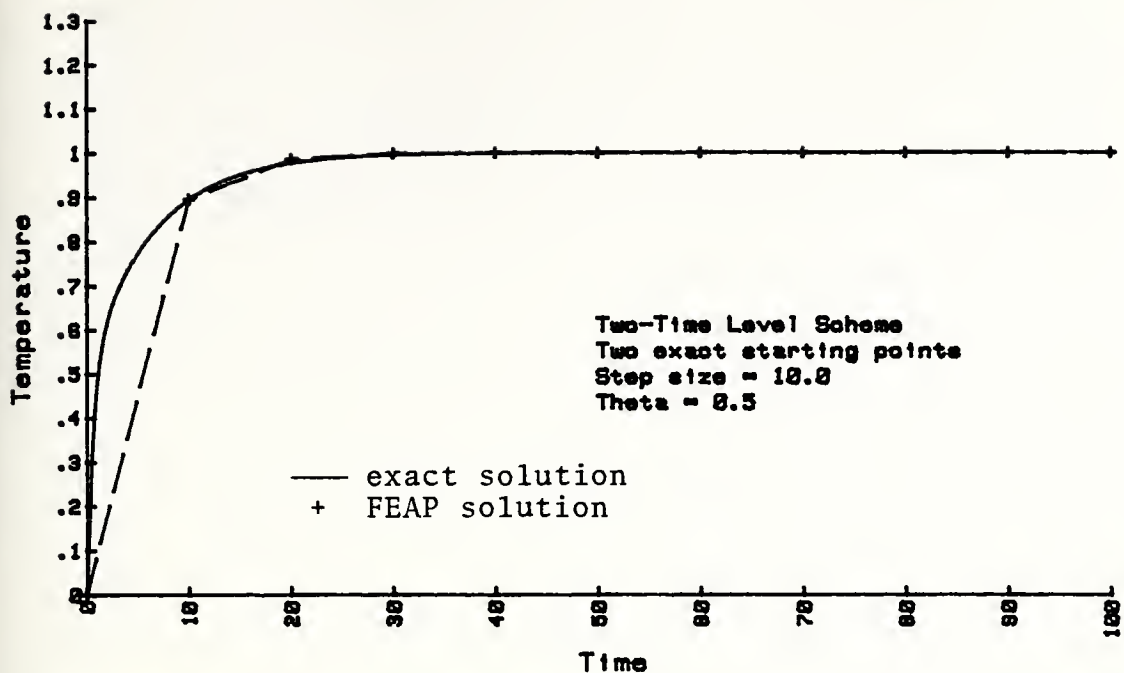
(c)



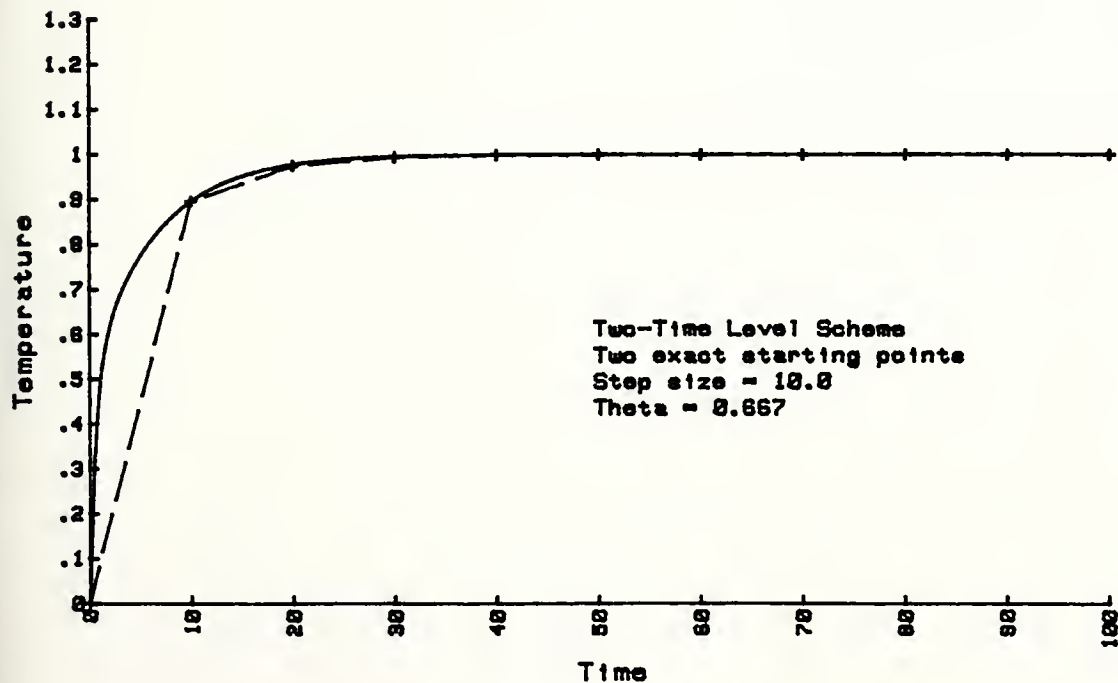
(d)

Figure 7. (continued)





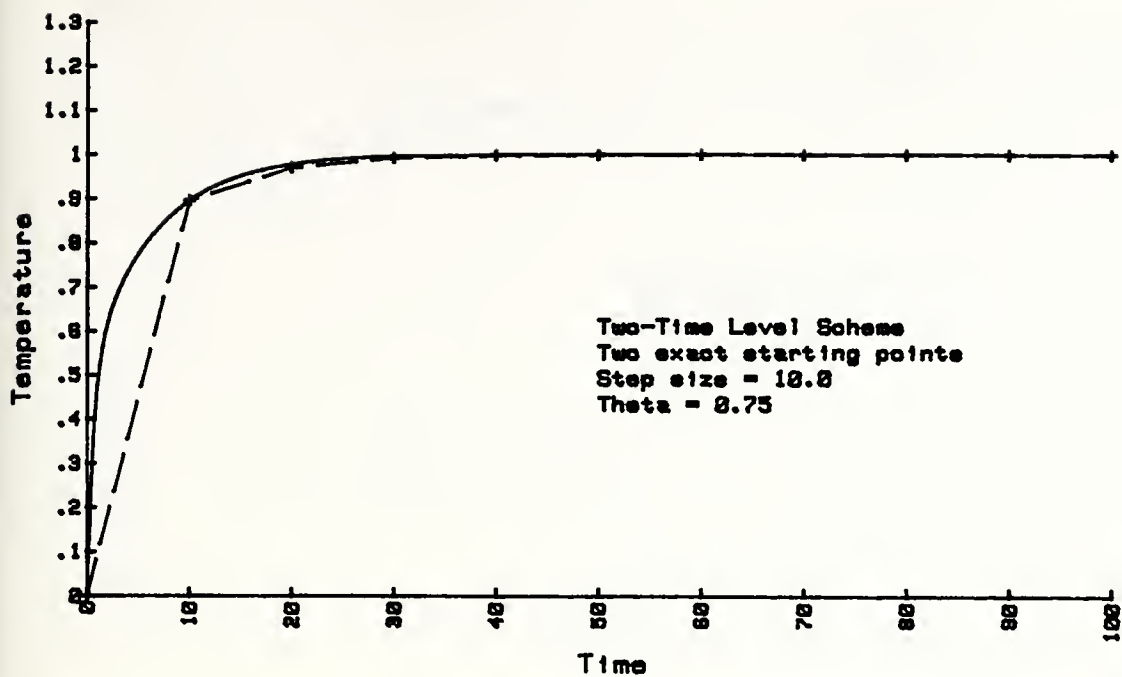
(a)



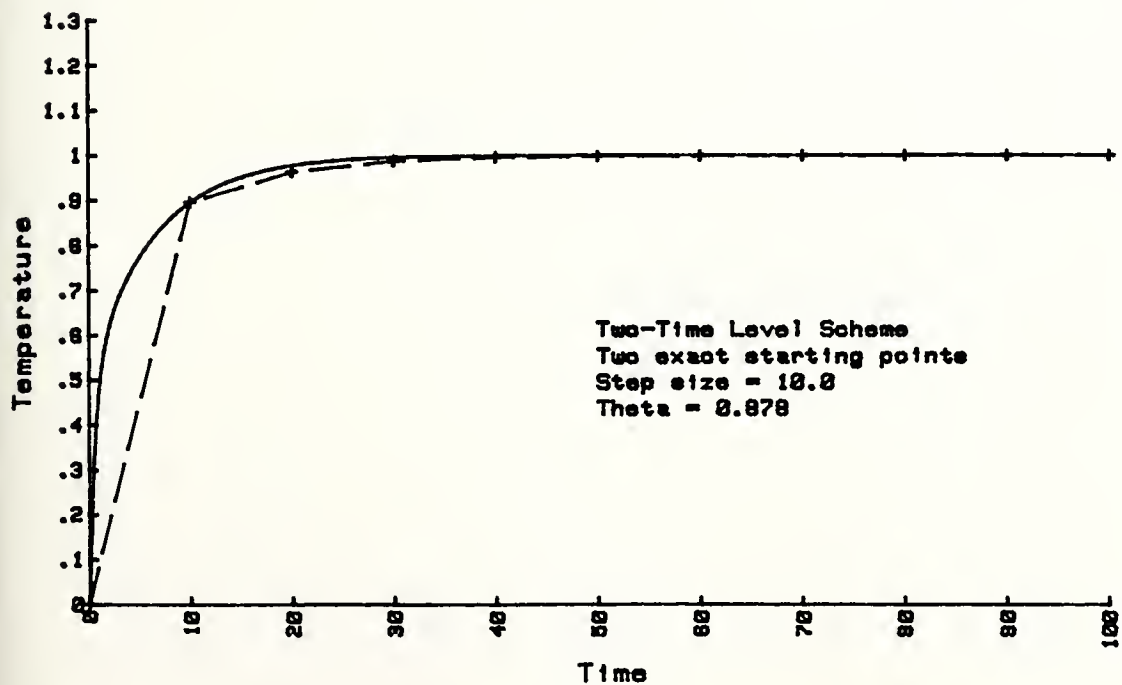
(b)

Figure 8.





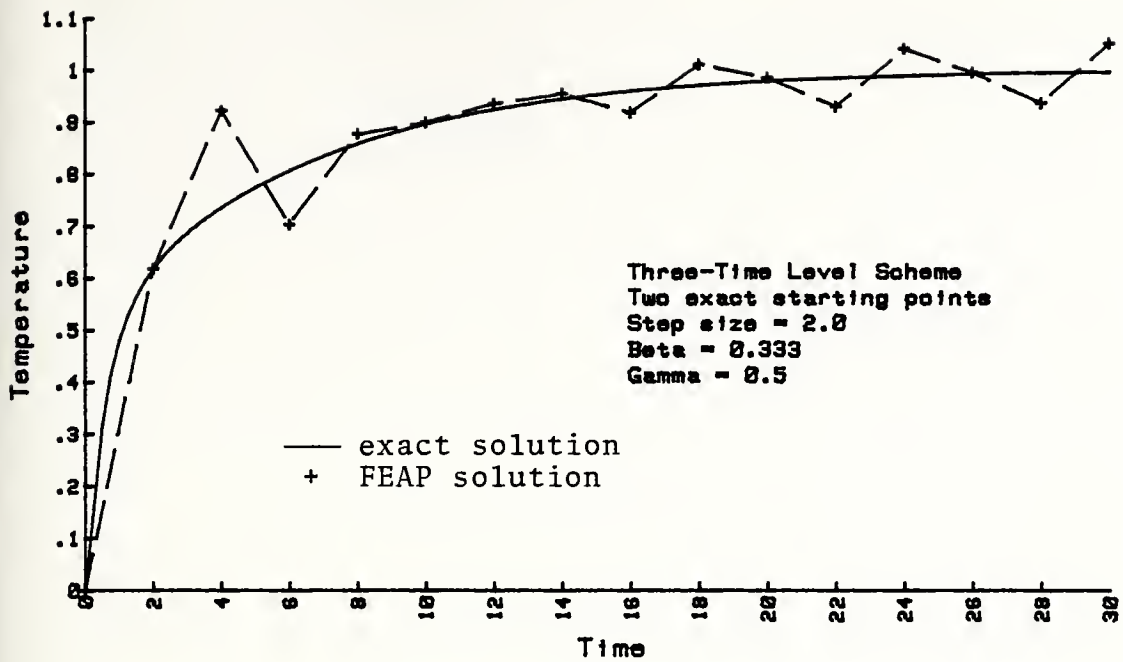
(c)



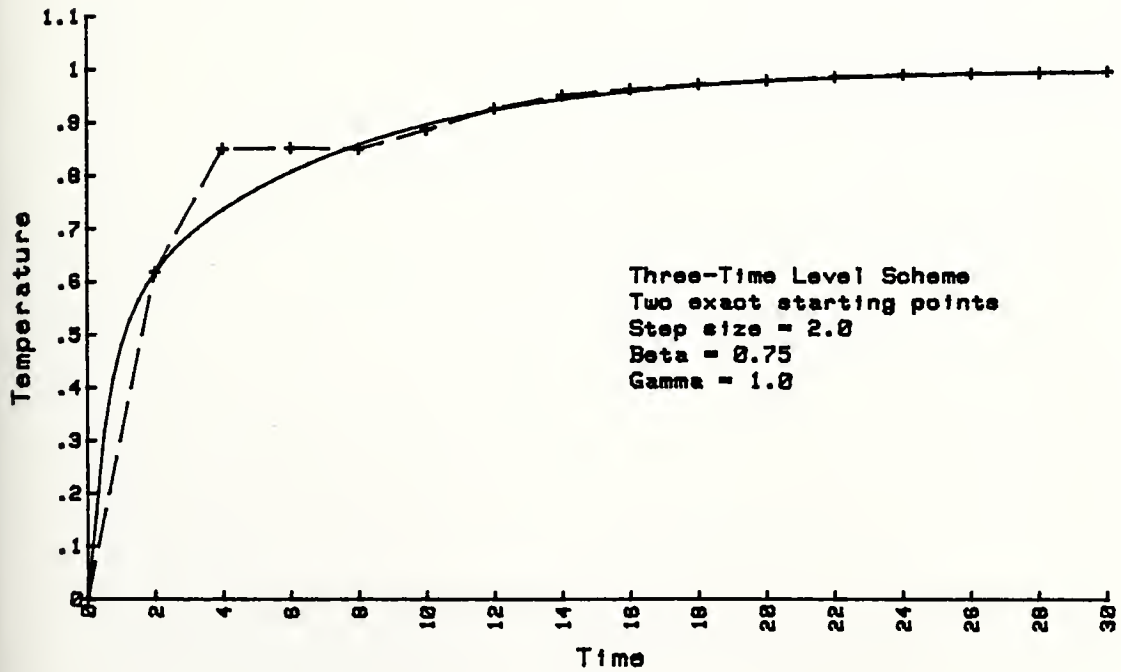
(d)

Figure 8. (continued)





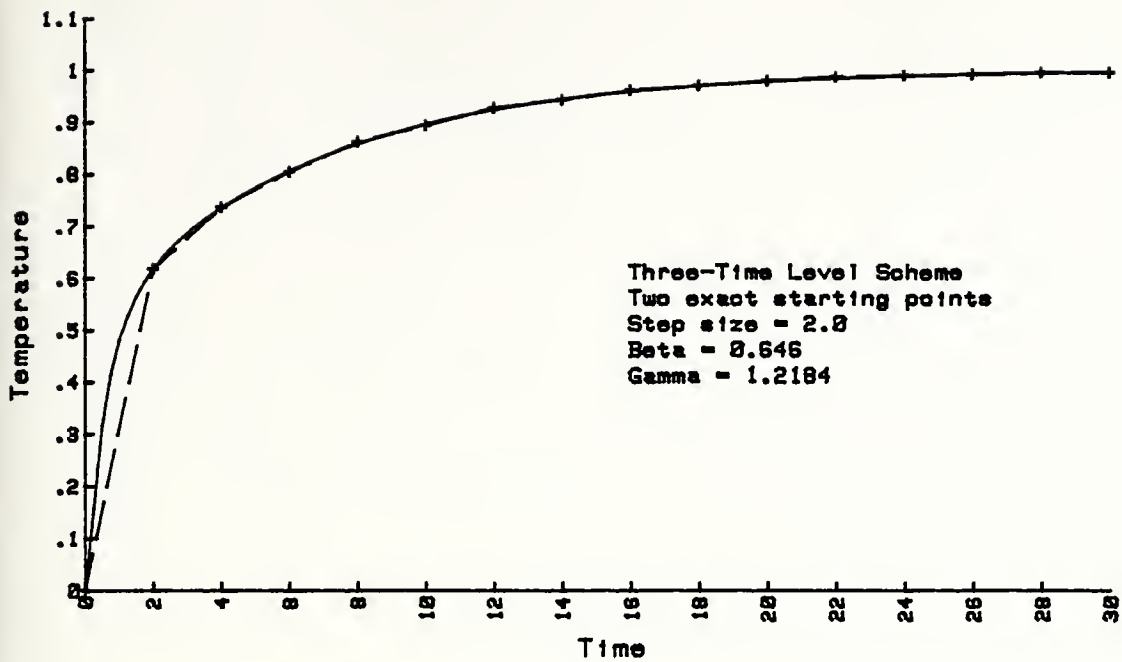
(a)



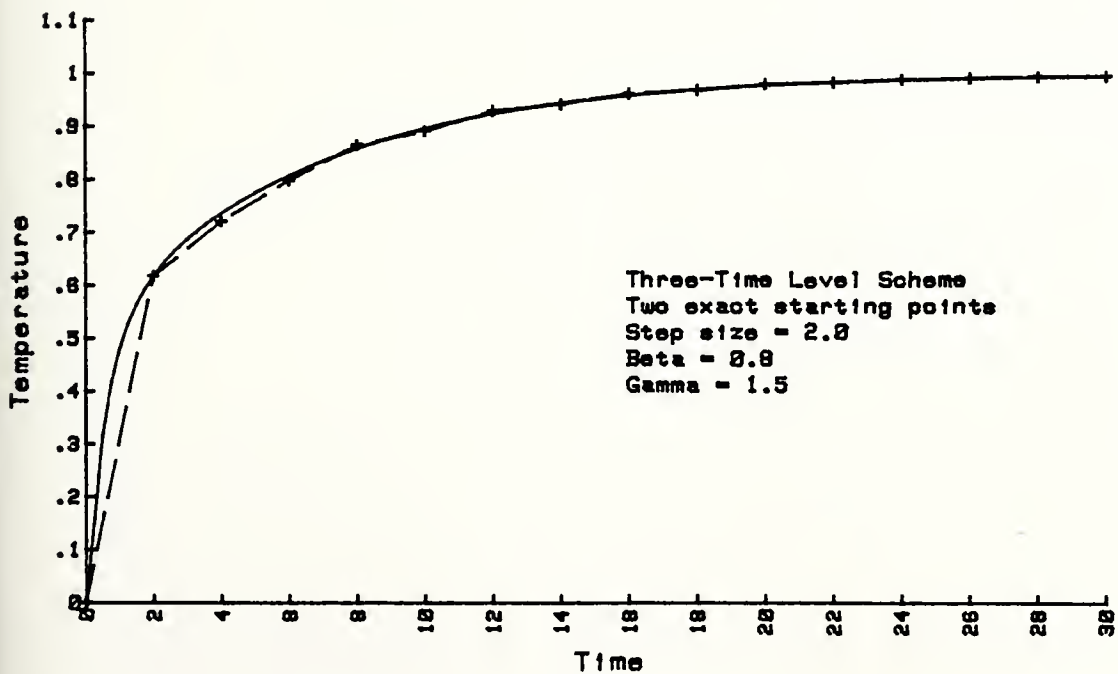
(b)

Figure 9.





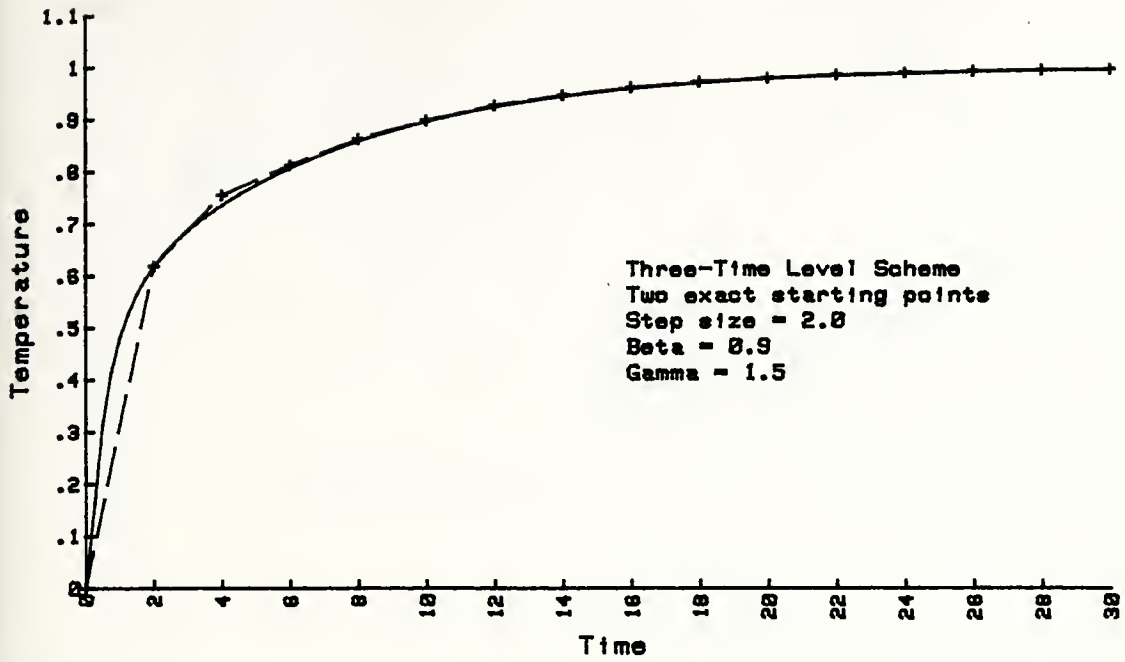
(c)



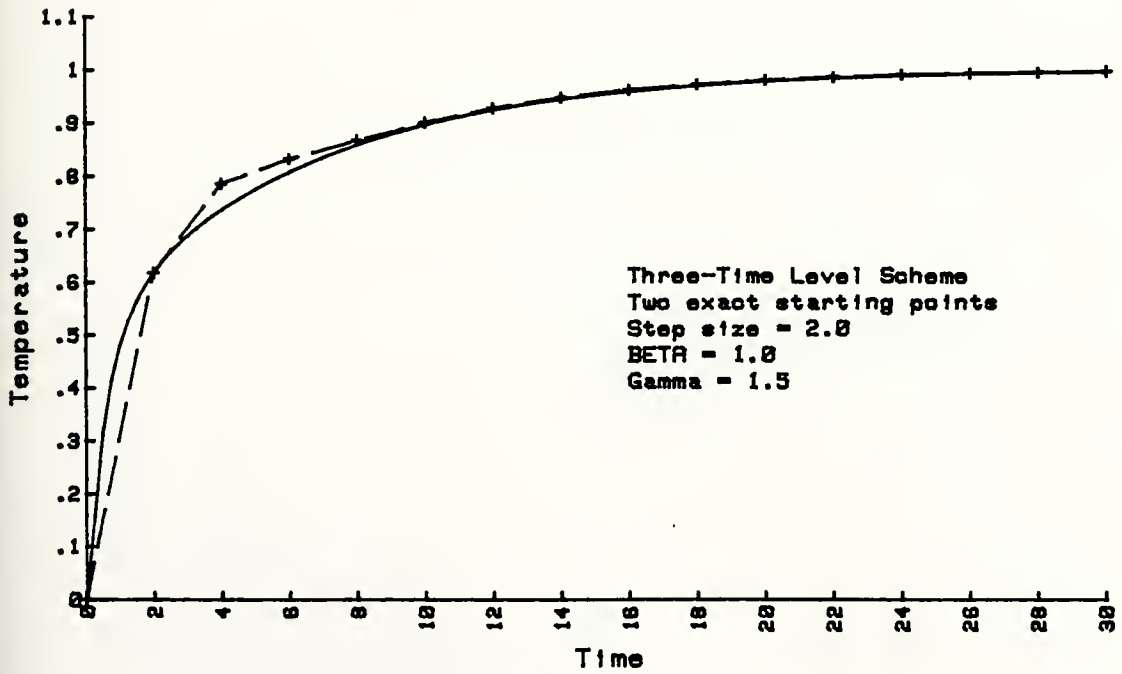
(d)

Figure 9. (continued)





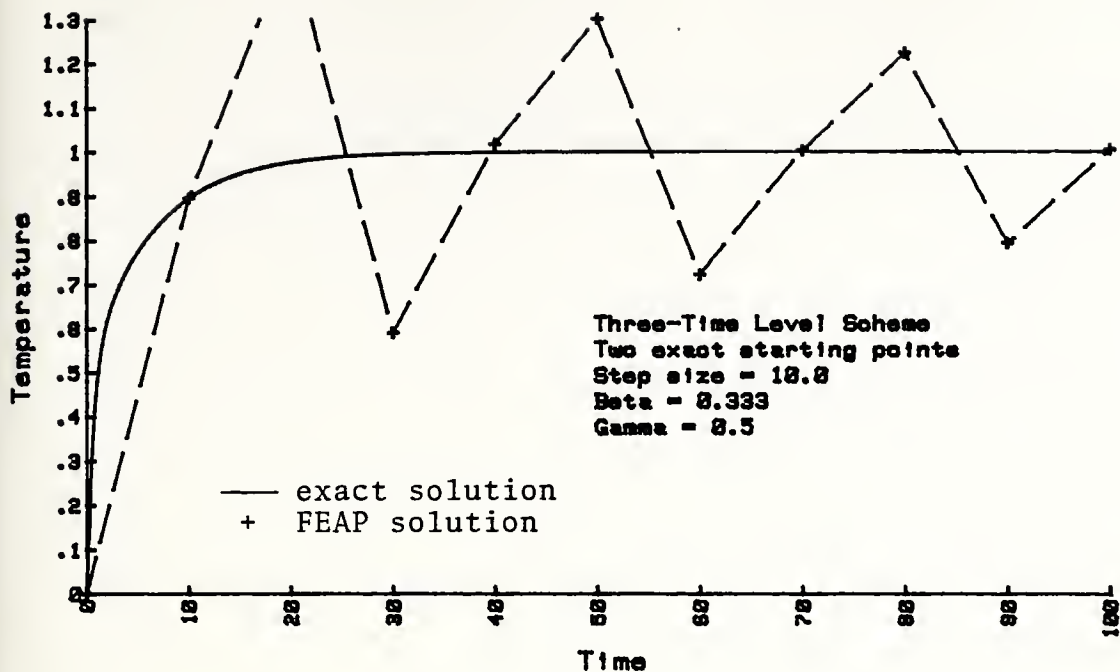
(e)



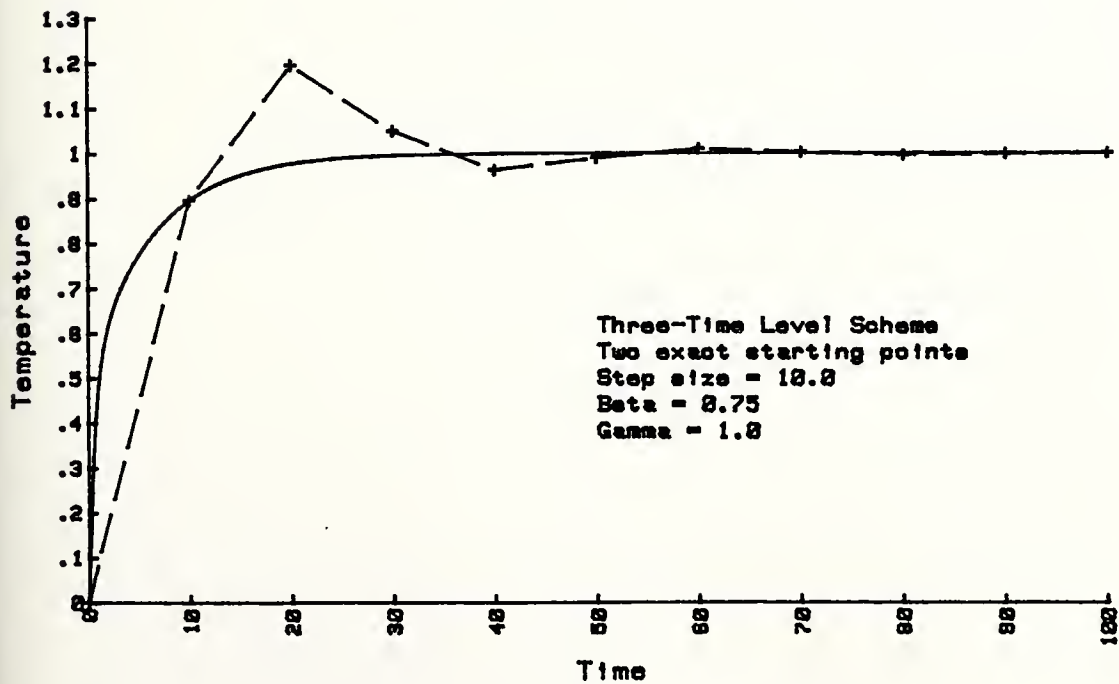
(f)

Figure 9. (continued)





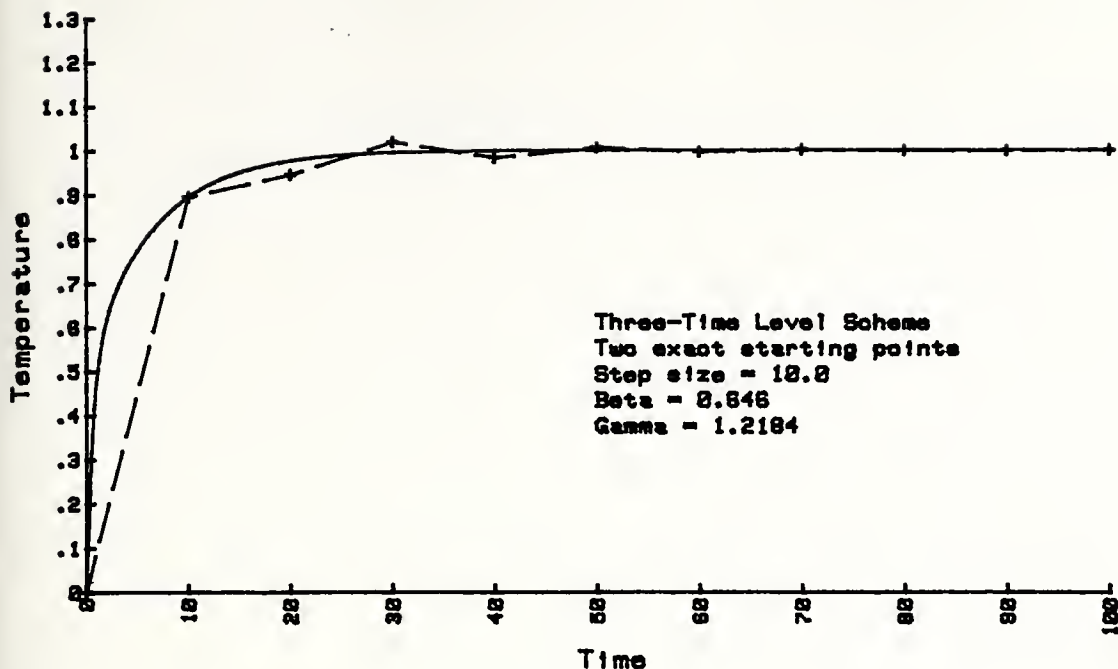
(a)



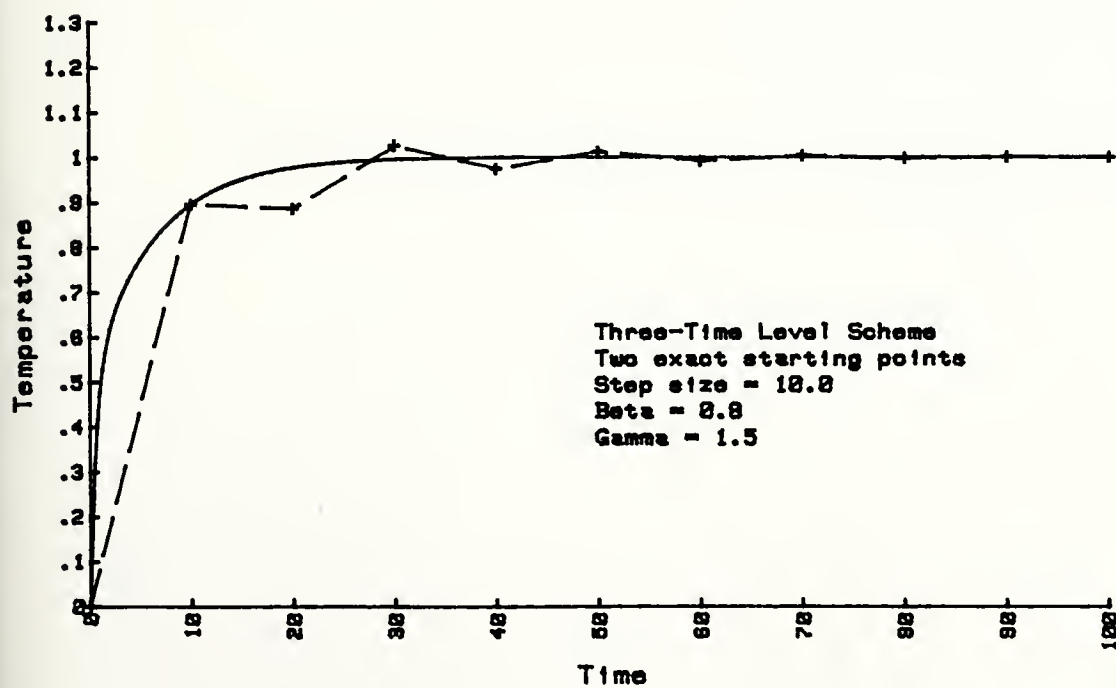
(b)

Figure 10.





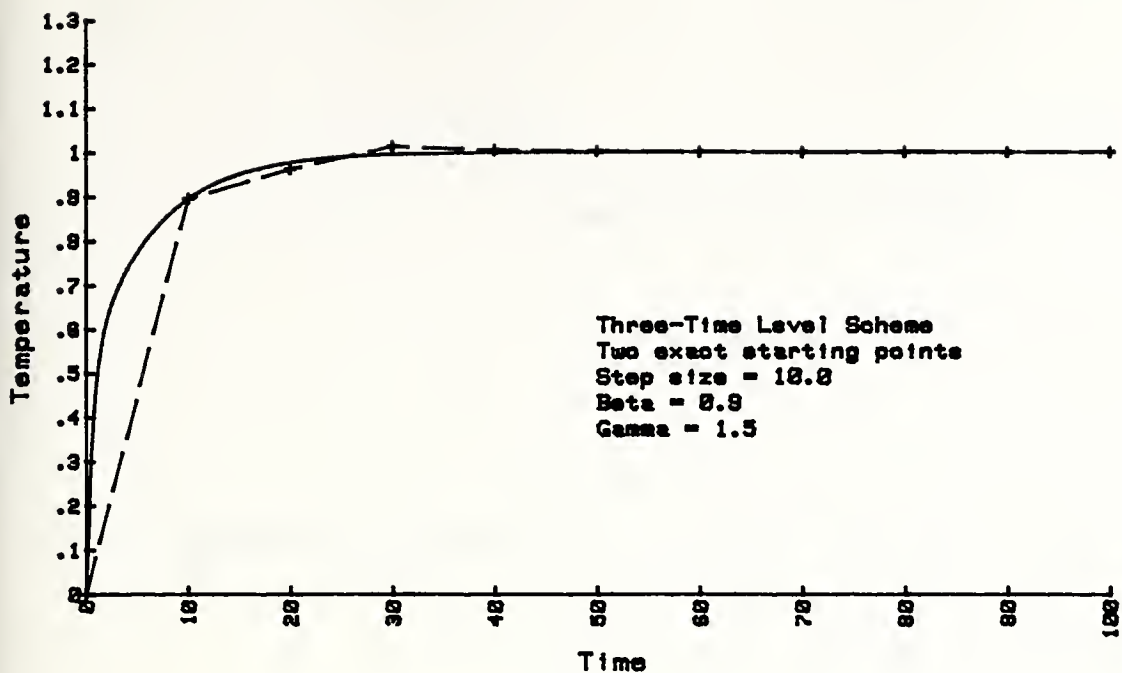
(c)



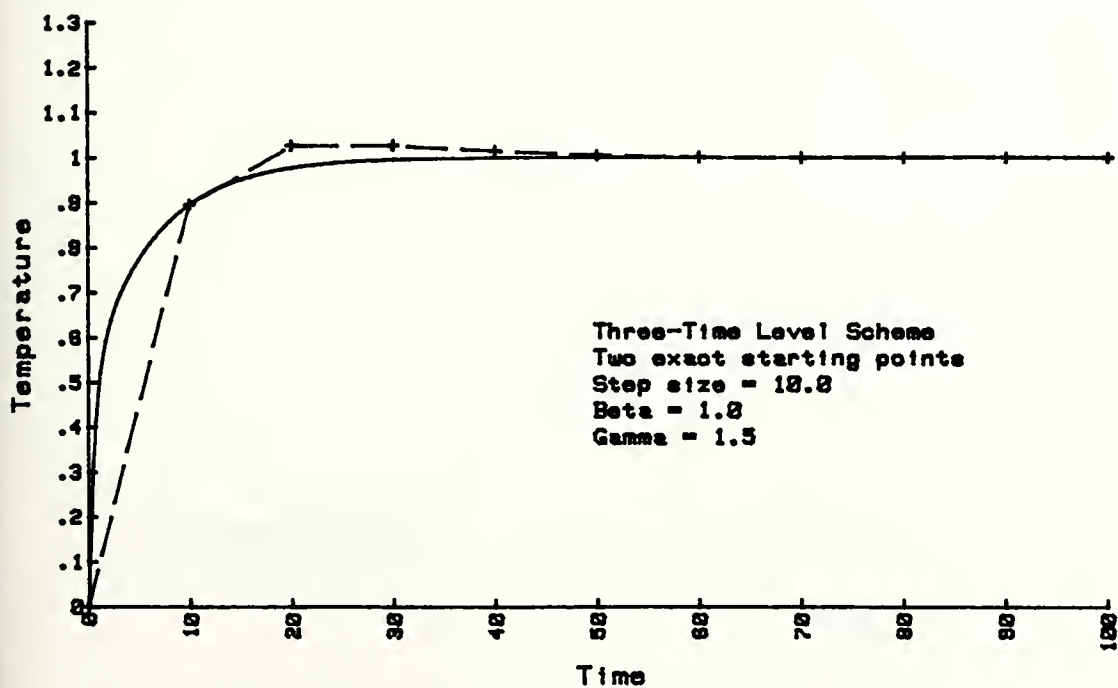
(d)

Figure 10. (continued)





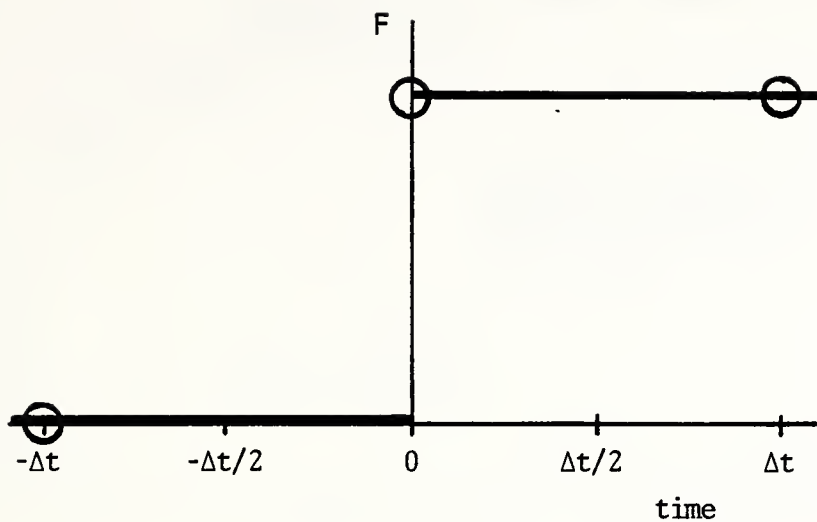
(e)



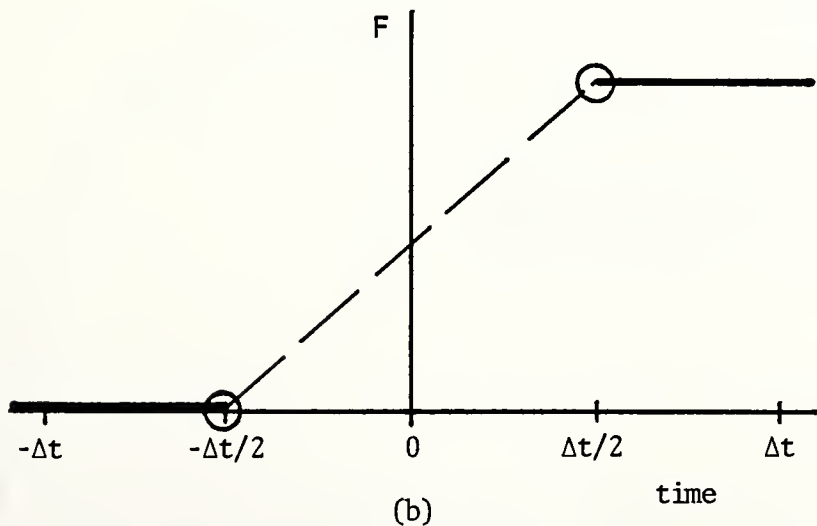
(f)

Figure 10. (continued)





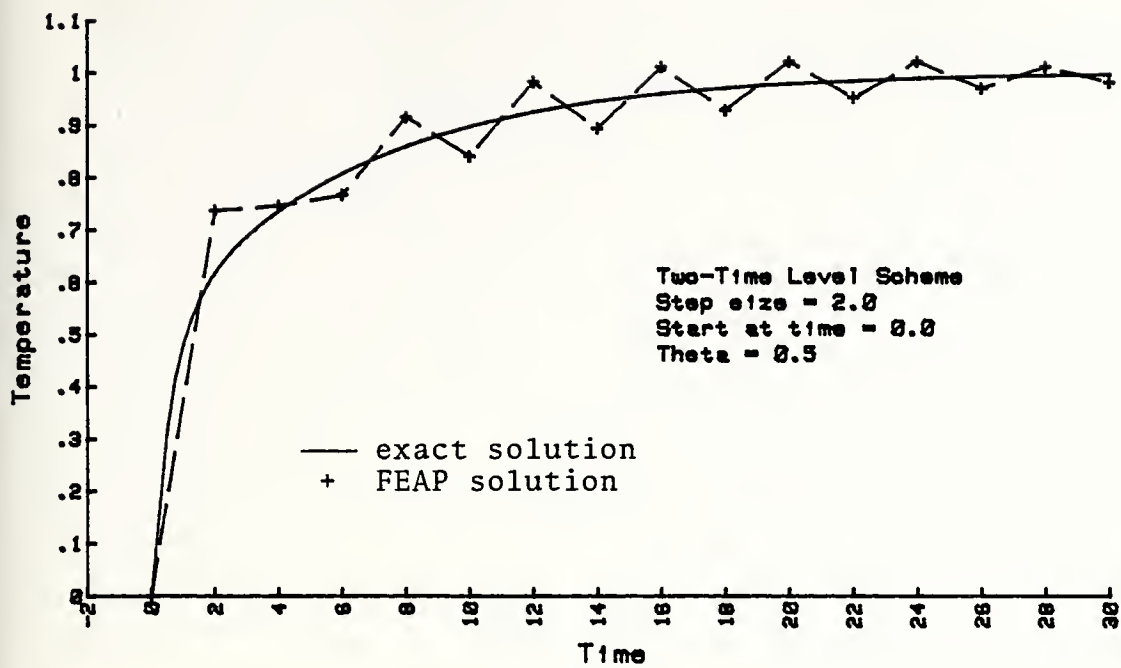
(a)



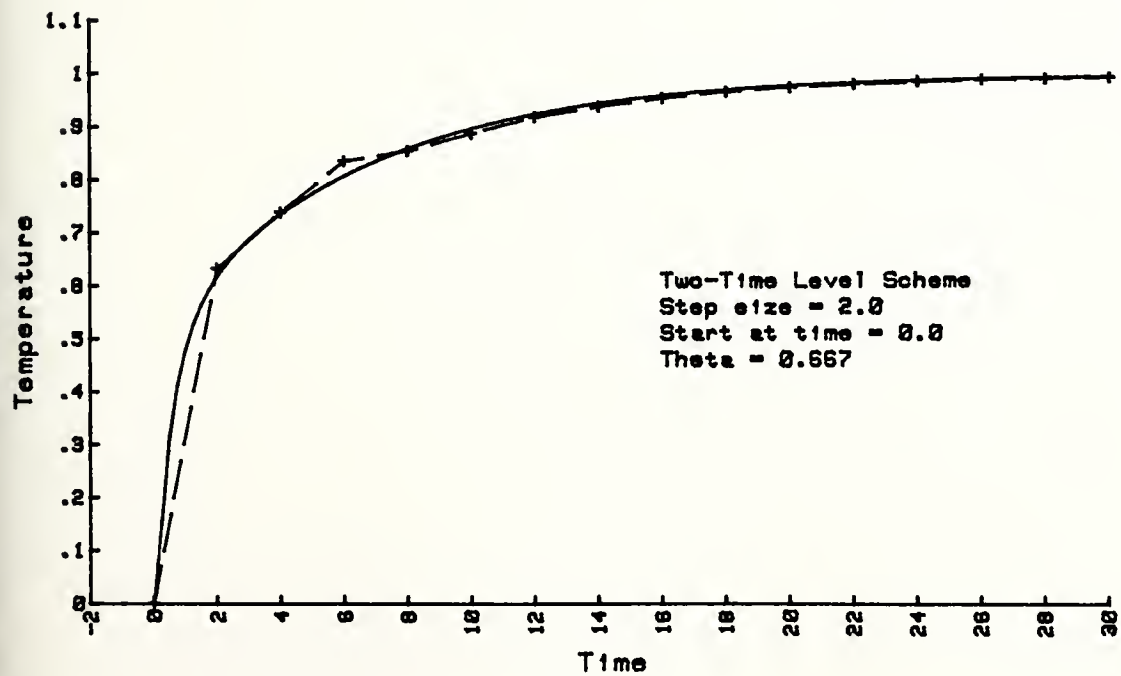
(b)

Figure 11. Step Change in the Loading Term





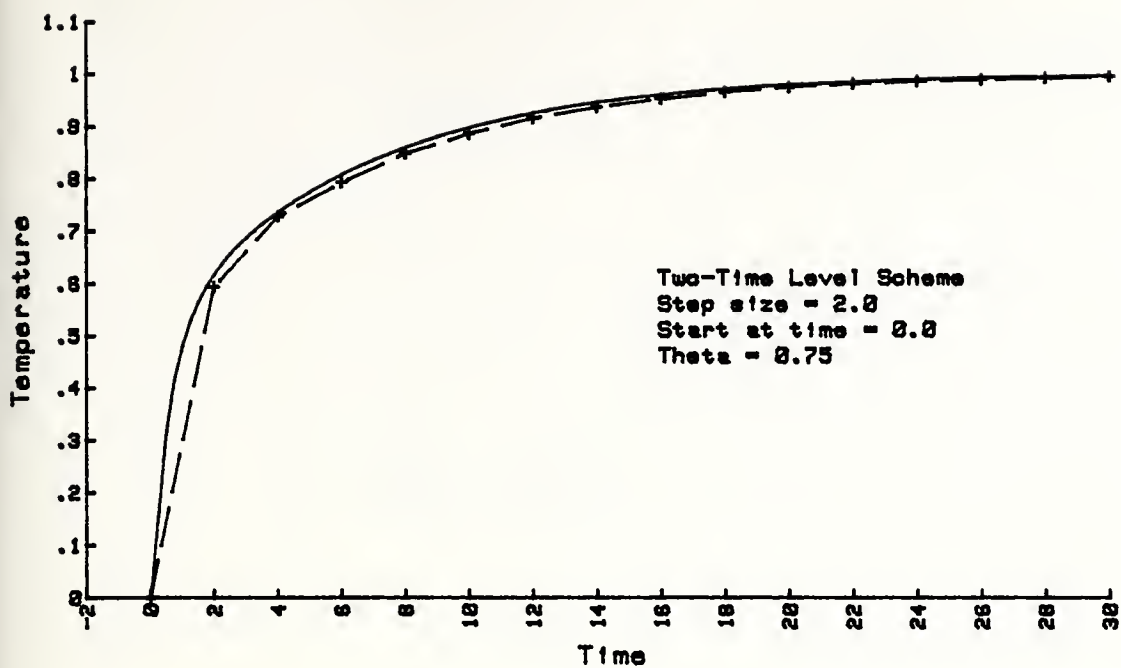
(a)



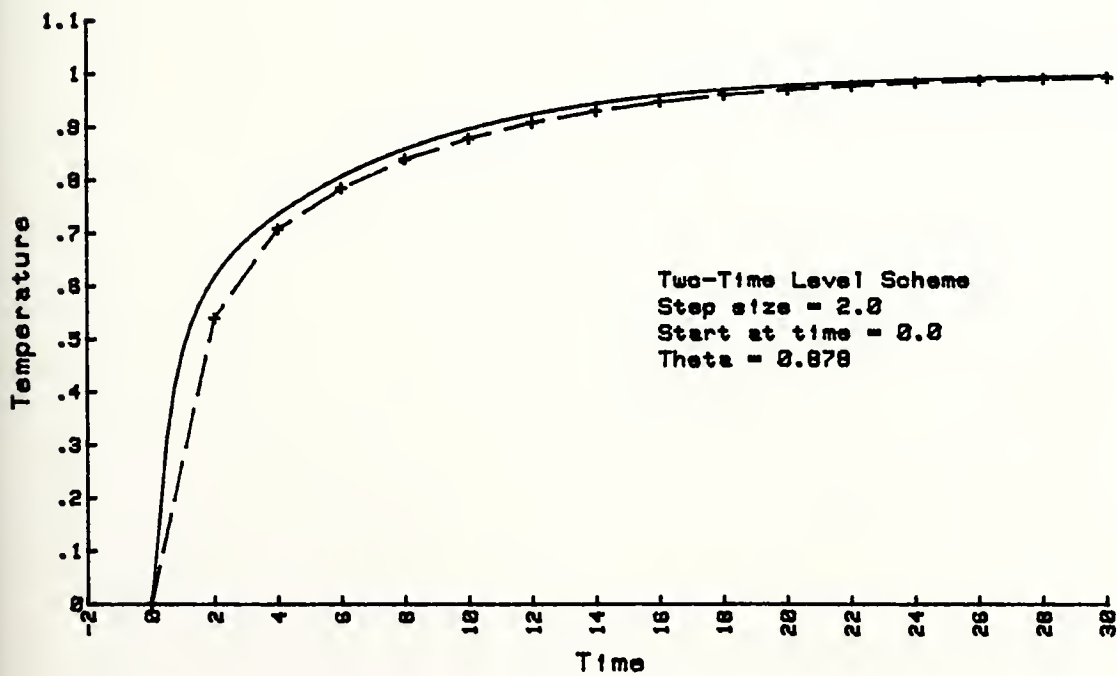
(b)

Figure 12.





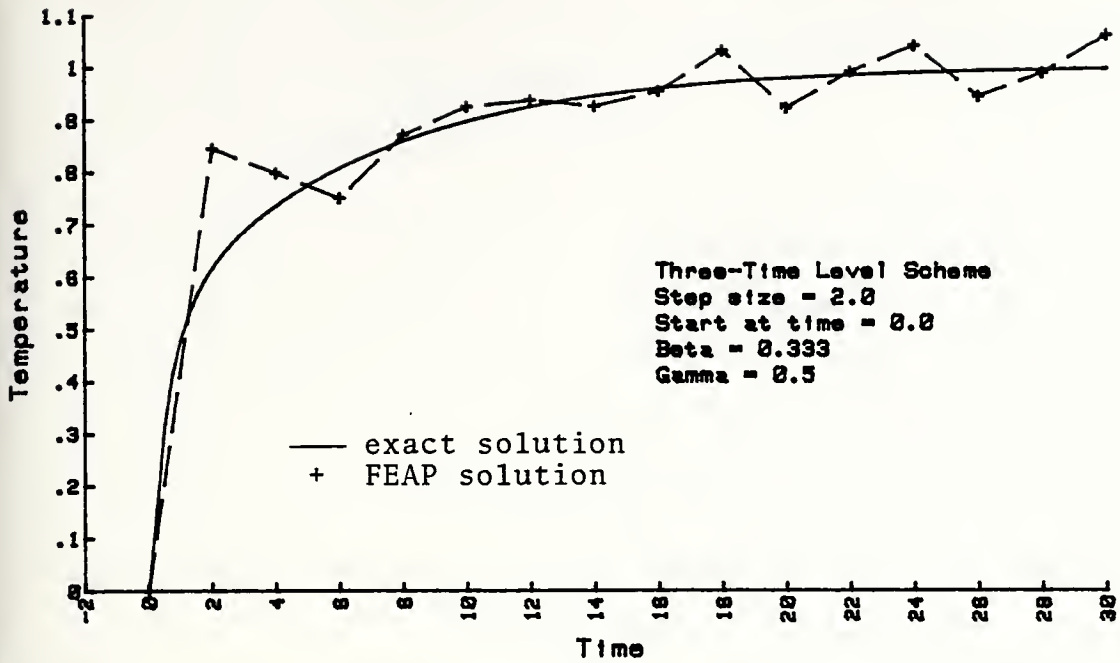
(c)



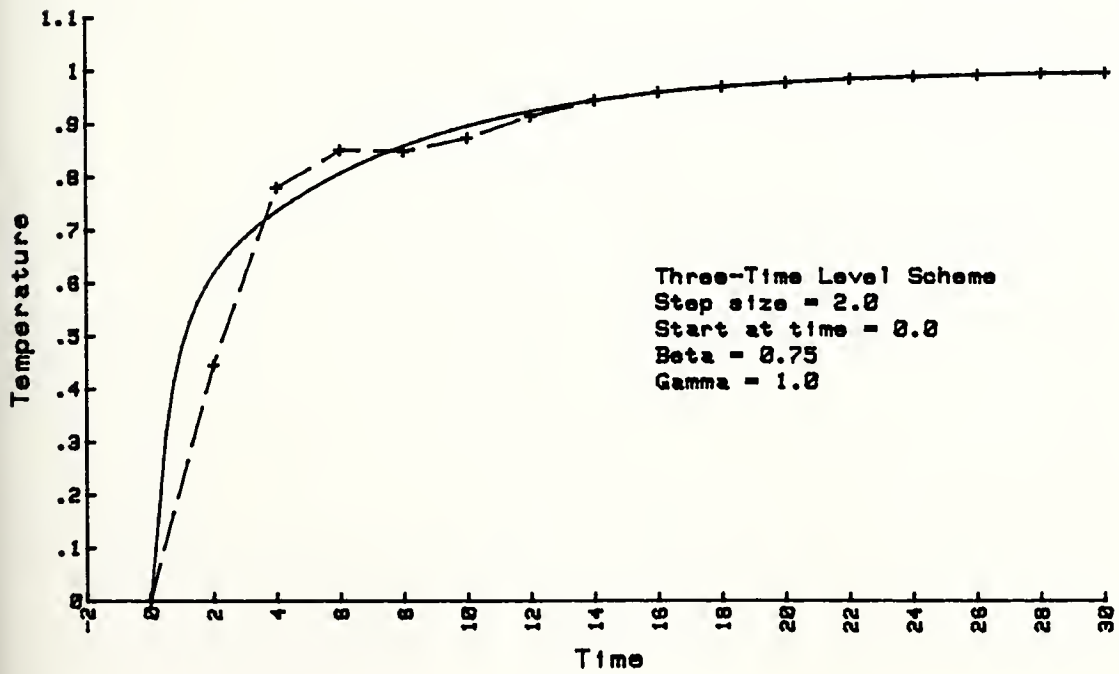
(d)

Figure 12. (continued)





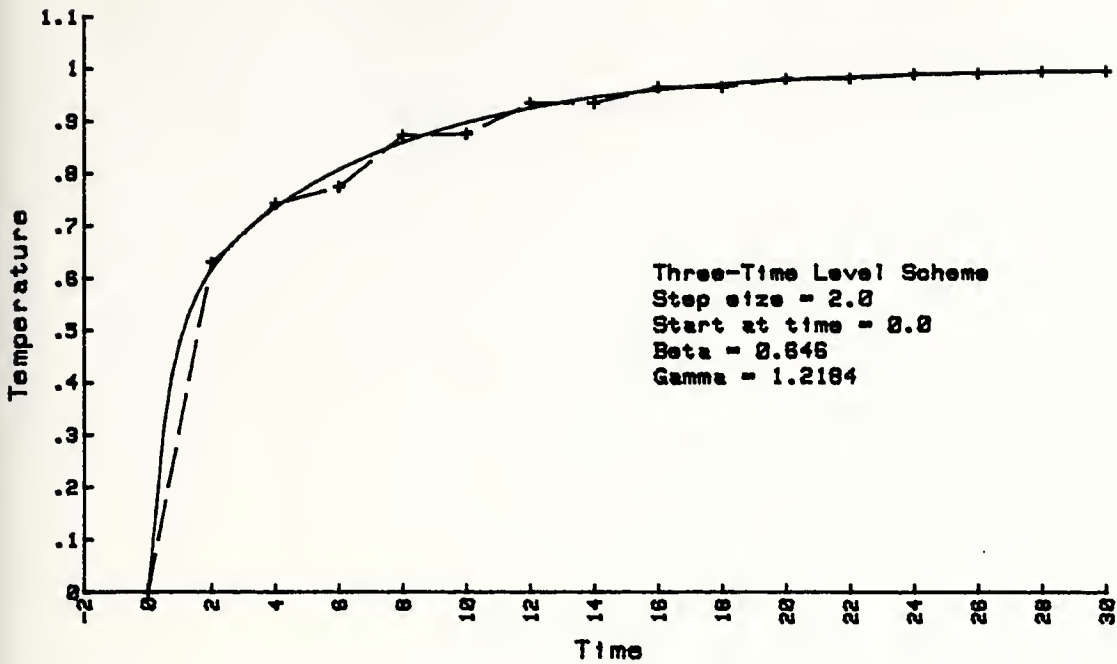
(a)



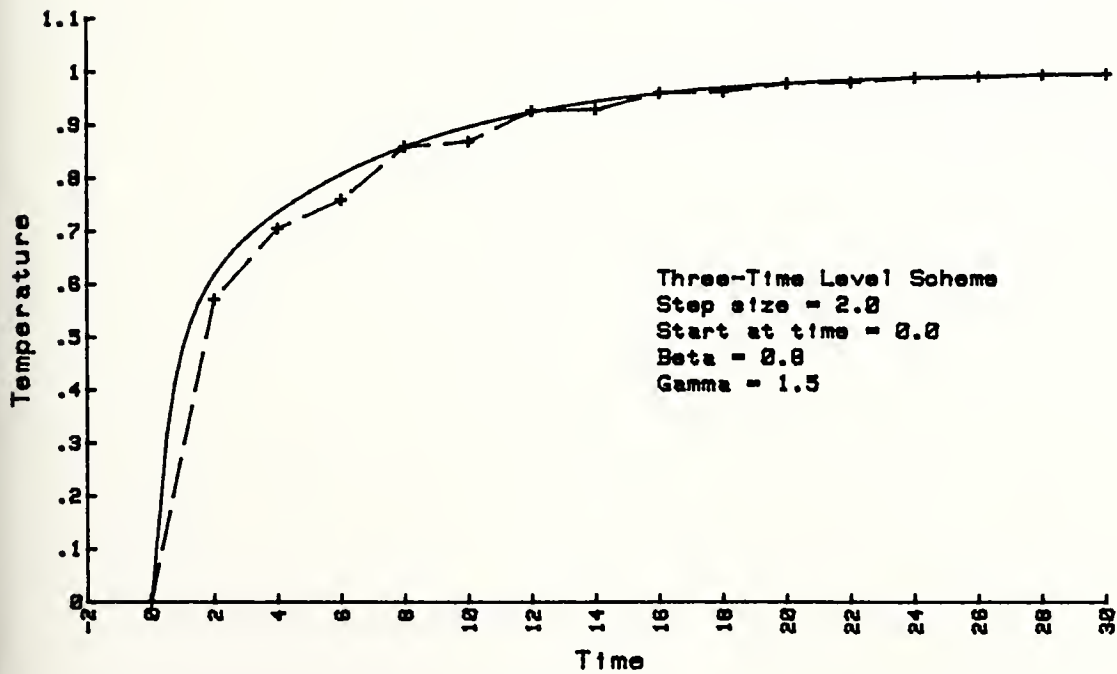
(b)

Figure 13.





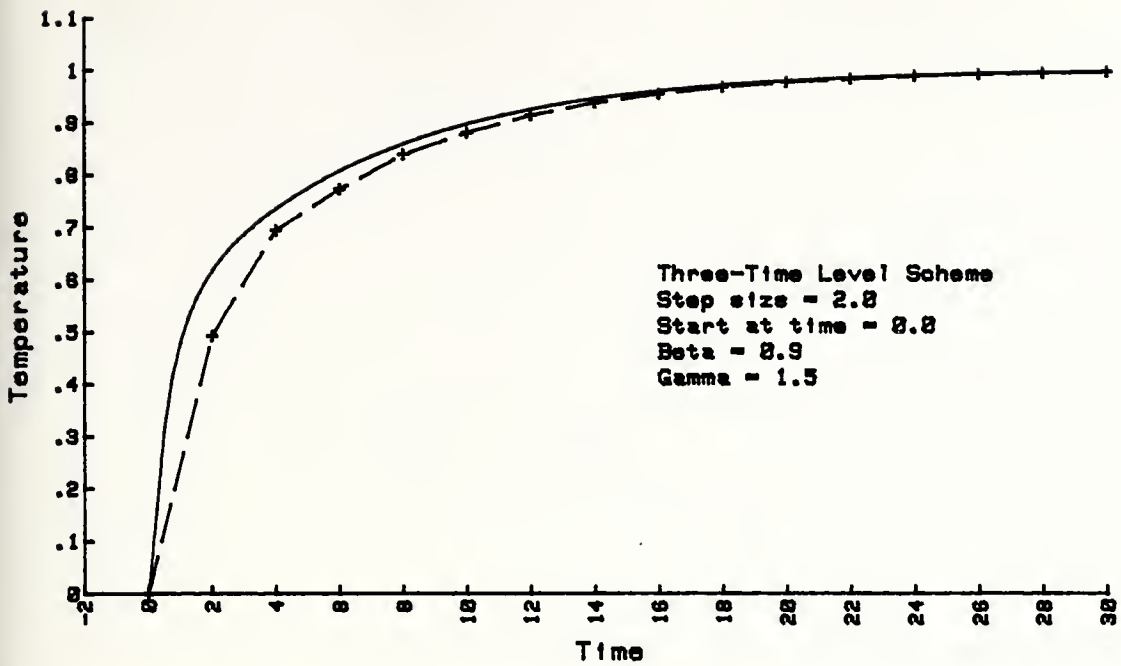
(c)



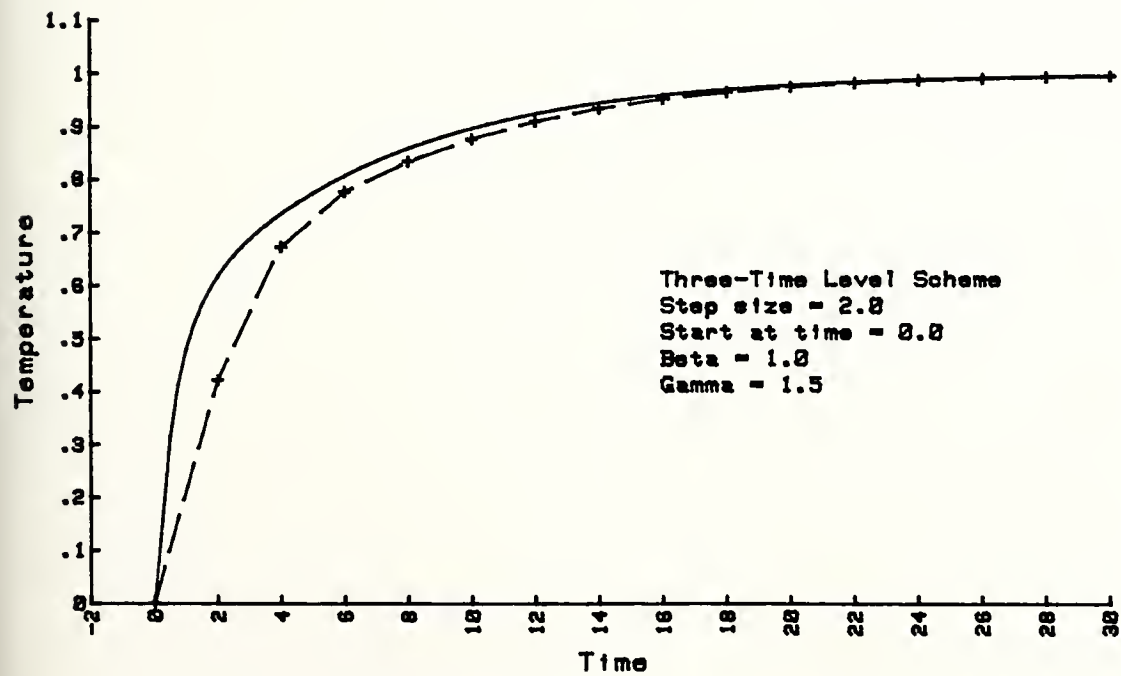
(d)

Figure 13. (continued)





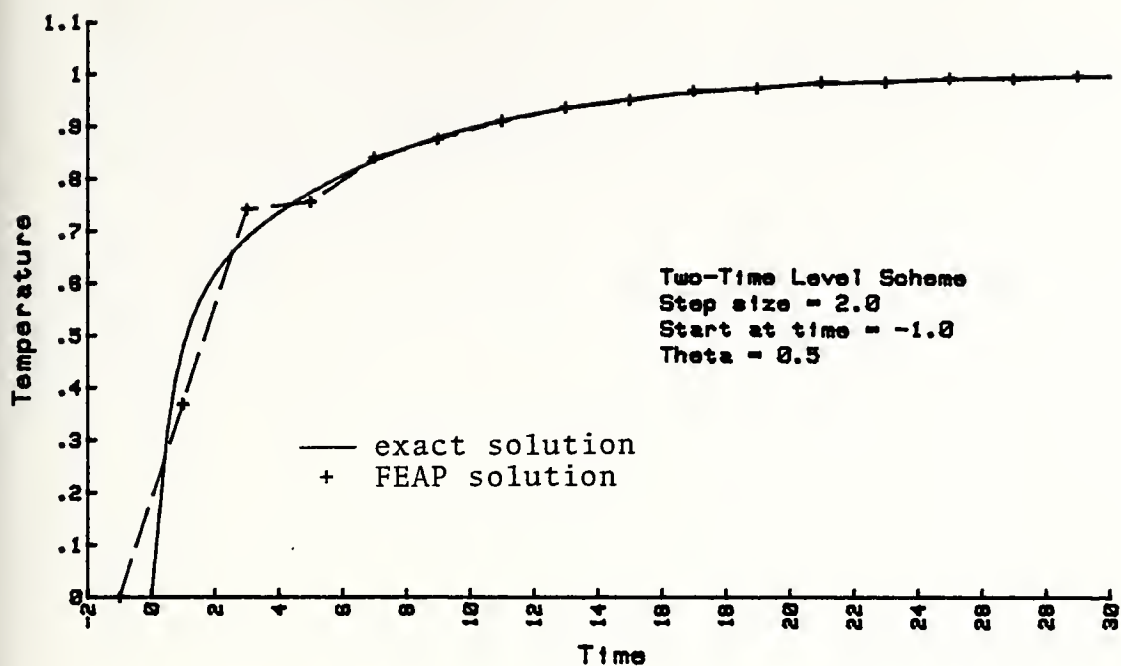
(e)



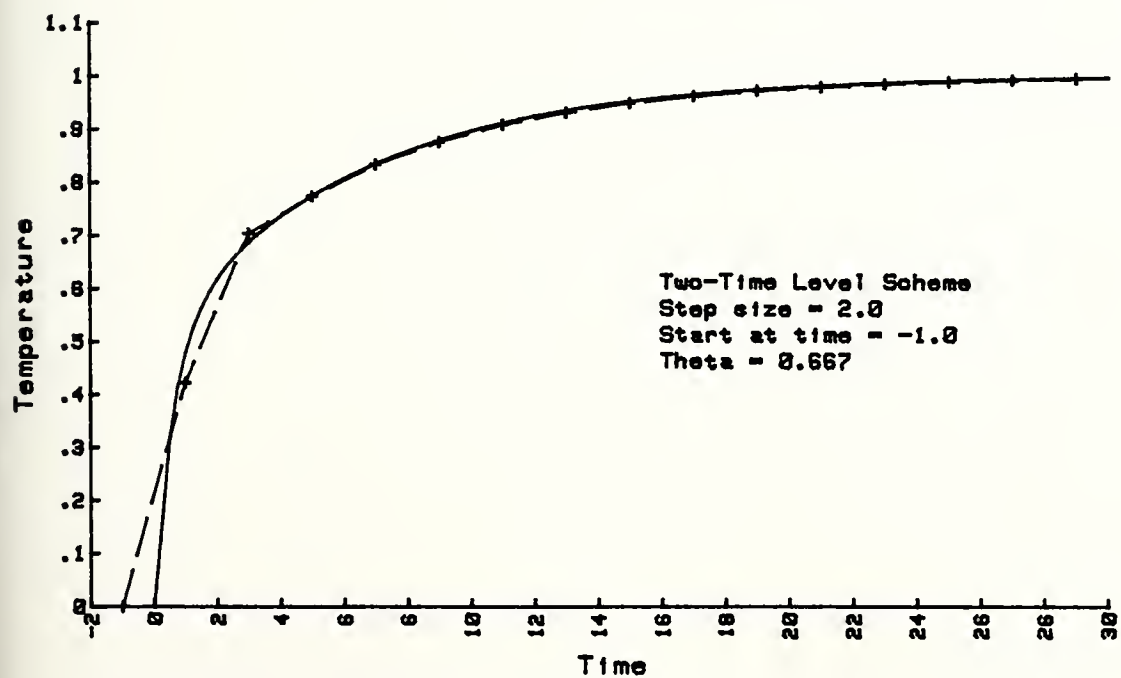
(f)

Figure 13. (continued)





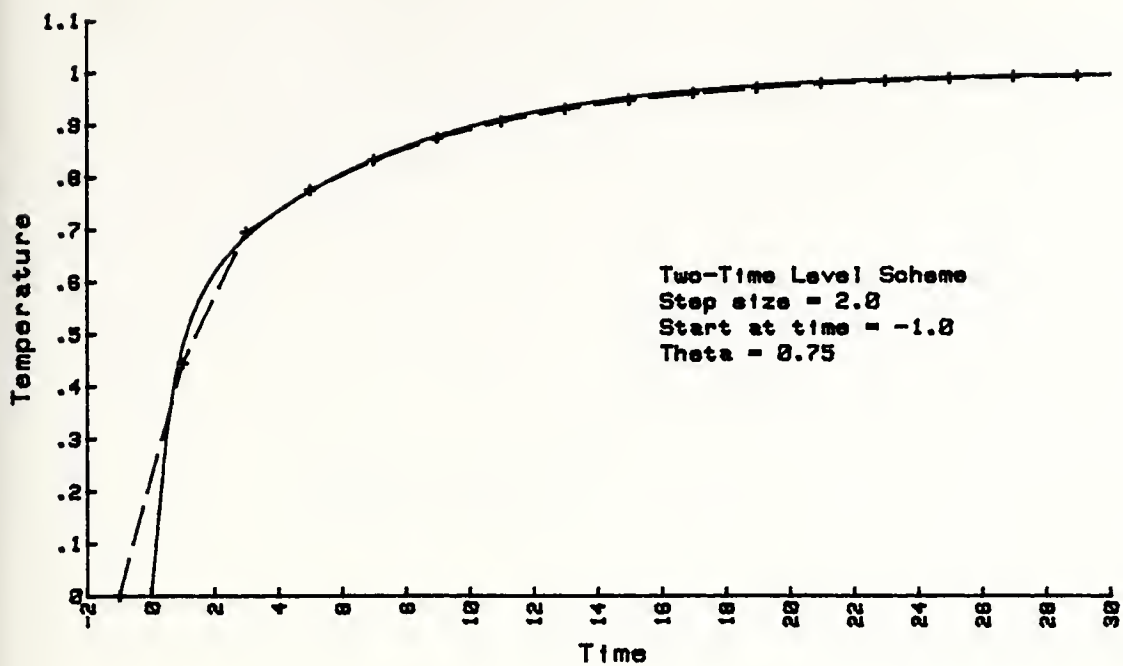
(a)



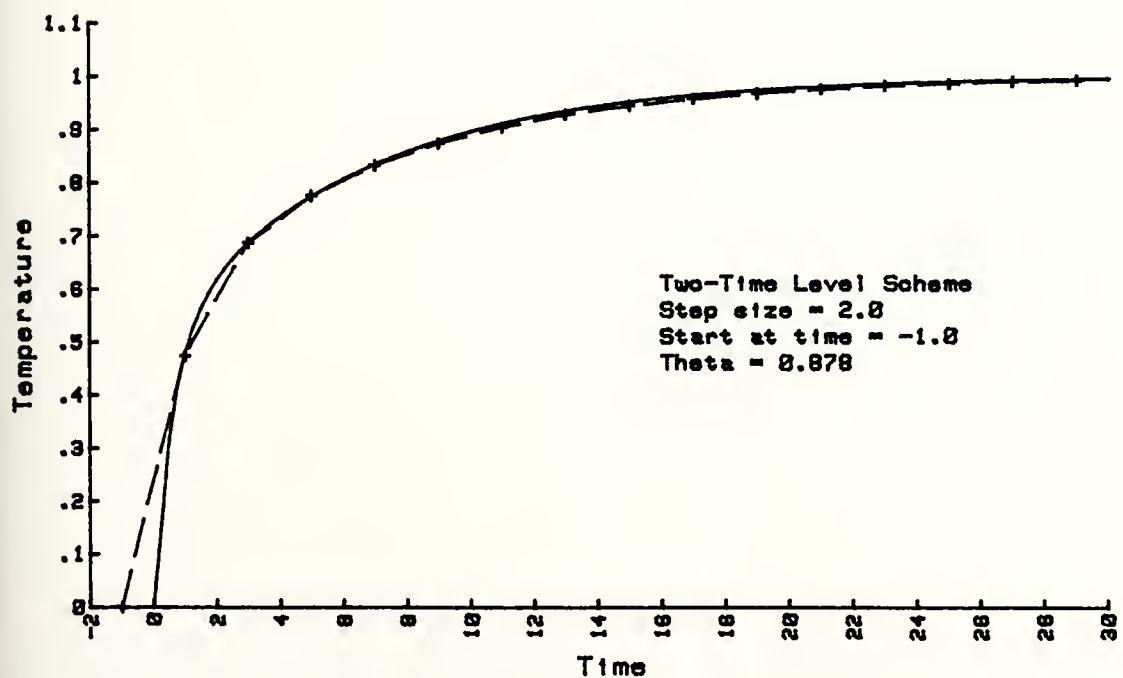
(b)

Figure 14.





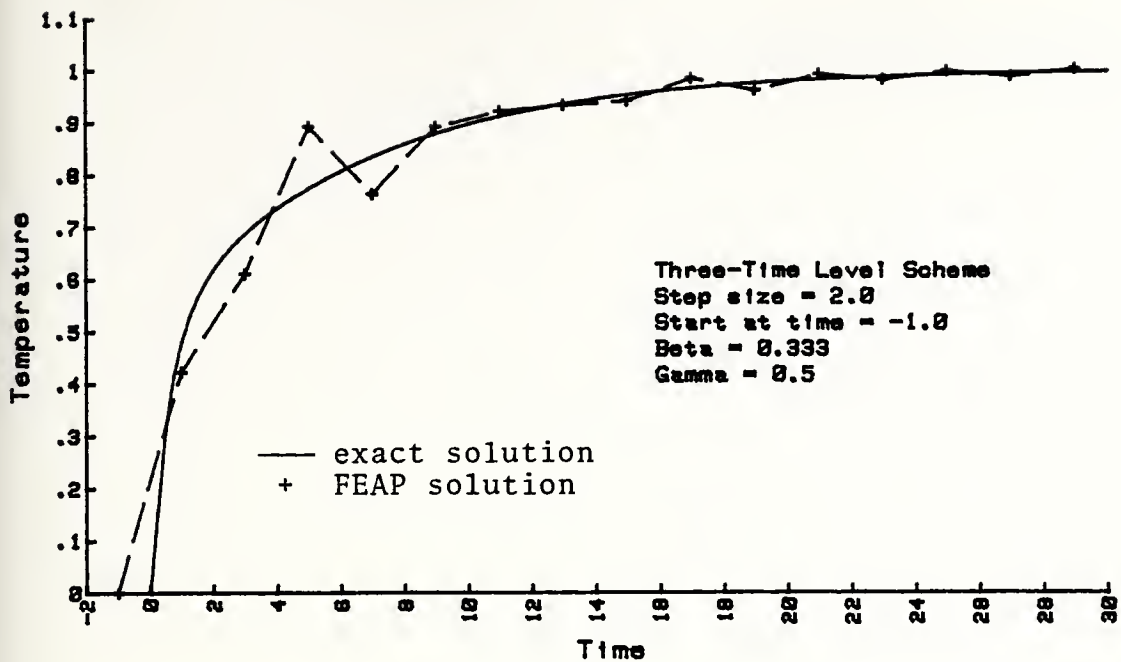
(c)



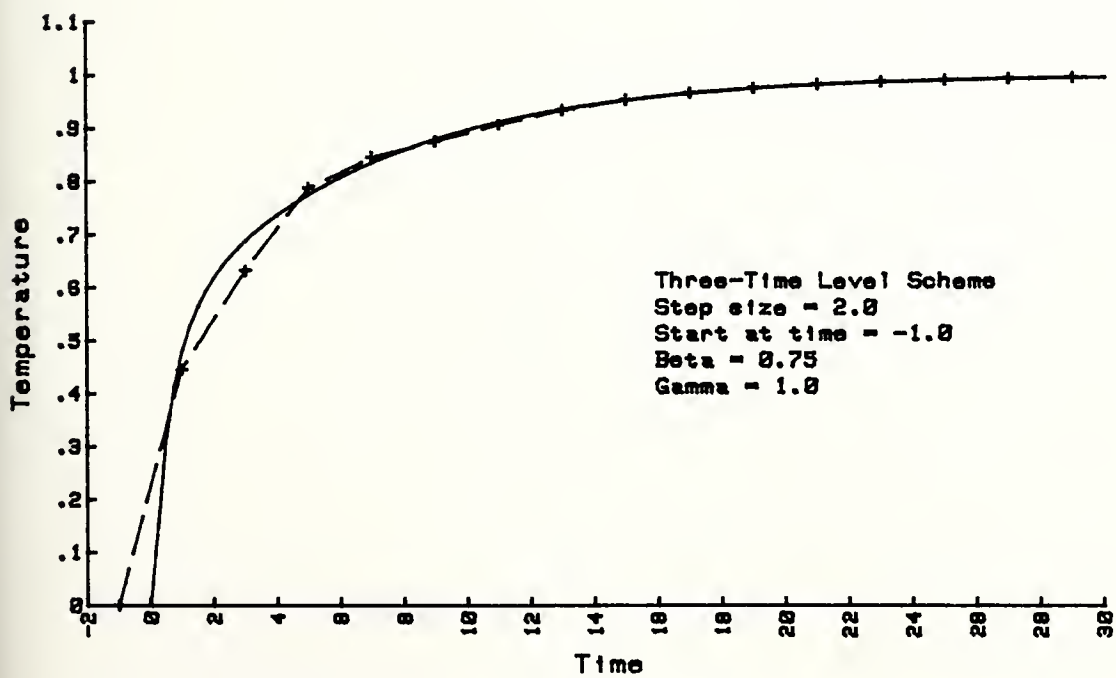
(d)

Figure 14. (continued)





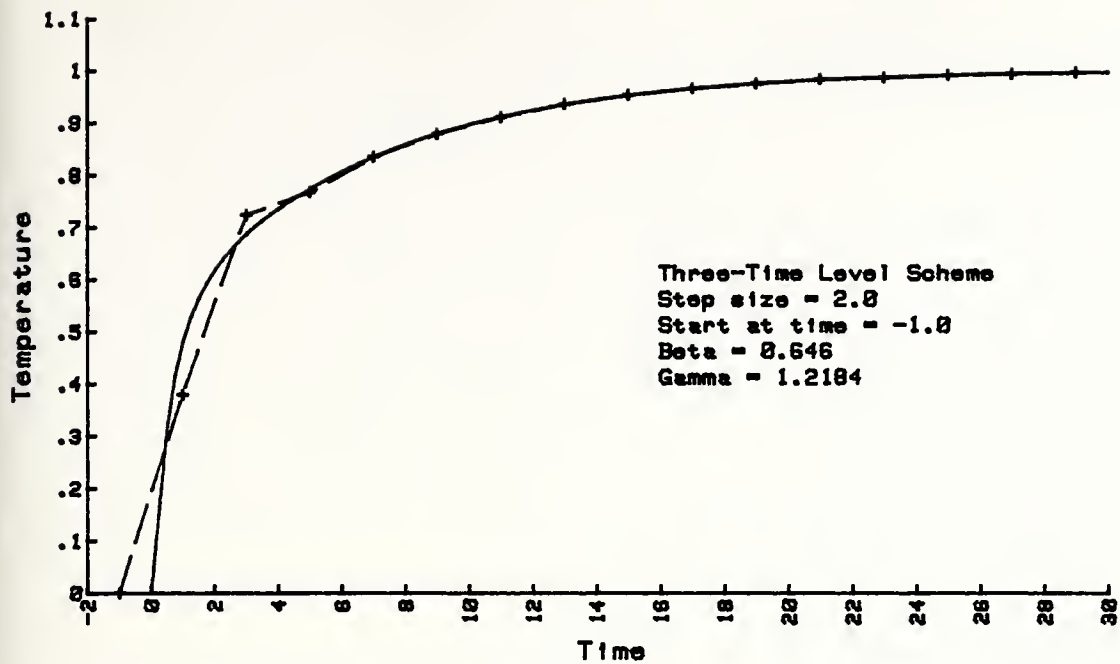
(a)



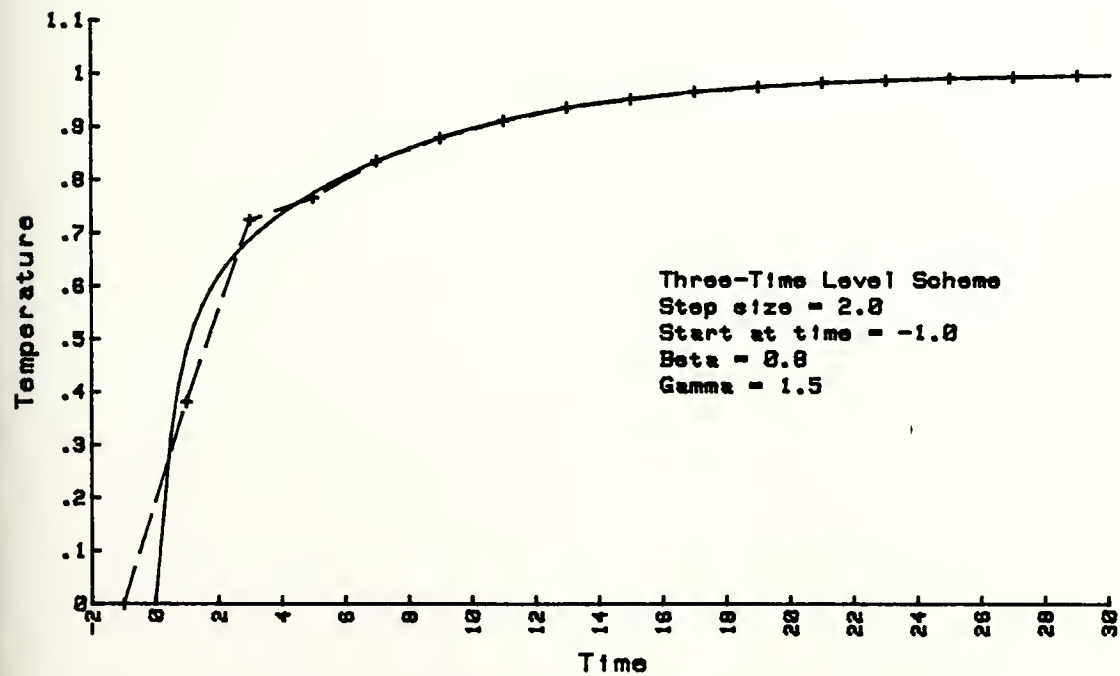
(b)

Figure 15.





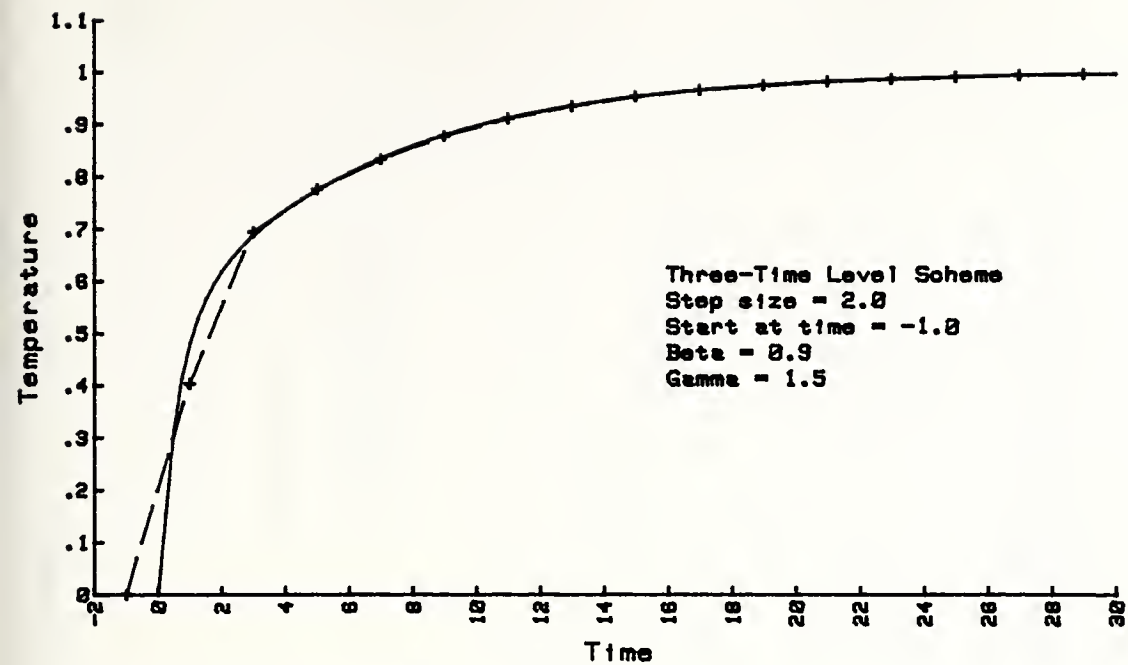
(c)



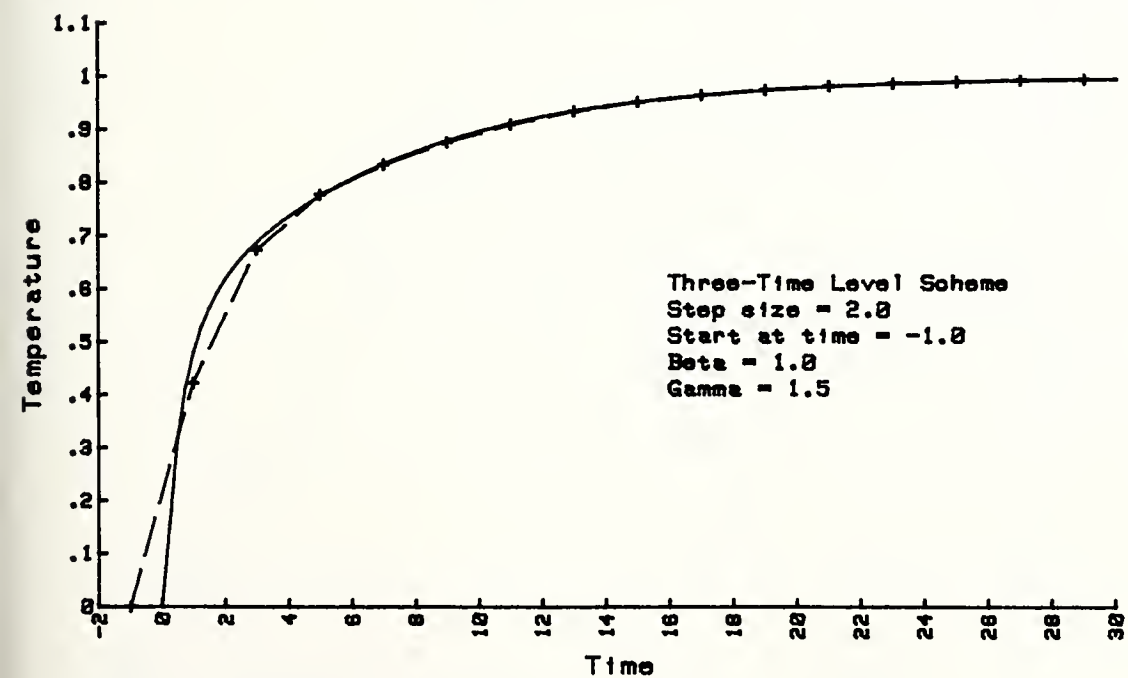
(d)

Figure 15. (continued)





(e)



(f)

Figure 15. (continued)



# Hollow Cylinder

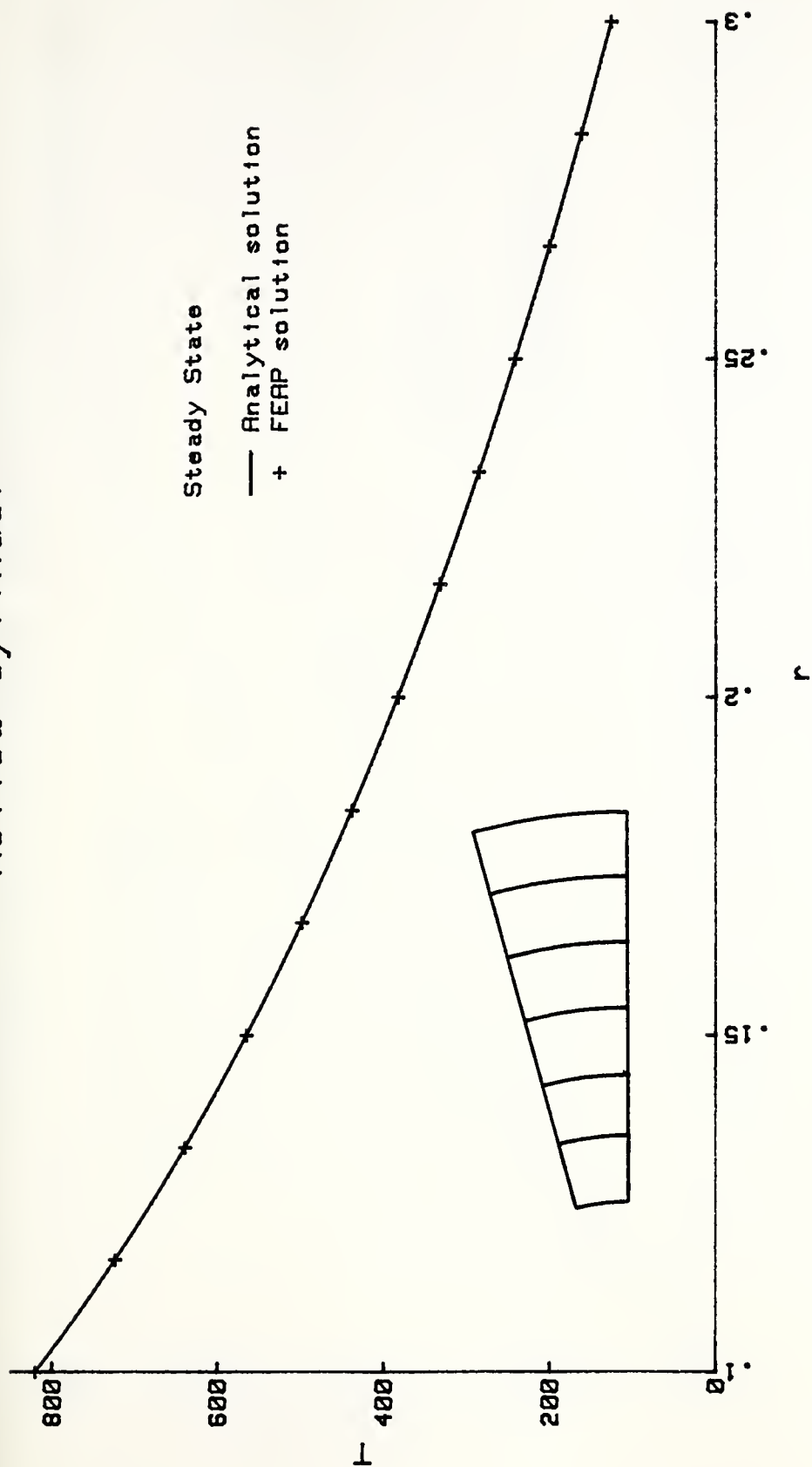


Figure 16. Radial Temperature Distribution in a Hollow Cylinder. The temperature  $T$  and the radius  $r$  are expressed in degrees Celsius and meters, respectively.



# Slab with Variable Conductivity

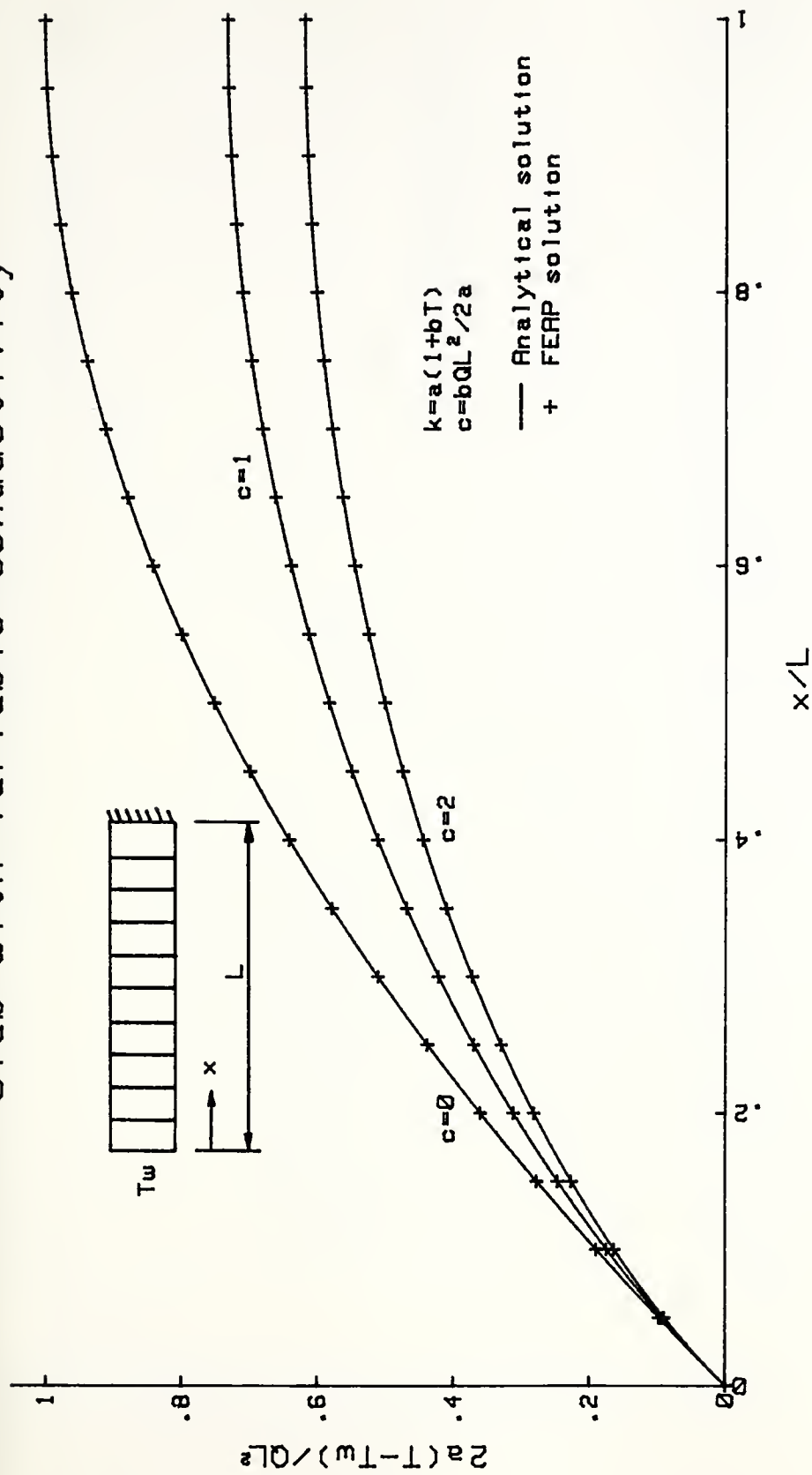


Figure 17. Effect of Linear Conductivity Variation on the Temperature of a Plate with Uniform Heat Generation.



# Hollow Cylinder

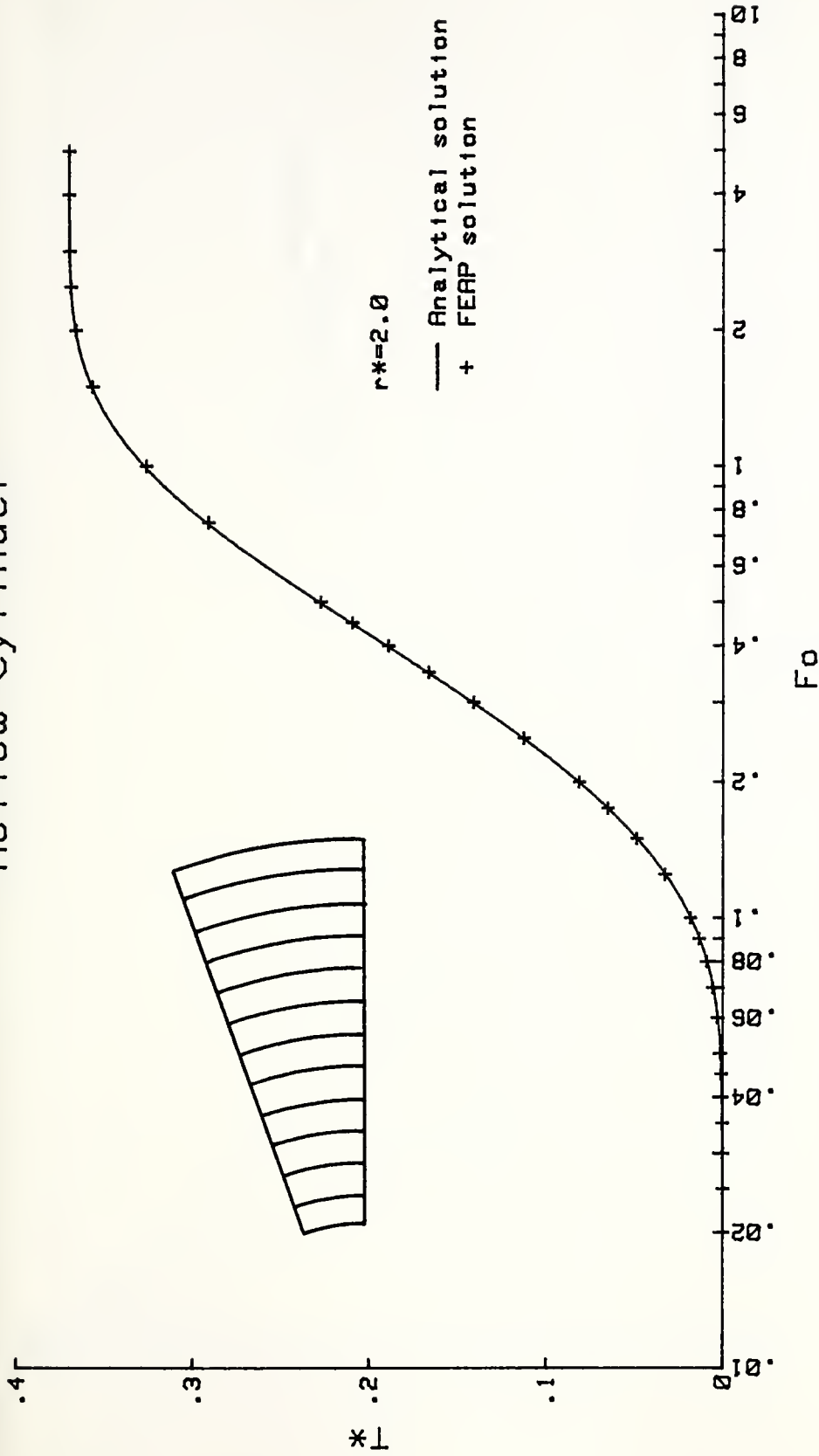


Figure 18. Unsteady Temperature of the Center Interior Points in a Hollow Cylinder.



# Hollow Cylinder

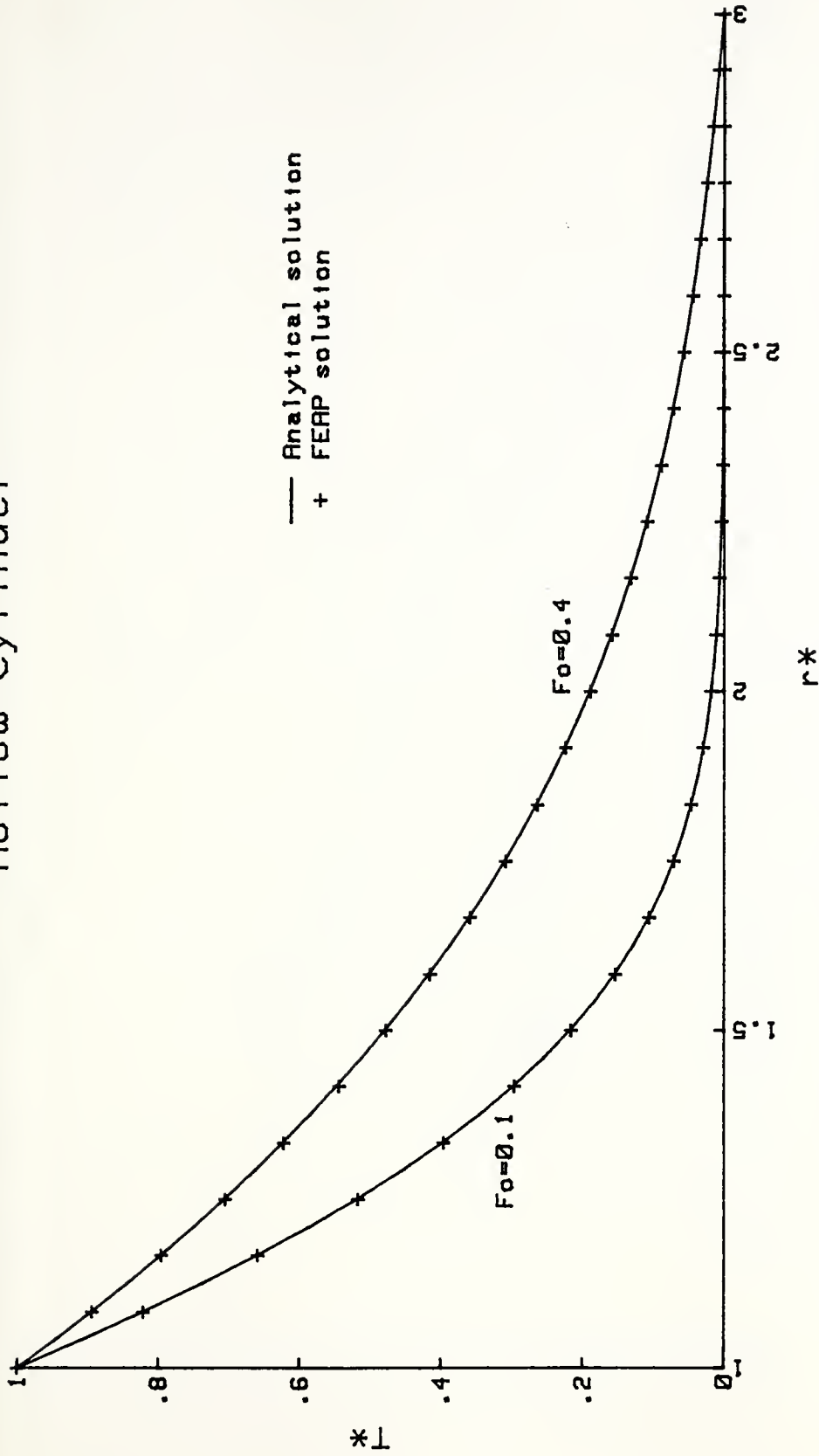


Figure 19. Unsteady Radial Temperature Distribution in a Hollow Cylinder



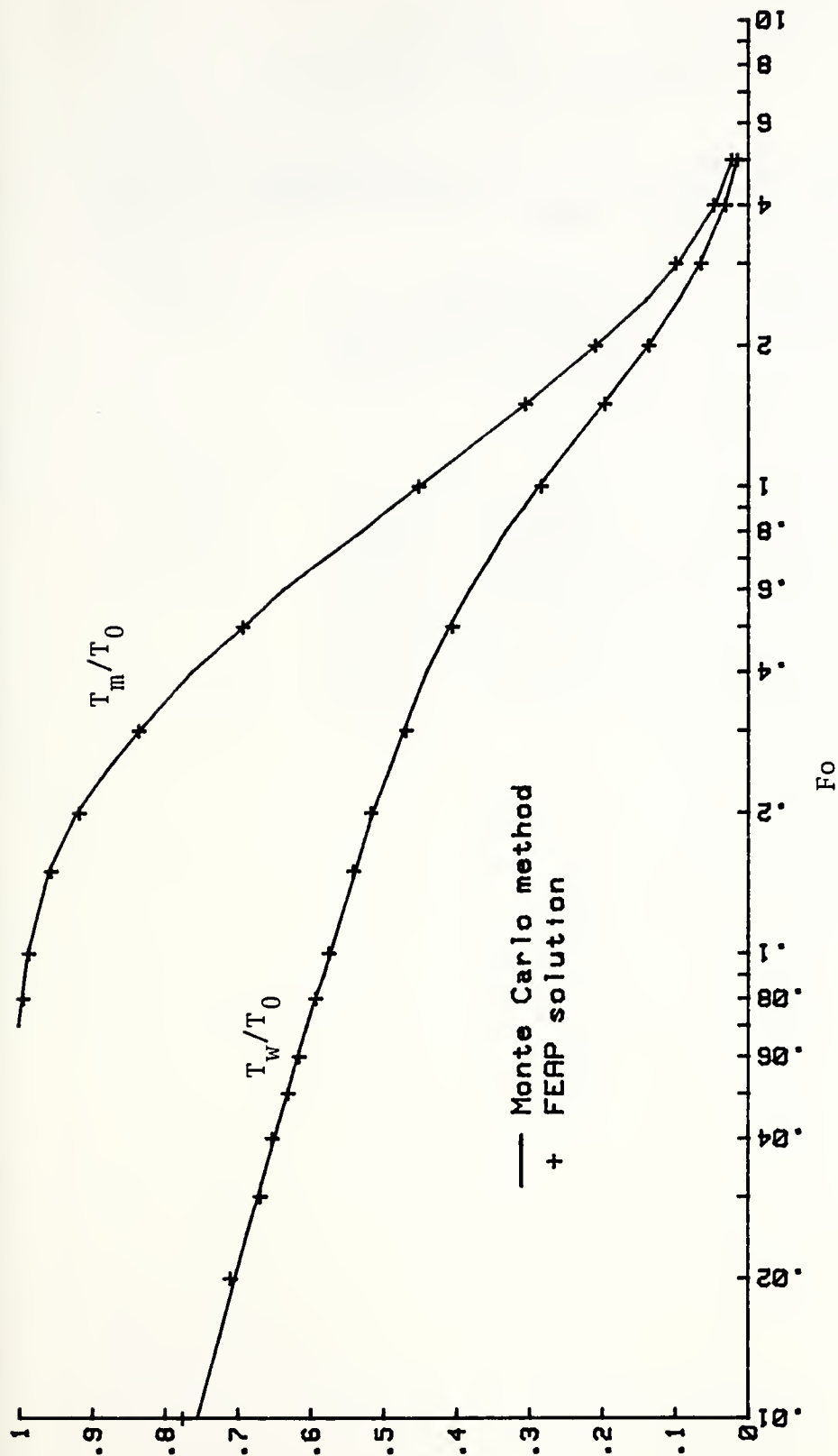


Figure 20. Unsteady Temperature Distribution in a Slab with Radiation/Convection Boundary Conditions.



# APPENDIX A

## USER INSTRUCTIONS FOR FEAP

### A.1 TITLE AND CONTROL INFORMATION

..... TITLE CARD-FORMAT(20A4)  
 THE TITLE CARD ALSO SERVES AS A START OF PROBLEM CARD.  
 THE FIRST FOUR (4) COLUMNS MUST CONTAIN THE START  
 WORD FEAP

COLUMNS	DESCRIPTION	VARIABLE
1 TC 4	MUST CONTAIN FEAP	TITL(1)
5 TO EC	ALPHANUMERIC INFORMATION TO BE PRINTED WITH OUTPUT AS PAGE HEADER.	TITL(I), I=2,20

..... CONTROL CARD-FORMAT(7I5)

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	NUMBER OF NODES	NUMNP
6 TC 10	NUMBER OF ELEMENTS	NUMEL
11 TC 15	NUMBER OF MATERIAL SETS	NUMMAT
16 TC 20	SPATIAL DIMENSION (MAXIMUM, UP TO 3)	NDM
21 TC 25	NUMBER OF UNKNOWN PER NODE (MAXIMUM, UP TO 6)	NDF
26 TC 30	NUMBER OF NODES PER ELEMENT (MAXIMUM)	NEN
31 TC 35	NOT USED	



## A.2 DATA INPUT: MACRO CONTROL STATEMENTS

..... INPUT MACRO CONTROL CARDS - FORMAT(A4)  
THE INPUT OF EACH DATA SEGMENT IS CONTROLLED BY THE  
VALUE ASSIGNED TO CC. THE FOLLOWING VALUES ARE ADMIS-  
SIBLE AND EACH CC CARD MUST BE IMMEDIATELY FOLLOWED  
BY THE APPROPRIATE DATA.

CC VALLE	DATA TO BE INPUT
COOR	COORDINATE DATA
PCLA	CONVERT POLAR TO CARTESIAN
	COORDINATES
SPLE	CONVERT SPHERICAL TO CARTESIAN
	COORDINATES
ELEM	ELEMENT DATA
MATE	MATERIAL DATA
BOUN	NODAL BOUNDARY CONDITION DATA
FORC	PRESCRIBED NODAL DISPLACEMENT/FORCE
	DATA. FOR HEAT TRANSFER PROBLEMS
	THIS IS PRESCRIBED NODAL TEMPERA-
	TURE/FLUX DATA
TEMP	TEMPERATURE DATA
PRIN	PRINT SUBSEQUENT MESH DATA
	(DEFAULT MODE)
NOPR	DO NOT PRINT SUBSEQUENT MESH DATA
PAGE	PRINTED OUTPUT CONTROL
ENC	MUST BE LAST CARD IN MESH DATA,
	TERMINATES MESH INPUT

EXCEPT FOR THE ENC CARD THE DATA SEGMENTS CAN BE  
IN ANY ORDER. IF THE VALUES OF BOUN, FORC, OR TEMP  
ARE ZERO, NO INPUT DATA IS REQUIRED.

..... COORDINATE DATA - FORMAT(2I5,7F10.0)  
MUST IMMEDIATELY FOLLOW A COOR MACRO CARD.  
THE COORDINATE DATA CARD CONTAINS THE NODE NUMBER N  
AND THE VALUE OF THE COORDINATES FOR THE NCCE. ONLY  
THE VALUES OF (XL(I), I=1, NCM) ARE USED, WHERE NCM IS  
THE VALUE INPUT ON THE CONTROL CARD.  
NODAL COORDINATES CAN BE GENERATED ALONG A STRAIGHT  
LINE DESCRIBED BY THE VALUES ON TWO SUCCESSIVE  
CARDS. THE VALUE OF THE NCCE NUMBER IS COMPUTED  
USING THE N AND NG ON THE FIRST CARD TO COMPLETE  
THE SEQUENCE N, N+NG, N+2\*NG, ETC. NG MAY BE INPUT  
AS A NEGATIVE NUMBER, IF IT HAS INCORRECT SIGN THE  
SIGN WILL BE CHANGED. NODES NEED NOT BE IN ORDER.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	NCCE NUMBER	N
6 TC 10	GENERATOR INCREMENT	NG
11 TC 20	X1 COORDINATE	XL(1)=X(1,N)
21 TC 30	X2 COORDINATE	XL(2)=X(2,N)
31 TC 40	X3 COORDINATE	XL(3)=X(3,N)

TERMINATE WITH BLANK CARD(S)

..... POLAR TO CARTESIAN CONVERSION DATA - FORMAT(3I5,2F10.0)  
MUST IMMEDIATELY FOLLOW A POLA MACRO CARD.  
THE POLAR COORDINATES (R, THETA) MUST BE PREVIOUSLY IN-  
PUT AS COORDINATE DATA (MACRO CARD COOR) WHERE R IS X1  
AND THETA IS X2. THE NODAL POLAR COORDINATES OF NODES  
N1, N1+INC, N1+2\*INC, ..., NE ARE CONVERTED TO  
CARTESIAN COORDINATES.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	FIRST NODE TO BE	N1
	CONVERTED	
6 TC 10	LAST NODE TO BE	NE



		CONVERTED	
11 TC	15	NODAL INCREMENT	INC
16 TC	25	X COORDINATE OF THE	XO
		ORIGIN	
26 TC	35	Y COORDINATE OF THE	YO
		ORIGIN	

TERMINATE WITH BLANK CARD(S)

..... SPHERICAL TO CART. CONVERSION DATA-FORMAT(3I5,3F10.0)  
 MUST IMMEDIATELY FOLLOW A SPHE MACRO CARD.  
 THE SPHERICAL COORDINATES (R,THETA,PHI) MUST BE PRE-  
 VICIOUSLY INPUT AS COORDINATE DATA (X1,X2,X3) FOLLOWING  
 MACRO CARD CCCR. THETA IS THE ANGLE FORMED BY R AND X  
 AND PHI IS THE ANGLE FORMED BY R AND Z.  
 THE NODAL SPHERICAL COORDINATES OF NODES NI, NI+INC,  
 NI+2\*INC,.....,NE ARE CONVERTED TO CARTESIAN COORDINATES

COLUMNS	DESCRIPTION	VARIABLE	
1 TO 5	FIRST NODE TO BE	NI	
6 TO 10	LAST NODE TO BE	NE	
	CONVERTED		
11 TC	15	NODAL INCREMENT	INC
16 TC	20	X COORDINATE OF THE	XO
		ORIGIN	
26 TC	35	Y COORDINATE OF THE	YO
		ORIGIN	
36 TC	45	Z COORDINATE OF THE	ZO
		ORIGIN	

TERMINATE WITH BLANK CARD(S).

..... ELEMENT DATA - FORMAT(16I5)  
 MUST IMMEDIATELY FOLLOW AN ELEM MACRO CARD.  
 THE ELEMENT DATA CARD CONTAINS THE ELEMENT NUMBER,  
 MATERIAL SET NUMBER (WHICH ALSO SELECTS THE ELEMENT  
 TYPE, SEE MATERIAL PROPERTY DATA), AND THE SEQUENCE  
 OF NODES CONNECTED TO THE ELEMENT. IF THERE ARE LESS  
 THAN NEN NODES EITHER LEAVE THE APPROPRIATE FIELDS  
 BLANK OR PUNCH ZEROS.  
 ELEMENTS MUST BE IN ORDER. IF ELEMENT CARDS ARE  
 OMITTED THE ELEMENT DATA WILL BE GENERATED FROM  
 THE PREVIOUS ELEMENT WITH THE SAME MATERIAL NUMBER  
 AND THE NODES ALL INCREMENTED BY LX ON THE PREVIOUS  
 ELEMENT. GENERATION TO THE MAXIMUM ELEMENT NUMBER  
 OCCURS WHEN A BLANK CARD IS ENCOUNTERED.

COLUMNS	DESCRIPTION	VARIABLE
1 TO 5	ELEMENT NUMBER	L
6 TO 10	MATERIAL SET NUMBER	IX(NEN1,L)
11 TO 15	NODE 1 NUMBER	IX(1,L)
16 TO 20	NODE 2 NUMBER	IX(2,L)
ETC.		
ETC.	NODE NEN NUMBER	IX(NEN,L)
ETC.	GENERATION INCREMENT	LX

TERMINATE WITH BLANK CARD(S).

..... MATERIAL PROPERTY DATA - FORMAT(15,4X,11,17A4)  
 MUST IMMEDIATELY FOLLOW A MATE MACRO CARD.

COLUMNS	DESCRIPTION	VARIABLE
1 TO 5	MATERIAL SET NUMBER	MA
6 TO 10	ELEMENT TYPE NUMBER	IEL
11 TO 78	ALPHANUMERIC INFORMATION	XHED
	TO BE CULPT	

EACH MATERIAL CARD MUST BE FOLLOWED IMMEDIATELY



BY THE MATERIAL PROPERTY DATA REQUIRED FOR THE  
ELEMENT TYPE IEL BEING USED, E.G., SEE SECTION A.4  
AND A.5 FOR HEAT TRANSFER ELEMENTS.

..... BOUNDARY RESTRAINT DATA - FORMAT(16I5)  
MUST IMMEDIATELY FOLLOW A BOUN MACRO CARD.  
FOR EACH NODE WHICH HAS AT LEAST ONE DEGREE OF FREEDOM  
WITH A SPECIFIED DISPLACEMENT, A BOUNDARY CONDITION  
CARD MUST BE INPUT. THE CONVENTION USED FOR BOUNDARY  
RESTRAINTS IS

.E.C.0 NO RESTRAINT, FORCE SPECIFIED  
.N.E.C RESTRAINED, DISPLACEMENT SPECIFIED  
VALUES OF FORCE OR DISPLACEMENT INPUT IN FORCE

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	NODE NUMBER	N
6 TC 10	GENERATION INCREMENT	NX
11 TO 15	DOF 1 BOUNDARY CODE	IDL(1)=ID(1,N)
16 TC 20	DOF 2 BOUNDARY CODE	IDL(2)=ID(2,N)
ETC.		
ETC.	DOF NDF BOUNDARY CODE	IDL(NDF)= ID(NDF,N)

WHEN GENERATING BOUNDARY CONDITION CODES FOR  
SUBSEQUENT NODES IDL IS SET TO ZERO IF IT WAS  
INPUT .E.C.0, AND IS SET TO -1 IF INPUT NEGA-  
TIVE. ALL DEGREES OF FREEDOM WITH NON-ZERO  
CODES ARE ASSUMED FIXED.

TERMINATE WITH ELANK CARD(S).

..... NODAL FORCED BOUNDARY VALUE DATA - FORMAT(2I5,7F10.0)  
MUST IMMEDIATELY FOLLOW A FORC MACRO CARD.  
FOR EACH NODE WHICH HAS A NON-ZERO NODAL FORCE OR  
DISPLACEMENT A FORCE CARD MUST BE INPUT OR GENERATED.  
GENERATION IS THE SAME AS FOR CO-ORDINATE DATA. THE  
VALUE SPECIFIED IS A FORCE IF THE CORRESPONDING RES-  
TRAINT CODE IS ZERO AND A DISPLACEMENT IF THE CORRES-  
PONDING RESTRAINT CODE IS NON-ZERO.

COLUMNS	DESCRIPTION	VARIABLE
1 TO 5	NODE NUMBER	N
6 TC 10	GENERATION INCREMENT	NG
11 TO 20	DOF 1 FORCE (DISP.)	XL(1)=F(1,N)
21 TO 30	DOF 2 FORCE (DISP.)	XL(2)=F(2,N)
ETC.		
ETC.	DOF NDF FORCE (DISP.)	XL(NDF)= F(NDF,N)

TERMINATE WITH A ELANK CARD.

..... TEMPERATURE DATA CARD - FORMAT(2I5,F10.0)  
MUST IMMEDIATELY FOLLOW A TEMP MACRO CARD.  
NOT USED IN HEAT TRANSFER PROBLEMS.  
FOR EACH NODE WHICH HAS A NON-ZERO TEMPERATURE THE  
VALUE MUST BE INPUT. GENERATION OF VALUES CAN BE  
PERFORMED AS DESCRIBED FOR CO-ORDINATES.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	NODE NUMBER	N
6 TC 10	GENERATION INCREMENT	NG
11 TC 20	NODAL TEMPERATURE	XL(1)=T(N)

TERMINATE WITH ELANK CARD(S).

..... PAGE CONTROL DATA - FORMAT(A1)  
MUST IMMEDIATELY FOLLOW A PAGE MACRO CARD  
THE VALUE IN COLUMN 1 CONTROLS THE PRINTED OUTPUT



C  
C  
C  
C

ACCORDING TO THE FOLLOWING CONVENTION:  
1 TITLE IS WRITTEN AT THE TOP OF A NEW PAGE  
0 THE PRINTED OUTPUT IS CONTINUOUS WITHOUT  
SKIIPPING PAGES (DEFAULT MODE).



# A.3 PROBLEM SOLUTION: MACRO PROGRAMMING CCNANDS

.... MACRO PROGRAMMING COMMANDS - FORMAT(2(A4,1X),F10.0)  
 FOLLOWING IS A LIST OF MACRO INSTRUCTIONS WHICH  
 CAN BE USED TO CONSTRUCT SOLUTION ALGORITHMS. THE  
 FIRST INSTRUCTION MUST BE A CARD WITH MACR IN  
 COLUMNS 1 TO 4. THE INDICATED ISW VALUE IS USED BY  
 EACH ELEMENT ROUTINE TO PERFORM THE APPROPRIATE  
 COMPUTATIONS.

COLUMNS 1 TO 4	COLUMNS 6 TO 9	COLUMNS 11 TO 20	DESCRIPTION
CHEC CMAS			PERFORM CHECK OF MESH (ISW=2) CONSISTENT MASS FORMULATION (ISW=5)
CCNV DATA			DISPLACEMENT CONVERGENCE TEST READ DATA **MACRO COMMANDS. A **MACRO CARD IS INSERTED AS DATA FOLLOWING THE MACRO PROGRAM
DISP		N	PRINT NODAL DISPLACEMENTS EVERY N STEPS IN LOOP
**CT EIGE		V	SET TIME INCREMENT TO VALUE V COMPUTE DOMINANT EIGENVALUE AND VECTOR OF CURRENT MASS AND SYMMETRIC STIFFNESS
EXCD			EXPLICIT CENTERED DIFFERENCE INTEGRATION OF EQUATIONS OF MOTION USING LUMPED MASS. FIRST CALL RESERVES MEMORY ONLY.
FORM			FORM RIGHT HAND SIDE OF EQUA- TIONS (ISW=6)
LMAS LCCF		N	LUMPED MASS FORMULATION (ISW=5) LOOP N TIMES ALL INSTRUCTIONS BETWEEN MATCHING NEXT INSTRUCTION.
MESH			INPUT MESH CHANGES (MUST NOT CHANGE BOUNDARY RESTRAINTS). DATA FOLLOWS MACRO PROGRAM. SEE SECTION A.2.
NEXT OCE1	INIT		END OF LOOP INSTRUCTION INPUT INITIAL CONDITIONS AND CONSTANTS OF INTEGRATION FOR FIRST ORDER ORDINARY DIFFEREN- TIAL EQUATION SOLUTION (DATA FOLLOWS MACRO PROGRAM)
OCE1 OCE1 PRCP	LINE CUAL	1	TWO-POINT INTEGRATION SCHEME THREE-POINT INTEGRATION SCHEME INPUT PROPORTIONAL LOAD TABLE (DATA FOLLOWS MACRO PROGRAM)
REAC SOLV			COMPUTE NODAL REACTIONS (ISW=6) SOLVE TANGENT EQUATIONS. UPDATE NODAL DISPLACEMENTS.
STRE		N	PRINT ELEMENT VARIABLES (E.G., STRESSES) EVERY N STEPS IN LOOP (ISW=4)
TANG			SYMMETRIC TANGENT STIFFNESS FORMULATION (ISW=3)
TIME *TIM			ADVANCE TIME BY CT VALUE INDICATE EXECUTION TIME FOR EVERY MACRO INSTRUCTION
**TOL		V	SET SOLUTION CONVERGENCE TOLE- RANCE TO VALUE V
UTAN			UNSYMMETRIC TANGENT STIFFNESS FORMULATION (ISW=3)
END			END OF MACRO PROGRAM COMMANDS. DATA FOR PROGRAM FOLLOWS IN ORDER OF USE

..... FIRST ORDER C.D.E. SOLVER DATA



MUST FOLLOW THE MACRO CARD END AND CORRESPONDS TO THE MACRO INSTRUCTION CCE1 INIT. THE FIRST CARD INPUTS THE CONTROL INFORMATION AND THE NODAL INITIAL DISPLACEMENTS ARE INPUT ON THE FOLLOWING CARDS. NO AUTOMATIC TIME STEP ADJUSTMENT IS PERFORMED IF DUMAX OR DUMIN ARE LEFT BLANK OR ZERO.

..... CONTROL CARD - FORMAT(4F10.0)

COLUMNS	DESCRIPTION	VARIABLE
1 TC 10	INTEGRATION PARAMETER THETA FOR TWO POINT SCHEME (DEFAULT VALUE THETA=2/3)	C5
11 TC 20	INTEGRATION PARAMETER GAMMA FOR THREE POINT SCHEME (DEFAULT GAMMA=1.5)	C1
21 TC 30	INTEGRATION PARAMETER BETA FOR THREE POINT SCHEME (DEFAULT BETA=.8)	C2
31 TC 40	MAXIMUM DISPLACEMENT ALLOWED	DUMAX
41 TC 50	MINIMUM DISPLACEMENT ALLOWED	DUMIN

..... INITIAL CONDITION DATA - FORMAT(2I5,7F10.0)  
FOR EACH NCDE WHICH HAS A NON-ZERO INITIAL DISPLACEMENT A CARD MUST BE INPUT OR GENERATED. THE GENERATION IS THE SAME AS FOR COORDINATE DATA.

COLUMNS	DESCRIPTION	VARIABLE
1 TO 5	NCDE NUMBER	N
6 TO 10	GENERATION INCREMENT	NG
11 TO 10	CCF 1 DISPLACEMENT	XL(1)=U(1+N-1)
ETC.	DOF NDF DISPLACEMENT	XL(NCF)=

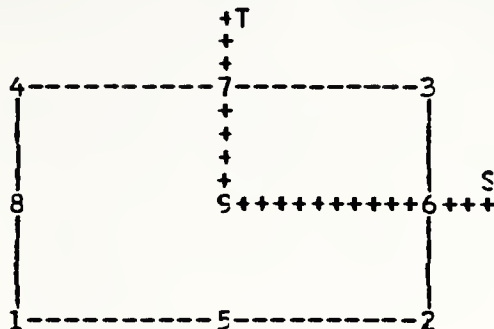
TERMINATE WITH BLANK CARD(S).

..... PROPORTIONAL LOAD CARD - FORMAT(2I5,6F10.0)  
A SIMPLIFIED PROPORTIONAL LOADING IS PERMITTED WITH  
 $PROP = A1 + A2*TIME + A3*(SIN(A4*TIME + A5))*L$   
WHERE THE COEFFICIENTS ARE INPUT ON A DATA CARD  
FOLLOWING THE END MACRO CARD ACCORDING TO THE  
FOLLOWING TABLE.

COLUMN	DESCRIPTION
6 TC 10	L
11 TC 20	MINIMUM TIME FOR WHICH PROCP IS COMPUTED
21 TC 30	MAXIMUM TIME FOR WHICH PROCP IS COMPUTED
31 TC 40	A1
41 TC 50	A2
51 TC 60	A3
61 TC 70	A4
71 TC 80	A5



#### A.4 TWO DIMENSIONAL HEAT TRANSFER ELEMENT DATA



MUST FOLLOW THE MATE MACRO CARD.  
THE TWO DIMENSIONAL HEAT TRANSFER ELEMENT IS CALLED  
ELMT02 AND THUS THE ELEMENT TYPE NUMBER IN COLUMN 10  
OF EACH MATERIAL SET NUMBER CARD MUST BE 2 WHEN THIS  
ELEMENT IS REQUESTED. THE SECOND CARD GIVES GENERAL  
INFORMATION AND IS PREPARED AS FOLLOWS:

#### .... GENERAL INFORMATION DATA - FORMAT(5F10.0,4I5)

COLUMNS	DESCRIPTION	VARIABLE
1 TC 1C	CONDUCTIVITY IN X DIR. (IGNORED IF TEMPERATURE DEPENDENT)	D(1)
11 TC 2C	CONDUCTIVITY IN Y DIR. (IGNORED IF TEMPERATURE DEPENDENT)	D(2)
21 TC 3C	SPECIFIC HEAT (IGNORED IF TEMPERATURE DEPEND- ENT)	D(3)
31 TC 4C	DENSITY (IGNORED IF TEMPERATURE DEPENDENT)	D(4)
41 TC 5C	HEAT GENERATION PER UNIT VOLUME (IGNORED IF TEMPERATURE DEPEND- ENT)	D(5)
51 TC 55	NUMBER OF INTEGRATION POINTS PER DIRECTION (DEFAULT 4)	NGP
56 TC 6C	GEOMETRY TYPE .EQ.2 FOR AXISYMMETRY .NE.2 FOR PLANE GEOMETRY	KAT
61 TC 65	TOTAL NUMBER OF LINES WITH SPECIFIED BOUND- ARY CONDITIONS IN ELEMENTS WITH THE SAME MATERIAL SET NUMBER	NLBC
66 TC 7C	CODE TO INDICATE IF ANY OF THE MATERIAL PROPER- TIES IS TEMPERATURE DE- PENDENT. .EQ.0 IF ALL MATERIAL PROPERTIES ARE CONSTANT .NE.0 IF ANY OF THE MATE- RIAL PROPERTIES IS TEM- PERATURE DEPENDENT	INOL

THE FOLLOWING DATA ARE REQUIRED IF ANY OF THE MATERIAL  
PROPERTIES IS TEMPERATURE DEPENDENT. THEIR ORDER IS  
CRUCIAL AND THOSE CARDS WHICH CORRESPOND TO CONSTANT  
MATERIAL PROPERTIES MUST BE OMITTED.



C.... MATERIAL PROPERTY CODE -FORMAT(4I5)  
 THIS CARD IS ALWAYS REQUIRED IF INOL.NE.0 AND MUST  
 IMMEDIATELY FOLLOW THE GENERAL INFORMATION DATA CARD.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	.EQ.0 CCNSTANT CONDUCTI- VITY IN X DIR.	IKX
6 TC 10	.NE.0 TEMP. CEP. KX .EQ.0 CCNSTANT CONDUCTI- VITY IN Y DIR.	IKY
11 TC 15	.NE.0 TEMP. CEP. KY .EQ.0 CCNSTANT HEAT CA PACITY (SPECIFIC HEAT * DENSITY)	IRCC
16 TC 20	.NE.0 TEMP. CEP. HEAT CAPACITY .EQ.0 CCNSTANT HEAT GE- NERATION PER UNIT VOL.	IQ
	.NE.0 TEMP. CEP. Q	

C.... CONDUCTIVITY IN X DIRECTION TABLE  
 THE TEMPERATURE DEPENDENT CONDUCTIVITY IN THE X  
 DIRECTION (KX) IS CALCULATED BY LINEAR INTERPOLATION  
 BETWEEN TWO CONSECUTIVE ENTRIES IN THIS TABLE. THE  
 FIRST CARD OF THIS SET OF DATA INDICATES THE NUMBER  
 OF ENTRIES IN THE TABLE. IT MUST BE FOLLOWED BY THE  
 SAME NUMBER OF CARDS WITH A PAIR OF VALUES (TEMPE-  
 RATURE AND CORRESPONDENT PROPERTY) IN EACH ONE. THEIR  
 ORDER IS CRUCIAL. THE PAIRS MUST BE ORDERED ACCORDING  
 TO THE INCREASING VALUE OF TEMPERATURE.  
 OMIT IF IKX.EC.0

FIRST CARD -FORMAT(I5)

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	NUMBER OF ENTRIES OF THE TABLE TO BE INPUT	N

TABLE DATA - FORMAT(2F10.0)  
 FOR EACH PAIR OF VALUES A CARD MUST BE INPUT. THE  
 TOTAL IS THE NUMBER SPECIFIED IN THE PRECEDING CARD.  
 LOWEST TEMPERATURE IN THE FIRST CARD, SECOND LOWEST  
 IN THE SECOND CARD, ETC.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 10	TEMPERATURE	XX(I)
11 TC 20	CONDUCTIVITY	YY(I)

C.... CONDUCTIVITY IN Y DIRECTION TABLE  
 PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE  
 X DIRECTION TABLE.  
 OMIT IF IKY.EC.0

C.... HEAT CAPACITY TABLE  
 PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE  
 X DIRECTION TABLE.  
 OMIT IF IRCC.EC.0

C.... HEAT GENERATION PER UNIT VOLUME TABLE  
 PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE  
 X DIRECTION TABLE.  
 OMIT IF IQ.EC.0

C.... LINE BOUNDARY CONDITION DATA - FORMAT(3I5,2F10.0)  
 OMIT IF NLEC.EC.0.  
 A CARD MUST INPUT FOR EACH LINE BOUNDARY CONDITION.  
 IF THE SAME LINE IS SUBJECTED TO MORE THAN ONE TYPE

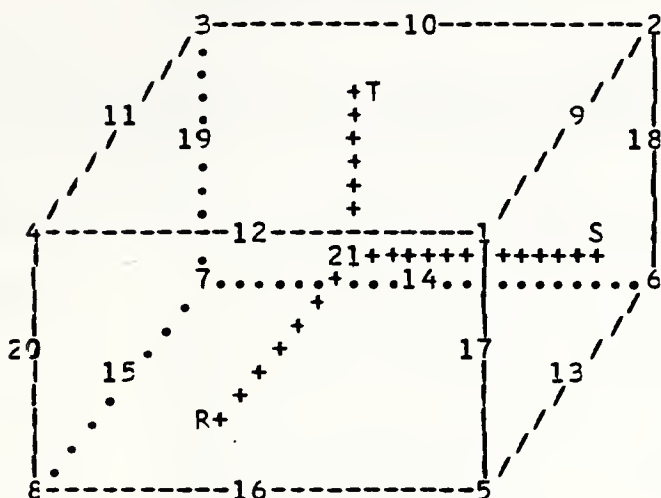


CF BOUNDARY CONDITION, A CARD MUST BE USED FOR EACH  
 ONE OF THESE TYPES.  
 THE TOTAL NUMBER OF CARDS MUST BE NLBC.  
 THE PROPERTY VALUE IS DEFINED:  
 FOR CONVECTION - CONSTANT HEAT TRANSFER COEFFICIENT  
 (IGNORED IF KBCOND(I).EQ.4)  
 FOR FLUX - FLUX PER UNIT AREA  
 FOR RADIATION - PRODUCT OF EMISSIVITY BY STEFAN-  
 BOLTZMAN CONSTANT  
 THE AMBIENT TEMPERATURE IS IGNORED FOR FLUX BOUNDARY  
 CONDITION

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	ELEMENT NUMBER	KEL(I)
6 TC 10	CODE TO INDICATE BOUN- DARY CONDITION TYPE	KBCOND(I)
	.EQ.1 FLUX	
	.EQ.2 CONVECTION (CONS- TANT COEFFICIENT)	
	.EQ.3 RADIATION	
	.EQ.4 CONVECTION (TEMP. DEP. COEFF.)	
11 TC 15	CODE TO INDICATE LINE	KLINE(I)
	.EQ.1 S=+1 LINE	
	.EQ.-1 S=-1 LINE	
	.EQ.2 T=+1 LINE	
	.EQ.-2 T=-1 LINE	
16 TC 25	PROPERTY VALUE	PRCPB(I,1)
26 TC 35	AMBIENT TEMPERATURE	PROPB(I,2)



# A.5 THREE DIMENSIONAL HEAT TRANSFER ELEMENT DATA



MUST FOLLOW THE MATE MACRO CARD.  
THE THREE DIMENSIONAL HEAT TRANSFER ELEMENT IS CALLED  
ELMT03 AND THUS THE ELEMENT TYPE NUMBER IN COLUMN 10  
OF EACH MATERIAL SET NUMBER CARD MUST BE 3 WHEN THIS  
ELEMENT IS REQUESTED. THE SECOND CARD GIVES GENERAL  
INFORMATION AND IS PREPARED AS FOLLOWS:

## ..... GENERAL INFORMATION DATA - FORMAT(6F10.0,4I5)

COLUMNS	DESCRIPTION	VARIABLE
1 TC 1C	CONDUCTIVITY IN X DIR. (IGNORED IF TEMPERATURE DEPENDENT)	D(1)
11 TC 2C	CONDUCTIVITY IN Y DIR. (IGNORED IF TEMPERATURE DEPENDENT)	D(2)
21 TC 3C	CONDUCTIVITY IN Z DIR. (IGNORED IF TEMPERATURE DEPENDENT)	D(3)
31 TC 4C	SPECIFIC HEAT (IGNORED IF TEMPERATURE DEPENDENT)	D(4)
41 TC 5C	DENSITY (IGNORED IF TEMPERATURE DEPENDENT)	D(5)
51 TC 6C	HEAT GENERATION PER UNIT VOLUME (IGNORED IF TEMPERATURE DEPENDENT)	D(6)
61 TC 6C	NUMBER OF INTEGRATION POINTS PER DIRECTION (DEFAULT 4)	NGP
66 TC 7C	GEOMETRY TYPE •EQ.2 FOR AXISYMMETRY •NE.2 FOR PLANE GEOMETRY	KAT
71 TC 7C	TOTAL NUMBER OF SURFACES WITH SPECIFIED BOUNDARY CONDITIONS IN ELEMENTS WITH THE SAME MATERIAL SET NUMBER	NSBC
76 TC 8C	CODE TO INDICATE IF ANY OF THE MATERIAL PROPERTIES IS TEMPERATURE DEPENDENT. •EQ.0 IF ALL MATERIAL PROPERTIES ARE CONSTANT	INCL



.NE.0 IF ANY OF THE MATERIAL PROPERTIES IS TEMPERATURE DEPENDENT

THE FOLLOWING DATA ARE REQUIRED IF ANY OF THE MATERIAL PROPERTIES IS TEMPERATURE DEPENDENT. THEIR ORDER IS CRUCIAL AND THOSE CARDS WHICH CORRESPOND TO CONSTANT MATERIAL PROPERTIES MUST BE OMITTED.

.... MATERIAL PROPERTY CODE -FORMAT(4I5)  
THIS CARD IS ALWAYS REQUIRED IF INOL.NE.0 AND MUST IMMEDIATELY FOLLOW THE GENERAL INFORMATION DATA CARD.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	.EQ.0 CONSTANT CONDUCTIVITY IN X DIR.	IKX
6 TC 10	.NE.0 TEMP. DEP. KX .EQ.0 CONSTANT CONDUCTIVITY IN Y DIR.	IKY
11 TC 15	.NE.0 TEMP. DEP. KY .EQ.0 CONSTANT CONDUCTIVITY IN Z DIR.	IKZ
16 TC 20	.NE.0 TEMP. DEP. KZ .EQ.0 CONSTANT HEAT CAPACITY (SPECIFIC HEAT * DENSITY)	IRQC
21 TC 25	.NE.0 TEMP. DEP. HEAT CAPACITY .EQ.0 CONSTANT HEAT GENERATION PER UNIT VOL.	IQ
	.NE.0 TEMP. DEP. Q	

.... CONDUCTIVITY IN X DIRECTION TABLE  
THE TEMPERATURE DEPENDENT CONDUCTIVITY IN THE X DIRECTION (KX) IS CALCULATED BY LINEAR INTERPOLATION BETWEEN TWO CONSECUTIVE ENTRIES IN THIS TABLE. THE FIRST CARD OF THIS SET OF DATA INDICATES THE NUMBER OF ENTRIES IN THE TABLE. IT MUST BE FOLLOWED BY THE SAME NUMBER OF CARDS WITH A PAIR OF VALUES (TEMPERATURE AND CORRESPONDENT PROPERTY) IN EACH ONE. THEIR ORDER IS CRUCIAL. THE PAIRS MUST BE ORDERED ACCORDING TO THE INCREASING VALUE OF TEMPERATURE.  
CMIT IF IKX.EQ.0

FIRST CARD -FORMAT(I5)

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	NUMBER OF ENTRIES OF THE TABLE TO BE INPUT	N

TABLE DATA - FORMAT(2F10.0)  
FOR EACH PAIR OF VALUES A CARD MUST BE INPUT. THE TOTAL IS THE NUMBER SPECIFIED IN THE PRECEDING CARD. LOWEST TEMPERATURE IN THE FIRST CARD, SECOND LOWEST IN THE SECOND CARD, ETC.

COLUMNS	DESCRIPTION	VARIABLE
1 TC 10	TEMPERATURE	XX(I)
11 TC 20	CONDUCTIVITY	YY(I)

.... CONDUCTIVITY IN Y DIRECTION TABLE  
PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE X DIRECTION TABLE.  
CMIT IF IKY.EQ.0

.... CONDUCTIVITY IN Z DIRECTION TABLE  
PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE X DIRECTION TABLE.



CMIT IF IKZ.EC.0

.... HEAT CAPACITY TABLE  
PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE  
X DIRECTION TABLE.  
CMIT IF IRCC.EC.0

.... HEAT GENERATION PER UNIT VOLUME TABLE  
PREPARED IN THE SAME WAY AS THE CONDUCTIVITY IN THE  
X DIRECTION TABLE.  
CMIT IF IC.EQ.0

.... SURFACE BOUNDARY CONDITION DATA - FORMAT(3I5,2F10.0)  
CMIT IF ISEC.EC.0.  
A CARD MUST INPUT FOR EACH SURFACE BOUNDARY CONDITION.  
IF THE SAME SURFACE IS SUBJECTED TO MORE THAN ONE TYPE  
OF BOUNDARY CONDITION, A CARD MUST BE USED FOR EACH  
ONE OF THESE TYPES.  
THE TOTAL NUMBER OF CARDS MUST BE NSBC.  
THE PROPERTY VALUE IS DEFINED:  
FOR CONVECTION - CONSTANT HEAT TRANSFER COEFFICIENT  
(IGNORED IF KBCCNC(I).EQ.4)  
FOR FLUX - FLUX PER UNIT AREA  
FOR RADIATION - PRODUCT OF EMISSIVITY BY STEFAN-  
BOLTZMAN CONSTANT  
THE AMBIENT TEMPERATURE IS IGNORED FOR FLUX BOUNDARY  
CONDITION

COLUMNS	DESCRIPTION	VARIABLE
1 TC 5	ELEMENT NUMBER	KEL(I)
6 TC 10	CODE TC INDICATE BOUN- DARY CONDITION TYPE	KBCCNC(I)
	.EQ.1 FLUX	
	.EQ.2 CONVECTION (CONS- TANT COEFFICIENT)	
	.EQ.3 RADIATION	
	.EQ.4 CONVECTION (TEMP. DEP. COEFF.)	
11 TC 15	CODE TC INDICATE SURF.	KSURF(I)
	.EQ.1 R=+1 SURF	
	.EQ.-1 R=-1 SURF	
	.EQ.2 S=+1 SURF	
	.EQ.-2 S=-1 SURF	
	.EQ.3 T=+1 SURF	
	.EQ.-3 T=-1 SURF	
16 TC 25	PROPERTY VALUE	PROPB(I,1)
26 TC 35	AMBIENT TEMPERATURE	PRCPB(I,2)



SAMPLE EXAMPLE

B.1 DATA CARDS

FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

33 6 1 2 1 8

COORD

1 5 .1

31 .3

4 5.116666667

29 .283333333

2 5 .1 3.

32 .3 3.

3 5 .1 6.

33 .3 6.

5 5.116666667 6.

30 .283333333 6.

POLA

1 33 1

ELEM

1 1 1 1 6 8 3 4 7 5 2 5

BOUND

1 1 -1

3 -1

FORC

1 1 820.

3 820.



MATE	1	2			
10.	6	2	10.	20.	1
END			1		
MACR			200.		
TANG					
FORM					
SOLV					
DISP					
END					
STOP					



FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

NUMBER OF NODAL POINTS = 33  
NUMBER OF ELEMENTS = 6  
NUMBER OF MATERIAL SETS = 1  
DIMENSION OF COORDINATE SPACE = 2  
DEGREES OF FREEDOM/NODE = 1  
NODES PER ELEMENT (MAXIMUM) = 8  
EXTRA D.O.F. TO ELEMENT = 0

FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

NODAL COORDINATES

NODE	1 COORD	2 COORD
1	0.1000	0.0
2	0.1000	3.0000
3	0.1000	6.0000
4	0.1167	0.0
5	0.1167	6.0000
6	0.1333	0.0
7	0.1333	3.0000
8	0.1333	6.0000
9	0.1500	0.0
10	0.1500	6.0000
11	0.1667	0.0
12	0.1667	3.0000
13	0.1667	6.0000
14	0.1833	0.0
15	0.1833	6.0000
16	0.2000	0.0
17	0.2000	3.0000



18	0.2000	6.0000
19	0.2167	0.0
20	0.2167	6.0000
21	0.2333	0.0
22	0.2333	3.0000
23	0.2333	6.0000
24	0.2500	0.0
25	0.2500	6.0000
26	0.2667	0.0
27	0.2667	3.0000
28	0.2667	6.0000
29	0.2833	0.0
30	0.2833	6.0000
31	0.3000	0.0
32	0.3000	3.0000
33	0.3000	6.0000

FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

CARTESIAN COORDINATES COMPUTED FROM POLAR INPUT WITH XO = 0.0 YO = 0.0

NODE	1-COORD	2-COORD
1	0.1000	0.0
2	0.0999	0.0052
3	0.0995	0.0105
4	0.1167	0.0
5	0.1160	0.0122
6	0.1333	0.0
7	0.1332	0.0070
8	0.1326	0.0139
9	0.1500	0.0
10	0.1492	0.0157
11	0.1667	0.0
12	0.1664	0.0087
13	0.1658	0.0174
14	0.1833	0.0



15	0.1823	0.0192
16	0.2000	0.0
17	0.1997	0.0105
18	0.1989	0.0209
19	0.2167	0.0
20	0.2155	0.0226
21	0.2333	0.0
22	0.2330	0.0122
23	0.2321	0.0244
24	0.2500	0.0
25	0.2486	0.0261
26	0.2667	0.0
27	0.2663	0.0140
28	0.2652	0.0279
29	0.2833	0.0
30	0.2818	0.0296
31	0.3000	0.0
32	0.2996	0.0157
33	0.2984	0.0314

# FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

## ELEMENTS

ELEMENT	MATERIAL	1 NODE	2 NODE	3 NODE	4 NODE	5 NODE	6 NODE	7 NODE	8 NODE
1	1	1	6	8	3	4	7	5	2
2	1	6	11	13	8	9	12	10	7
3	1	11	16	18	13	14	17	15	12
4	1	16	21	23	18	19	22	20	17
5	1	21	26	28	23	24	27	25	22
6	1	26	31	33	28	29	32	30	27

# FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

NODAL B.C.



NODE 1 B.C.  
 1 -1  
 2 -1  
 3 -1

# FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

## NODAL FORCE/DISPL

NODE 1 FORCE  
 1 820.0000  
 2 820.0000  
 3 820.0000

# FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

## MATERIAL PROPERTIES

MATERIAL SET 1 FOR ELEMENT TYPE 2

DEGREE OF FREEDOM ASSIGNMENTS	LOCAL NUMBER	GLOBAL NUMBER
	1	1

## HEAT CONDUCTION ELEMENT

CONDUCTIVITY KX = 10.00000 KY = 10.00000  
 SPECIFIC HEAT 0.0  
 DENSITY 0.0  
 HEAT GENER/UNIT VOL 0.0

3 GAUSS PTS/DIR  
 1 LINES WITH SPECIFIED BOUNDARY CONDITIONS  
 PLANE ANALYSIS



FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

LINE B.C.

ELEM	B.C.	LINE	PROPERTY VALUE	TEMPERATURE
6	2	1	200.0000	20.00000

FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

MACRO INSTRUCTIONS

MACRO STATEMENT	VARIABLE 1	VARIABLE 2
-----------------	------------	------------

TANG	0.0	0.0
FORM	0.0	0.0
SOLV	0.0	0.0
DISP	0.0	0.0
END	0.0	0.0

**MACRO INSTRUCTION	1	START EXECUTION**	TANG	V1 =	0.0	, V2 =	0.0
**MACRO INSTRUCTION	2	START EXECUTION**	FORM	V1 =	0.0	, V2 =	0.0

FORCE CONVERGENCE TEST

RNMAX =	5911.5	RN =	5911.5	TOL =	0.10000D-08		
**MACRO INSTRUCTION	3	START EXECUTION**	SOLV	V1 =	0.0	, V2 =	0.0
ENERGY (DR**DR)	=	0.4753833140D 07					
**MACRO INSTRUCTION	4	START EXECUTION**	DISP	V1 =	0.0	, V2 =	0.0

FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

PROPORTIONAL LOAD 1.0000

FEAP \* \* RADIAL TEMPERATURE DISTRIBUTION IN A HOLLOW CYLINDER

NODAL DISPLACEMENTS TIME 0.0

NODE	1	COORD	2	COORD	1	DISPL
1	0.1000	0.0	0.82000D 03			



2	0.0999	0.0052	0.820000	03
3	0.0995	0.0105	0.820000	03
4	0.1167	0.0	0.722560	03
5	0.1160	0.0122	0.722560	03
6	0.1333	0.0	0.638110	03
7	0.1332	0.0070	0.638110	03
8	0.1326	0.0139	0.638110	03
9	0.1500	0.0	0.563650	03
10	0.1492	0.0157	0.563650	03
11	0.1667	0.0	0.497020	03
12	0.1664	0.0087	0.497020	03
13	0.1658	0.0174	0.497020	03
14	0.1833	0.0	0.436770	03
15	0.1823	0.0192	0.436770	03
16	0.2000	0.0	0.381750	03
17	0.1997	0.0105	0.381750	03
18	0.1989	0.0209	0.381750	03
19	0.2167	0.0	0.331140	03
20	0.2155	0.0226	0.331140	03
21	0.2333	0.0	0.284280	03
22	0.2330	0.0122	0.284280	03
23	0.2321	0.0244	0.284280	03
24	0.2500	0.0	0.240660	03
25	0.2486	0.0261	0.240660	03
26	0.2667	0.0	0.199850	03
27	0.2663	0.0140	0.199850	03
28	0.2652	0.0279	0.199850	03
29	0.2833	0.0	0.161520	03
30	0.2818	0.0296	0.161520	03
31	0.3000	0.0	0.125380	03
32	0.2996	0.0157	0.125380	03
33	0.2984	0.0314	0.125380	03

\*\*END OF MACRO EXECUTION\*\*



## FEAP LISTING

101











```

109 GO TC 102 IF (.NOT. PRT) RETURN
      DO 113 J = 1, NUMNP
      IF (PRT2) GC TC 111
      DO 114 L = 1, NDM
      IF (X(L, J).NE.0.000) GC TC 111
      CONTINUE
110 GO TC 111
111 MCT = MCT - 1 GO TO 112
      MCT = 50
      WRITE(6, 2000) O, HEAD, (CC(L), L=1, 3), (L, CD(1), CD(2), L=1, NDM)
112 IF (PCCMP(X(1, J), BL)) WRITE(6, 2008) N
      IF (.NOT. FCOMP(X(1, J), BL)) WRITE(6, 2009) J, (X(L, J), L=1, NDM)
113 CONTINUE
      RETURN
1000 FORMAT(2I5, 7F10.0)
2000 FORMAT(A1, 20A4//5X, 5HACDAL, 3A4//6X, 4+NODE, 9(I7, A4, A2))
2008 FORMAT(I10, 32H HAS NCT BEEN INPUT OR GRNERATED)
2009 FORMAT(I10, 9F13.4)
3000 FORMAT(5I, 43H**FATAL ERROR 02** ATTEMPT TO GENERATE NODE, I5, 3H IN
      1 END, 3A4)

      SUBROUTINE GENELM(ICL, IX, NEN1, PRT, ERR)
      IMPLICIT REAL*8 (A-H, C-Z)
      LOGICAL PRT, ERR
      COMMON /CDATA/O, HEAD(20), NUMNP, NUMEL, NUMMAT, NEN, NEQ, IPR
      DIMENSION ICL(1), IX(NEN1, 1)

      C..... READ AND/OR GENERATE ELEMENT CCNNECTIONS
      C
      L = 206 I = 1, NUMEL, 5C
      DO 207 I = 1, NUMEL, 5C
      IF (PRT) WRITE(6, 2001) C, HEAD, (K, K=1, NEN)
      J = 1, INC(NUMEL, I+49)
      DO 208 N = I, J
      IF (L-N) 202, 203
      READ(5, 1001) L, LK, (ICL(K), K=1, NEN), LX
      IF (L.EQ.0) L = NUMEL+1
      IF (LX.EQ.0) LX=1
      IF (L-N) 201, 202, 203
      WRITE(6, 3001) L, N
      ERR = 1
      GO TC 206
      LX = LX
      DO 207 K = 1, NEN

```



```

207 IF(ICL(K).GT.NUMNP.CR.ICL(K).LT.0) GO TO 208
    IX(K,L) = IDL(K)
    IX(NEN1,L) = LK
    GO TO 205
203 IX(NEN1,N) = IX(NEN1,N-1)
    CO 204 K = 1,NEN
    IX(K,N) = IX(K,N-1) + NX
    IF(IX(K,N-1).EQ.0) IX(K,N) = 0
    IF(IX(K,N).GT.NUMNP.CR.ICL(K,N).LT.0) GO TO 208
    CONTINUE
204 IF(PRT) WRITE(6,2002) N,IX(NEN1,N),(IX(K,N),K=1,NEN)
205 GO TO 206
208 WRITE(6,2002) N
    ERR = .TRUE.
    CONTINUE
206 RETURN (1615)
1001 FORMAT(A1,2CA4//5X,8ELEMENTS//3X,7HELEMENT,2X,8HMATERIAL,
2001 14(I12,5F NODE)/(20X,14(I13,5H NODE)))
2002 14(I12,5F NODE)/(20X,14(I18))
3001 FORMAT(2I10,14I8//20X,14I8)
3002 FORMAT(2X,20H**ERROR 03** ELEMENT,15,22H APPEARS AFTER ELEMENT,15)
    END
    FORMAT(5X,20H**ERRRCR 04** ELEMENT,15,18F HAS ILLEGAL NODES)

SUBROUTINE GENBCC(ICL,ID,NDF,PRT,III)
IMPLICIT REAL*8 (A-F,O-Z)
LOGICAL PRT
COMMON /CCATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /LABEL/ POIS(6),A(6),BC(2),DI(6),CD(3),TE(3),FD(3)
DIMENSION ICL(6),ID(NCF,1)

C..... READ AND/OR GENERATE RESTRAINT CODES
C
IF(PRT) WRITE(6,2000) O,HEAD,(I,BC,I=1,NDF)
III = 1
N = C
NG = 0
L = N
LG = N
READ(5,1CC1) N,NG,ICL
IF(N.LE.C.OR.N.GT.NUMNP) GO TO 60
CO 51 I = 1,NDF
ID(I,N) = IDL(I)
IF(L.NE.N.AND.IDL(I).EQ.C.AND.ID(I,L).LT.0) ID(I,N) = -1
CONTINUE
LG = I SIGN(LG,N-L)
L = L + LG
51
52

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00284

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IF((N-L)*LG.LE.O) GC TC 402
CO 53 I = 1,NDF
IF(IC(I,L-LG).LT.O) IC(I,L) = -1
GO TC 52
CO 58 N = 1,NUMNP
DO 56 I = 1,NDF
IF(IC(I,N).NE.O) GO TO 57
CCNTINCE
GO TO 58
IF(PRT) WRITE(6,2001) N,(ID(I,N),I=1,NDF)
CONTINUE
RETURN
FORMAT(16I5)
1001 FORMAT(A1,2CA4//5X,17HNCCAL B.C. //6X,4HNODE,9(I7,A4,A2)/1X)
2000 FORMAT(110,5I13)
2001 ENC

SUBROUTINE PMESH(ICL,IE,C, ID,X,IX,F,T,NDF,NDM,NEN1,III)
IMPLICIT REAL*8 (A-F,C-Z)

C..... DATA INPLT ROUTINE FOR MESH DESCRIPTION
C
LCCICAL FFT,ERR,PCCMF
COMMON /CCATA/ O,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /ELCATA/ DM,N,MA,MCT,IEL,NEL
COMMON /LABEL/ PDIS(6),A(6),BC(2),DI(6),CD(3),TE(3),FD(3)
COMMON /DIMENS/ IEL(7,1),D(10,1),ID(NDF,1),X(NDM,1),IX(NEN1,1),XHED(7)
1 ICL(6),F(NCF,1),T(1),WD(12),VA(2)
1 DATA WC/4FCOOR,4HELEN,4HMATE,4HBCUN,4HFCRC,4HTEMP,4HEND ,4HPRIN,
4FNOPR,4HPAGE,4HFOA,4HSPHE/
DATA BL/4HBLAN/,VA/4F VAL,2HUE/,LIST/12/
INITIALIZE ARRAYS
ERR = .FALSE.
IF(III.LT.O) GO TO 10
PRT = .TRUE.
CO 101 N = 1,NUMNP
DO 100 I = 1,NDF
ID(I,N) = 0.0CO
X(I,N) = 0.0CO
T(N) = C.CCO
READ(5,100) CC
CO 20 I = 1,LIST
IF(PCCMF(CC,WC(I))) GC TC 30
CONTINUE
GO TC 10
30 GO TC (1,2,3,4,5,6,7,8,9,11,12,13),I

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00332

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C..... NOCAL CQCRINATE DATA INPUT
1 CALL GENVEC(NDM,X,CI,PRT,ERR,.TRUE.)
C..... GO TC 1C [ATA INPUT
2 ELEMENT GENELM(ICL,IX,NEN1,PRT,ERR)
C..... GO TO 1C
3 MATERIAL DATA INPUT
WRITE(6,20C4) O,HEAC
CO 3C4 N=1,NUMMAT
READ(5,1002) MA,IEL,ICL,XHED
IF(MA.EC.C) GO TO 1C
CO 300 I = 1,NDF
IE(I,MA) = ICL(I)
CO 301 I = 1,NDF
IF(ICL(I).NE.O) GO TC 303
300 CONT INLE RESET TO DEFAULT VALUES
301 ALL ZERO I = 1,NDF
C..... CO 302 I = 1
302 IE(I,MA) = IEL
303 WRITE(6,20C3) MA,IEL,XHED,(I,IE(I,MA),I=1,NCF)
304 CALL ELMLIB(D(1,MA),IDL,X,IX,IDL,IDL,IDL,NCF,NDM,NDF,1)
C..... GO TC 10
4 READ THE RESTRAINT CCNITIONS FOR EACH NCDE
CALL GENBCC(IDL,ID,NDF,PRT,III)
C..... GO TC 10
5 FORCE/GENVEC(NDF,F,FC,PRT,ERR,.FALSE.)
C..... GO TC 10
6 TEMPERATURE DATA INPLT
CALL GENVEC(1,T,TE,PRT,ERR,.FALSE.)
7 GO TC 1C STOP
8 IF(ERR) RETURN
9 PRT = .TRUE.
GO TC 10
PRT = .FALSE.
GO TO 1C
11 REAC(5,1000) O
C..... GO TC 1C
12 CONVERT IC POLAR COORDINATES TO CARTESIAN
CALL FCPLAR(X,NDM,PFT)
C..... GO TC 10
13 CCNVERT SPHERICAL CQCRINATES TO CARTESIAN
CALL SPFE(X,NDM,PRT)
GO TC 1C
1000 FORMAT(A4,75X,A1)
1002 FORMAT(15,4X,I1,6I2,7A4)

```







```

2001 FORMAT(I10,6F13.4)
3000 1 16,SF TC NE = ,I6)
      1 ENC
      SUBCLINE SETMEM(J)
C..... MCNITR AVAILABLE MENCY IN BLANK COMMON
C
COMMON N(1)
COMMON /PSIZE/ MAX
K = J
IF(K.LE.MAX) RETURN
WRITE(6,1000) K,MAX
STOP
1000 1 17X,11FREQUIRED =,I8/17X,11HAVAILABLE =,I8/)
      1 ENC
      SUBROUTINE SPHERE (X,NCM,PRT)
      IMPLICIT REAL*8 (A-F,C-Z)
C..... CCNVERT SPHERICAL TC CARTESIAN COORDINATES
C
      LOGICAL FRT
      DIMENSION X(NDM,1)
      COMMON /CCATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
      IF(NCM.LT.3) RETURN
      MCT = C
      TH = CATAN(1.0D0)/45.CDC
      READ(5,1000) NI,NE,INC,XO,YO,ZO
      IF(NI.LE.C) RETURN
      IF(NI.GT.NUMNP.OR.NE.GT.NUMNP) GO TO 300
      INC = ISIGN(MAXO(IABS(INC),1),NE-NI)
      N = NI
      R = X(1,N)
      ANGZ = X(2,N)*TH
      ANGZ = X(3,N)*TH
      X(1,N) = X(3,N)*R*DSIN(ANGZ)*DCOS(ANGX)
      X(2,N) = X(3,N)*R*DSIN(ANGZ)*DSIN(ANGX)
      X(3,N) = X(3,N)*R*DCOS(ANGZ)
      MCT = MCT - 1
      IF(MCT.GT.0) GO TO 250
      IF(PRT) WRITE(6,2000) C,HEAD,XO,YO,ZO,(I,I=1,NDM)
      MCT = 50

```



```

250 IF(PRT) WRITE(6,2001) N,(X(I,N),I=1,NCM)
    N = N + INC
    IF((NE-N)*INC.GE.0) CC TC 200
    IF(MOD(NE-NI,INC).EQ.0) GO TO 100
    NI = NE
    N = NE
    GO TO 200
C
300 ERROR
    WRITE(6,3000) NI,NE
    STOP
C
1000 FORMATS
2000 FORMAT(A1,20A4//5X,62HCARTESIAN COORDINATES COMPUTED FROM SPHERICA
    1L INPUT WITH X0 = ,E12.4,6H Y0 = ,E12.4,6H Z0 = ,E12.4
    2 //6X,4FNCDE,6(I7,6H-CCCRD))
    2001 FORMAT(11C,6F13.4)
    3000 FORMAT(5X,49F**FATAL ERROR 16** ATTEMPT TO CONVERT NODES NI = ,
        I6,5F TC NE = ,I6)
    1 END

```

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...00429
...00430
...00431
...00432
...00433
...00434
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...00437
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...00447
...00448
...00449

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```

C..... SUBROUTINE PMACIO(CT,WD,ENDM,ERR,IPR,LL,NE)
C..... IMPLICIT REAL*8 (A-H,C-Z)
C..... MACFC PROGRAM INPUT/OUTPUT ROUTINE

LOGICAL ERR,PCOMP
DIMENSION CT(4,1),WC(1)
READ MACFC CARDS
LL = 1
LMAX = 16
CALL SETMEN(NE+LMAX*4*IPR)
CT(1,1) = WC(7)
CT(3,1) = 1.0
LL = LL + 1
IF (LL.LT.LMAX) GO TO 110
LMAX = LMAX + 16
CALL SETMEN(NE+LMAX*4*IPR)
READ(5,1000) (CT(J,LL),J=1,4)
WRITE(6,2000) (CT(J,LL),J=1,4)
IF (.NOT. PCOMP(CT(1,LL),ENDM)) GO TO 100
CT(1,LL) = WC(8)
SET LOCIP MARKERS
NE = NE + LMAX*4*IPR
LX = LL - 1
GO TO 230
IF (.NOT. FCCMF(CT(1,L),WC(7))) GO TO 230
J = 1
K = 1 + 1
DO 210 I = K,LL
IF(PCOMP(CT(I,I),WD(7))) J = J + 1
IF(J.GT.9) GO TO 401
IF(PCCMF(CT(1,I),WD(8))) J = J - 1
IF(J.EC.C) GO TO 220
CONTINUE
GO TO 400
CT(4,I) = L
CT(4,L) = I
CONTINUE
J = C
DO 240 L = 1,LL
IF(PCCMF(CT(1,L),WD(7))) J = J+1
IF(PCCMF(CT(1,L),WD(8))) J = J-1
IF(J.EC.C) RETURN
WRITE(6,4000)
ERR = .TRUE.
WRITE(6,4001)
ERR = .TRUE.

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...00450
...00451
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```

1000 RETURN
2000 FORMAT(1X,A4,1X,A4,1X,2F10.0)
3000 FORMAT(10X,A4,1X,A4,1X,2G15.5)
4000 FORMAT(5X,46F**FATAL ERROR 10** UNBALANCED LCOP/NEXT MACROS )
4001 ENDC
4002 ENDC
4003 ENDC
4004 ENDC
4005 ENDC
4006 ENDC
4007 ENDC
4008 ENDC
4009 ENDC
4010 ENDC
4011 ENDC
4012 ENDC
4013 ENDC
4014 ENDC
4015 ENDC
4016 ENDC
4017 ENDC
4018 ENDC
4019 ENDC
4020 ENDC
4021 ENDC
4022 ENDC
4023 ENDC
4024 ENDC
4025 ENDC
4026 ENDC
4027 ENDC
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4029 ENDC
4030 ENDC
4031 ENDC
4032 ENDC
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4034 ENDC
4035 ENDC
4036 ENDC
4037 ENDC
4038 ENDC
4039 ENDC
4040 ENDC
4041 ENDC
4042 ENDC
4043 ENDC
4044 ENDC
4045 ENDC

SUBROUTINE PMACR (UL,XL,TL,LD,P,S,IE,D,ID,X,IX,F,T,JDIAG,B,DR,CT
1,NDF,NCA,NEN1,NST,NENC)
IMPLICIT REAL*8 (A-H,C-Z)

MACRO INSTRUCTION SUBPRCGRAM

CONTRCLS FRCBLEM SCLUTION AND OUTPUT ALGORITHMS BY
ORDER CF SPECIFYING MACRC COMMANDS IN ARRAY WD.

LOGICAL AFR,BFR,CFR,AFL,BFL,CFL,DFL,EFL,FFL,GFL,TFL,PCOMP
LOGICAL ERR,EER
COMMON /C1/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /C2/ AENGY
COMMON /C3/ PDIS(6),Z(6),BC(2),DI(6),CD(3),TE(3),FD(3)
COMMON /C4/ NOI,NC,N1,N2,N3,N4,N5,N6,A7,N8,N9,N10,N11,N12,N13
COMMON /C5/ PROP
COMMON /C6/ MAX
COMMON /C7/ TCATA,TIME,DT,C1,C2,C3,C4,C5
COMMON /C8/ WD(25),CT(4,1),CTL(4),LVS(9),LVE(9),JDIAG(1),
1 UL(1),XL(1),TL(1),D(1),P(1),S(1),IE(7,1),D(1),ID(1),X(1),
2 IX(1),F(1),B(1),CR(1)
1 DATA WD/4HTCL,4HDT,4HSTRE,4HDISP,4HTANG,4HFORM,4HLOOP,4HNEXT,
2 4HPROP,4HDATA,4HCONV,4HSCLV,4HLMAS,4HCMAS,4HMEST,
3 4HEICE,4HEXCC,4HUTAN,4HREAC,4HCHEC,
4 4HTIM/
DATA NWC/25/,ENDM/4HEND /,NV/1/,NC/1/,NA/1/
INTEGER CLCCK(6)
SET INITIAL VALUES OF PARAMETERS
CT = C.CCC
PROP = 1.0CCC
RNMAX = 0.0CC0
TIME = C.CC0
TOL = 1.0CC-9
UN = 0.0CC0
AFL = .TRUE.
BFL = .FALSE.
BFR = .FALSE.
C.....

```



[illegible]



```

C.....
19  WRITE(6,2003) O,HEAL,TIME,PRCP
    CALL FFTCIS(IC,X,B,F,NDM,NDF)
    GO TO 330
C.....
5   FORM TANGENT STIFFNESS
    IF(CFL) CALL PSETM(NC,NE,JDIAG(NEQ)*IPR,CFL)
    CALL PZERRC(M(NC),JDIAG(NEQ)*IPR)
    CFR = .TRUE.
    IF(J.EC.5) CFR = .FALSE.
    IF(GFL) CALL PSETM(NA,NE,JDIAG(NEQ)*IPR,GFL)
    IF(NFL) CT(0) PROP = PROPLD(TIME,0)
    CALL PZERRC(M(NA),JDIAG(NEQ)*IPR)
    CALL PFCRM(UL,XL,TL,LD,P,S,IE,D,ID,X,IX,F,I,JDIAG,DR,M(NA),M(NC),
2    NCF,NDM,NENI,NST,3,B,M(NV),.TRUE.,.FALSE.,CFR,.FALSE.,I)
    AFR = .TRUE.
    GO TO 330
C.....
6   FORM CLT CF BALANCE FORCE FOR TIME STEP/ITERATION
    IF(NPLD) CT(0) PROP = PROPLD(TIME,0)
    CALL PLCLCF(IC,F,DR,NNEG,PROP)
    CALL PFCRM(UL,XL,TL,LD,P,S,IE,D,ID,X,IX,F,I,JDIAG,DR,M(NA),M(NC),
2    NCF,NDM,NENI,NST,3,B,M(NV),.FALSE.,.FALSE.,.TRUE.,.FALSE.,I)
    BFR = .TRUE.
    RN = DSGRT(DCT(DR,DR,NEG))
    RNMAX = CMAX1(RNMAX,RN)
    WRITE(6,2005) RNMAX,RN,TCL
    IF(RN.GE.RNMAX*TOL) GC TC 330
    LX = LVE(LV)
    LO = LVS(LV)
    CT(3,LX) = CT(3,LO)
    L = LX - 1
    GO TO 330
C.....
7   SET LOCP START INDICATORS
    LV = LV + 1
    LX = CT(4,L)
    LVS(LV) = L
    LVE(LV) = LX
    CT(3,LX) = 1.000
    GO TO 330
C.....
8   LOOP TERMINATER CONTROL
    N = CT(4,L)
    NT(3,L) = CT(3,L) + 1.000
    IF(CT(3,N)) LV = LV - 1
    IF(CT(3,N)) L = N
    IF(CT(3,L)) LE = CT(3,N)
    GO TO 330
C.....
9   INPUT PROPORTIONAL LCAD TABLE
    NPLD = CT(3,L)
    PRCP = PROPLD(0.000,NPLD)
    GO TO 330
C.....
    READ COMMAND

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10 READ(5,1000) (CTL(I),I=1,4)
   IF(.NOT.(CTL(2,L),CTL(1))) GO TO 402
   IF(PCCMF(CTL(1),WC(1))) TCL = CTL(3)
   IF(PCCMF(CTL(1),WC(2))) CT = CTL(3)
   IF(PCCMF(CTL(1),WC(4))) GO TO 101
   GO TO 30
101 ISW = CT(3,L) ISW = 5
   IF(ISW.EQ.0) REWIND ISW
   IF(ISW.NE.5) TIME,(B(I),I=1,NEG)
   READ(15) TIME
   GO TO 30
C....
11 INCREMENT TIME
   TIME = TIME + DT
   INMAX = C.CDC
   RNMAX = 0.00
   GO TO 30
C....
12 COMPUTE CONVERGENCE TESTS
   RN = CCT(CR,DR,NEG)
   UN = UN + DGT(B,B,NEG)
   UN = MAX1(UN,RN)
   CN = DSCRT(RN)
   WRITE(6,2002) CN,RN,TOL
   LX = LVE(LV)
   LO = LVS(LV)
   IF(RN.LT.CN*TOL) CT(3,LX) = CT(3,LO)
   GO TO 30
C....
13 SOLVE THE EQUATIONS
   IF(CFR) CCT(131)
   CALL ACTCOL(M(NA),DR,JDIAG,NEG,AFR,BFR)
   IF(BFR) WRITE(6,2007) AENGY
   GO TO 12
131 CALL UACTCL(M(NA),M(NC),CR,JDIAG,NEG,AFR,BFR)
132 AFR = .FALSE.
   IF(.NOT.BFR) GO TO 30
   BFR = .FALSE.
   DO 133 N = 1,NEG
   B(N) = B(N) + DR(N)
133 GO TO 30
C....
14 FORM A LUMPED MASS APPROXIMATION
   AFL = .FALSE.
   BFL = .TRUE.
   IF(EFL) CALL PSETM(NA,NE,NEG*IPR,EFL)
139 CALL PZEEC (M(NN),NEG*IPR)
   GO TO 140
C....
15 FORM A CONSISTENT MASS APPROXIMATION
   AFL = .FALSE.

```



```

152 IF(DFL) CALL PSETM(NM,NE,JDIAG(NEQ)*IPR,DFL)
140 CALL PZERO(M(NM),JDIAG(NEQ)*IPR)
      CALL PFCRM(UL,XL,TL,LD,P,S,IE,D,ID,X,IX,F,T,JDIAG,M(NN),M(NM),
1      M(NV),ACF,ACM,NEN1,NST,5,B,M(NV),AFL,BFL,.FALSE.,.FALSE.,4)
      GO TC 330
C.....
16 CHANGE MESH DATA
      I = 1
      PMESH(LD,IE,D,ID,X,IX,F,T,NDF,NDM,NEN1,I)
      IF(I.GT.0) GO TO 404
      GO TC 330
C.....
17 COMPLETE DOMINANT EIGENVALUE BY INVERSE ITERATION
      J = NM
      IF(DFL) J = NN
      CALL PEIGS(M(NA),M(J),F,X,B,DR,ID,IX,JDIAG,NDF,NDM,NEN1,DFL)
      GO TC 330
18 IF(FFL) GO TO 181
C.....
      MACRC*EXCC* EXPLICIT INTEGRATION OF EQUATIONS OF MOTION
      NQ = NQ + MOD((NQ-1),IPR)
      NV = NV + NQ*IPR
      NR = NV + NDF*NUMNP*IFR
      NE = NR + NQ*IPR
      CALL SETMEM(NE+1)
      CALL PZERF(M(NQ),NE-NQ)
      FFL = .TRUE.
      GO TC 330
181 IF(.NOT.BFR.OR.EFL) GC TC 403
      CALL ELPCAT(DR,B,M(NC),M(NR),M(NN),DT,NEQ)
      GO TC 330
C.....
20 COMPLETE REACTIONS AND PRINT
      CALL PZERF(UL,XL,TL,LD,P,S,IE,D,ID,X,IX,F,T,JDIAG,DR,CR,CR,
1      CALL PFCRM(UL,XL,TL,LD,P,S,IE,D,ID,X,IX,F,T,JDIAG,DR,CR,CR,
      CALL NCF,ACM,NEN1,NST,6,B,M(NV),.FALSE.,.TRUE.,.FALSE.,1)
      GO TC 330
C.....
23 PRINT ARRAYS AS OUTPUT
      IF(PCOMP(CT(2,L),WD(5)),CR.PCOMP(CT(2,L),WC(15)))
1      WRITE (6,2006) (N,JDIAG(N),N=1,NEQ)
      IF(PCOMP(CT(2,L),WD(5)).AND.AFR) CALL PRITE(M(NA),JDIAG(NEQ),
1      SHSTIFFNESS)
      IF(PCOMP(CT(2,L),WD(6)).AND.BFR) CALL PRITE(DR,NEQ,
1      SHFORCE)
      IF(PCOMP(CT(2,L),WD(15)).AND.AFL) CALL PRITE(M(NM),JDIAG(NEQ),
1      15PCASISTENT MASS)
      IF(PCOMP(CT(2,L),WD(14)).AND.BFL) CALL PRITE(M(NN),NEQ,
1      11FLUMPCL MASS)
      GO TC 330
C.....
      CHECK MESH FOR ERRORS

```







```

3000  FORMAT(5X,29F**WARNING** MACRO INSTRUCTION  ,I4,
4002  18H IS NOT EXECUTABLE ),
4003  1FORMAT(5X,57F**FATAL ERROR 12** MACRO LABEL MISMATCH ON A READ CCM
4004  1MAND),
1FORMAT(5X,63F**FATAL ERROR 13** MACRO EXCD MUST BE PRECEDED BY LMA
1S AND FCRN)
1FORMAT(5X,84F**FATAL ERROR 14** ATTEMPT TO CHANGE BOUNDARY RESTRAI
1INT CCCE$ DURING MACRC EXECUTION )
END
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SUBROUTINE ACTCOL(A,B,JDIAG,NEQ,AFAC,BACK)
IMPLICIT REAL*8 (A-F,O-Z)
LOGICAL AFAC,BACK
COMMON/ENGYS/AENGY
DIMENSION A(1),B(1),JDIAG(1)
DATA TOL/1.D-7/

C.....
C      ACTIVE COLUMN PROFILE SYMMETRIC EQUATION SOLVER
C.....
FACTCR A TC UT*D*U, REDUCE B
AENGY = C.OOO
JR = 0
DO 600 J = 1,NEQ
  JD = JDIAG(J)
  JH = JC - JR
  IS = J - JH + 2
  CG = A(JD)
  IF(JH-2) 50,300,100
  IF(.NOT.AFAC) GO TO 500
  IE = JR - 1
  K = JR + 2
  ID = JDIAG(IS-1)
  REDUCE ALL EQUATIONS EXCEPT DIAGONAL
  DO 200 I = IS,IE
    IR = JC
    ID = JDIAG(I)
    IH = MINC(ID-IR-1,I-IS+1)
    IF(IF-GT.C) A(K) = A(K) - DOT(A(K-IH),A(ID-IT),IH)
    K = K + 1
  REDUCE DIAGONAL TERM
  IF(.NOT.AFAC) GO TO 500
  IR = JR + 1
  IE = JC - 1
  K = JR - JC
  DO 400 I = IR,IE
    ID = JDIAG(K+I)
    C = A(I)
    A(I) = A(I)*A(ID)
    A(JD) = A(JD) - D*A(I)
  CONTINUE
  IF(DG*A(JD).LT.O.OOO) WRITE(6,2000) J
  IF(DABS(A(JD)).LT.TOL*DABS(DG)) WRITE(6,2001) J
  IF(A(JD).EQ.O.OOO) WRITE(6,2002) J
  REDUCE RHS
  IF(BACK) E(J) = B(J) - DCT(A(JR+1),B(IS-1),JH-1)
  IF(A(JC).NE.O.OOO.ANC.AFAC) A(JD) = 1.OOO/A(JD)
  JR = JC
  IF(.NOT.EACK) RETURN

```



```

C.....
CIVICE BY DIAGONAL PIVOTS
CO 700 I = 1, NEQ
ID = JCIAG(I)
AENG = AENG + B(I)*B(I)*A(ID)
700 B(I) = E(I)*A(ID)
C.....
BACKSUBSTITUTE
JD = NEG
JD = JCIAG(J)
800 L = E(J)
J = J - 1
IF(J.LE.C) RETURN
JR = JCIAG(J)
IF(JC-JR.LE.1) GO TC 1000
IS = J - JD + 1
K = JR - IS + 1
DO SCO I = IS, J
B(I) = E(I) - A(I+K)*D
900 JR = JR - 1
1000 GO TC ECC
2000 FORMAT(5X, 31F**WARNING C1** SIGN OF EQUATION, I5, 18H CHANGED IN ACT
1CCL)
2001 FORMAT(5X, 23H**WARNING 02** EQUATION, I5, 45H LCST AT LEAST 7 SIGNIF
1ICANT DIGITS IN ACTCOL)
2002 FORMAT(5X, 32F**WARNING 03** PIVOT OF EQUATION, I5, 18H IS ZERO IN AC
1TCOL)
1CCL)
END

SUBROUTINE ADDSTF(A,B,C,S,P,JDIAG,LD,NST,NEL,AFL,BFL,CFL,ITP)
C
C.....
C
ASSEMBLE GLOBAL ARRAYS
IMPLICIT REAL*8 (A-F,G-Z)
LOGICAL AFL,BFL,CFL
DIMENSION A(1),B(1),JCIAG(1),P(1),S(NST,1),LC(1),C(1)
CO 200 J = 1, NEL
K = IAES(LC(J))
IF(K.EQ.C) GO TO 20C
IF(BFL) E(K) = B(K) + P(J)
IF(.NOT.AFL.AND..NOT.CFL) GC TO 200
L = JCIAG(K) - K
CO 100 I = 1, NEL
M = IAB$(LC(I))
IF(M.GT.K.CR.M.EQ.0) GO TO 100
N = L + M
IF(AFL) A(M) = A(M) + S(I,J)
IF(CFL) C(M) = C(M) + S(J,I)
CONTINUE
100

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[illegible]



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5  CALL ELMTCS(C,U,X,IX,T,S,P,I,J,K,ISW)
10 RETURN
400 WRITE(6,4000) IEL
STOP
4000 FORMAT(5),39H**FATAL ERRCR 04** ELEMENT CLASS NUMBER,I3,6H INPUT)
ENC

C.....
SUBROUTINE EUPDAT(DR,U,V,A,XM,DT,NEQ)
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION L(1),V(1),A(1),DR(1),XM(1)
UPDATE SOLUTION USING EXPLICIT CENTRAL DIFFERENCES
DATA CTFP/C.000/
CTF = CT/2.000
DTAV = CTH + DTHP
CTFP = CTF
DO 100 N = 1,NEQ
  A(N) = DR(N)/XM(N)
  V(N) = V(N) + DTAV*A(N)
  L(N) = L(N) + DT*V(N)
100 RETURN
ENC

SUBROUTINE NCRM(X,Y,N)
IMPLICIT REAL*8 (A-H,O-Z)
C.....
C      NORMALIZE Y TO UNIT VECTOR X
DIMENSION X(1),Y(1)
SCALE = DSQRT(DOT(Y,Y,N))
DO 100 I = 1,N
  X(I) = Y(I)/SCALE
100 RETURN
END
N

SUBROUTINE PADM(A,B,C,NN)
IMPLICIT REAL*8 (A-H,O-Z)
C.....
C      ADD LINEAR COMBINATION OF TWO ARRAYS TOGETHER
DIMENSION A(1),B(1)
DO 100 N=1,NN
  A(N) = A(N) + C*B(N)
100 RETURN
END
  
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SUBROUTINE PADDV(A,B,ID,C,NN)
IMPLICIT REAL*8 (A-F,C-Z)
DIMENSION A(1),B(1),ID(1)
DO 100 N=1,NN
K = ID(N)
IF(K.GT.C) A(N) = A(N) + C*B(K)
CONTINUE
RETURN
END

```

100

```

SUBROUTINE PADD(A,B,JD,C,NN)
IMPLICIT REAL*8 (A-F,C-Z)

```

C..... ADD LINEAR COMBINATION OF DIAGONAL ARRAY TO DIAGONAL OF AN ARRAY  
C

```

DIMENSION A(1),B(1),JD(1)
DO 100 N=1,NN
J = JD(N)
A(J) = A(J) + C*B(N)
RETURN
END

```

100

```

SUBROUTINE PSAL(A,C,NN)
IMPLICIT REAL*8 (A-F,C-Z)

```

SCALE AN ARRAY BY A CONSTANT

```

DIMENSION A(1)
DO 100 N=1,NN
A(N) = C*A(N)
RETURN
END

```

100

```

SUBROUTINE PRODIA(A,B,C,NEG)
IMPLICIT REAL*8 (A-F,C-Z)

```

ROUTINE TO FORM C = C + A\*B WHERE A IS A DIAGONAL MATRIX,  
B AND C ARE VECTORS

```

DIMENSION A(1),B(1),C(1)
DO 100 N=1,NEG
C(N) = C(N) + A(N)*B(N)
RETURN
END

```

100



```

SUBROUTINE PRCPML(A,B,C,JDIAG,NEQ)
IMPLICIT REAL*8 (A-F,O-Z)
DIMENSION A(1),B(1),C(1),JDIAG(1)

ROUTINE TC FORM C = C + A*B WHERE A IS A SYMMETRIC SQUARE MATRIX
STORED IN PRCPFILE FCFM,B,C ARE VECTORS, AND JCIAG LOCATES THE
DIAGONALS IN A

      JS = 1
      CO 200 JJ=1,NEQ
      JD = JCIAG(J)
      IF(JS.GT.JD) GO TO 2CO
      EJ = E(J)
      AB = A(JD)*BJ
      IF(JS.EC.JD) GO TO 150
      JB = J - JC
      JE = JC - I
      CO 100 JJ = JS,JE
      AB = AE + A(JJ)*B(JJ+JB)
      C(JJ+JB) = C(JJ+JB) + A(JJ)*BJ
      C(J) = C(J) + AB
      JS = JC + 1
      RETURN
      ENCL

FUNCTION FRCPLO(T,J)
IMPLICIT REAL*8 (A-F,O-Z)

PROPCRTICNAL LOAD TABLE (CNE LOAD CARD ONLY)

DIMENSION A(5)
IF(J.GT.C) GC TO 20C
CGMPUTE VALUE AT TIME T
FRCPLO = C.GCO
IF(T.LT.TMIN.OR.T.GT.TMAX) RETURN
L = MAXO(L,I)
PROPLC = A(1) + A(2)*T + A(3)*(DSIN(A(4)*T+A(5)))*L
RETURN
INPUT TABLE CF PROPCRTICNAL LOADS
I = 1
REAC(5,1000) K,L,TMIN,TMAX,A
WRITE(6,2CCO) I,K,L,TMIN,TMAX,A
RETURN
FORMAT(2I5,7F10.0)
FORMAT(5X,23HPROPCRTICNAL LCAD TABLE//24F NUMBER TYPE EXP.,)

```



```

1 14F MINIMUM TIME,15F MAXIMUM TIME, 5X,2HA1,13X,2HA2,13X,2HA3,
2 13X,2HA4,13X,2HA5/(3I8,7G15.5)
ENC

SUBROUTINE PRDIS(IC,X,B,F,NDM,NDF)
IMPLICIT REAL*8 (A-F,C-Z)

C..... CUTPUT ACCAL VALUES

LOGICAL FCGMP
COMMON/FFLCD/ PROCP
COMMON/CCATA/O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEG,IPR
COMMON/LABEL/ PDIS(6),A(6),BC(2),DI(6),CD(3),TE(3),FD(3)
COMMON/TCATA/ TIME,C1,C2,C3,C4,C5
DIMENSION X(NDM,1),B(1),(L(6),ID(NDF,1),F(NDF,1)
CATA BL/4+BLAN/
DO 102 I=1,NUMNP,50
WRITE(6,2000) O,HEAD,TIME,(I,CD(1),CD(2),I=1,NDM),(I,DI(1)
1 ,LI(2),I=1,NDF)
JJ = MINC(NUMNP,II+45)
DO 102 N=II,JJ,BL) GO TO 101
IF(PCCNF(X(I,N),BL)) GO TO 101
DO 100 I=1,NDF
UL(I) = F(I,N)*PROP
K = IABS(IC(I,N))
IF(K.GT.C) UL(I) = B(K)
WRITE(6,FCIS) N,(X(I,N),I=1,NDM),(UL(I),I=1,NDF)
CONTINUE
CONTINUE
RETURN
FORMAT(A1,20A4//5X,15HNOCAL DISPLACEMENTS,5X,4HTIME,E13.5//
6X,4HNOCE,9(I7,A4,A2))
1 ENC

SUBROUTINE PRITE (A,NN,WD)
IMPLICIT REAL*8 (A-F,O-Z)

C..... CUTPUT ARRAY

DIMENSION A(NN),WD(2)
WRITE(6,2000) WD
DO 100 I=1,NN,7
K = MINC(NN,I+6)
WRITE(6,2001) (J,A(J),J=I,K)
RETURN
FORMAT(/1CX,2A8,6H ARRAY /)
100
2000

```



```

2001  FORMAT(IX,7(I5,1PE12.4))
      END
      C.....
      C.....
      SUBROUTINE PROFIL (JDIAG,ID,IX,NDF,NEN1,NAC)
      IMPLICIT REAL*8 (A-T,C-Z)
      C.....
      C.....
      CCMPLTE PROFILE OF GLOBAL ARRAYS
      COMMON /CCATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
      DIMENSION JDIAG(1),IC(NDF,1),IX(NEN1,1)
      SET LP THE EQUATION NUMBERS
      NEQ=O
      DO 50 N = 1,NUMNP
      DO 40 I = 1,NDF
      J=ID(I,N)
      IF(J) =O,20,30
      NEQ= NEQ+1
      ID(I,N)=NEQ
      JDIAG(NEQ) = O
      GO TO 40
      ID(I,N) = O
      CONTINUE
      CCMPLTE COLUMN HEIGHTS
      DO 500 N = 1,NUMEL
      DO 400 I = 1,NEN
      II = IABS(IX(I,N))
      IF(II.EC.O) GO TO 4CC
      LO 300 K = 1,NDF
      KK = IC(K,II)
      IF(KK.EC.O) GO TO 3CC
      DO 200 J = 1,NEN
      JJ = IABS(IX(J,N))
      IF(JJ.EC.O) GO TO 200
      CO 100 L = 1,NDF
      LL = IC(L,JJ)
      IF(LL.EC.O) GO TO 100
      M = MAXO(KK,LL)
      JDIAG(M) = MAXO(JDIAG(M),IABS(KK-LL))
      CONTINUE
      CONTINUE
      CONTINUE
      CCMPLTE DIAGONAL PCINTERS FOR PROFILE
      NAD=1
      JDIAG(1) = 1

```



```

600      IF (NEC.EC.1) RETURN
        CO 6CO N = 2, NEQ
        JDIAG(N) = JDIAG(N) + JDIAG(N-1) + 1
        NAC = JDIAG(NEQ)
        RETURN
        END
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C.....
C      SUBROUTINE PLOAD(IC,F,B,NN,P)
C      IMPLICIT REAL*8 (A-F,D-Z)
C      FORM LCAC VECTOR IN CCMPACT FORM
C      DIMENSION ID(1),F(1),B(1)
C      CO 1CO N = 1,NN
C      J = IC(N)
C      IF (J.GT.C) B(J) = F(N)*P
C      RETURN
C      END
100
C.....
C      SUBROUTINE PFORM(UL,XL,TL,LD,P,S,IE,D,IC,X,IX,F,T,JDIAG,B,A,C,NDF,
C      1 NEM,NEN1,NST,ISW,U,UD,AFL,BFL,CFL,DFL,ITP)
C      IMPLICIT REAL*8 (A-F,C-Z)
C      COMPUTE ELEMENTS   ARRAYS AND ASSEMBLE GLOBAL ARRAYS
C      LOGICAL AFL,BFL,CFL,DFL
C      COMMON /CCATA/ O,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
C      COMMON /ELDATA/ DM,N,MA,ACT,IEL,NEL
C      COMMON /PLOAD/ PROP
C      DIMENSION XL(NDM,1),LC(NCF,1),P(1),S(NST,1),IE(7,1),D(10,1),ID(NCF,
C      1,1),X(NCM,1),IX(NEN1,1),F(NCF,1),JDIAG(1),B(1),C(1),UL(NCF,1)
C      2 LCCP,TL(1),T(1),U(1),CC(NCF,1)
C      LCCP CA BLEMENTS
C      IEL = C
C      CO 1IC N = 1,NUMEL
C      MA = I > (NEN1,N)
C      SET UP LOCAL ARRAYS
C      CO 1C8 I = 1,NEN
C      II = IABS(IX(I,N))
C      IF (I.NE.C) GO TO 1C5
C      TL(I) = C.CDO
C      DO 1C3 J = 1,NDM
C      XL(J,I) = 0.CDO
C      DO 1C4 J = 1,NDF
C      UL(J,I) = 0.CDO
C      UL(J,I+NEN) = 0.CDO
103

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104 LD(J,I) = 0
105 GO TC 108
106 IID = I1*NDF - NDF
107 NEL = I
108 TL(I) = T(II)
109 DO 106 J = 1,NDM
110 XL(J,I) = X(J,II)
111 CO 107 J = 1,NDF
112 JJ = IE(J,MA)
113 IF (J,LE,0) GO TO 107
114 K = IABS(IC(JJ,II))
115 UL(J,I) = F(JJ,II)*PROP
116 UL(J,I+NEN) = UD(JJ,II)
117 IF(K.GT.C) UL(J,I) = L(K)
118 IF(DFL) K = IID + JJ
119 LD(J,II) = K
120 IF(IX(I,N).GT.0) GO TO 108
121 IF(IX(I,N) = 1,NDF
122 CO 109 J = -LD(J,I)
123 LD(J,I) = -LD(J,I)
124 CONTINUE
125 FORM ELEMENT ARRAY
126 IF(IE(7,MA).NE.IEL) MCT = 0
127 IEL = IE(7,MA)
128 CALL ELMIB(D(1,MA),UL,XL,IX(1,N),TL,S,P,NCF,NDM,NST,ISW)
129 ADC TO TOTAL ARRAY
130 IF(AFL.CF.BFL.OR.CFL) CALL ADDSTF(A,B,C,S,P,JDIAG,LD,NST,NEL*NDF,
131 AFL,BFL,CFL,ITP)
132 CONTINUE
133 END FILE ITP
134 RETURN
135 END
136
137 SUBROUTINE PEIGS(A,B,F,X,Y,Z,ID,IX,JDIAG,NDF,NDM,NEN1,DFL)
138 IMPLICIT REAL*8 (A-F,C-Z)
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C.....
EIGP = 0.000
CALL ACTCCL(A,Z,JDIAG,NEG,..TRUE,..FALSE..)
DO 200 I=1,ITS
CALL PZFC(Z,NEG*IPR)
CALL PRCMUL(B,Y,Z,JCIAG,NEG)
RAYLEIGF GCOTIENT
EIG = AENGY/COT(Y,Z,NEG)
IF(DABS(EIG-EIGP).LT.TOL*DABS(EIG)) GC TC 300
CALL NCRM(Y,Z,NEG)
EIGP = EIG
INVERSE ITERATION
DO 200 I=1,ITS
CALL ACTCCL(A,Y,JDIAG,NEG,..FALSE,..TRUE..)
WRITE(6,2001) ITS
RETURN
300 WRITE(6,2000) O,HEAL,EIG,I
CALL NCRM(Z,Y,NEG)
CALL PRTCLS(ID,X,Z,F,NDM,NDF)
RETURN
2000 FORMAT(A1,20A4//5X,14HEIGENVALUE = ,G13.4/5X,14HITERATION = ,
1 I9/)
2001 FORMAT(5X,57H**FATAL ERRCR 09** NO CONVERGENCE IN EIGENVALUES, ITS
1 = ,I5)
END

C.....
SUBROUTINE PRTRER(R,NCF)
IMPLICIT REAL*8 (A-H,C-Z)
PRINT NCCAL REACTICNS

C.....
DIMENSION R(NDF,1),RSUM(6),ASUM(6)
COMMON /CCATA/ O,HEAL(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
DO 50 K=1,NDF
RSUM(K) = 0.000
ASUM(K) = 0.000
DO 100 N=1,NUMNP,50
J=MINC(NUMNP,N+49)
WRITE(6,2000) O,HEAL,(K,K=1,NDF)
DO 100 I=N,J,NDF
DO 75 K = 1,NDF
R(K,I) = -R(K,I)
RSUM(K) = RSUM(K) + R(K,I)
ASUM(K) = ASUM(K) + CABS(R(K,I))
WRITE(6,2001) I,(R(K,I),K=1,NDF)
PRINT STATICS CHECK
WRITE(6,2002) (RSUM(K),K=1,NDF)
WRITE(6,2003) (ASUM(K),K=1,NDF)
RETURN
2000 FORMAT(A1,20A4//5X,15HNODAL REACTIONS//6X,4FNODE,

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2001 1 6(I9,4F,CCF)}
2002 FORMAT(11C,6E13.4)
2003 FORMAT(/7X,3F-SUM,6E13.4)
2004 FORMAT(/3X,7F-ABS SUM,6E13.4)
2005 END

SUBROUTINE PSETM(NA,NE,NJ,AFL)
IMPLICIT REAL*8 (A-F,C-Z)
C.....
C..... SET FCINTER FOR ARRAYS
C.....
C..... LOGICAL AFL
C..... NA = NE
C..... NA = NA + MOD((NA-1),IPR)
C..... NE = NA + NJ
C..... AFL = .FALSE.
C..... CALL SETMEM(NE)
C..... RETURN
C..... END

SUBROUTINE PZERO(V,AN)
C.....
C..... ZER0 REAL ARRAY
C.....
C..... DIMENSION V(NN)
C..... DO ICC N = 1,NN
C..... V(N) = C.000
C..... RETURN
C..... END

SUBROUTINE UACTCL(A,C,B,JDIAG,NEQ,AFAC,BACK)
IMPLICIT REAL*8 (A-F,C-Z)
LOGICAL AFAC,BACK
DIMENSION A(1),B(1),JDIAG(1),C(1)
C.....
C..... UNSYMMETRIC, ACTIVE CCLUMN PROFILE EQUATION SOLVER
C..... FACTOR A TC UT*D*U, REDUCE B TO Y
C..... JR = C
C..... DO 300 J = 1,NEQ
C..... JD = JDIAG(J)
C..... JH = JE - JR
C..... IF(JH.LE.1) GO TO 300
C..... IS = J + 1 - JH
C..... IE = J - 1

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IF(.NOT.AFAC) GO TC 250
K = JR + 1
ID = 0
REDUCE ALL EQUATIONS EXCEPT DIAGONAL
DO 200 I = IS, IE
IR = IC
IH = JCIAG(I) - IR - 1, I - IS
IF(IH.EC.C) GO TO 150
A(K) = A(K) - DOT(A(K-IH), C(ID-IH), IH)
C(K) = C(K) - DOT(C(K-ID), A(ID-IH), IH)
IF(A(IL).NE.C.ODO) C(K) = C(K)/A(ID)
K = K + 1
REDUCE CIAGCNAL TERM
A(JD) = A(JD) - DOT(A(JR+1), C(JR+1), JH-1)
FORWARD REDUCE THE R.F.S.
IF(BACK) B(J) = B(J) - DCT(C(JR+1), B(IS), JT-1)
JR = JC
IF(.NOT.EACK) RETURN
BACKSUBSTITUTION
J = NE
JD = JCIAG(J)
IF(A(JC).NE.O.ODO) B(J) = B(J)/A(JD)
J = E(J)
J = J - 1
IF(J.LE.C) RETURN
JR = JCIAG(J)
IF(JC - JR.LE.1) GO TO 700
IS = J - JD + JR + 2
K = JR - IS + 1
DO 600 I = IS, J
B(I) = E(I) - A(I+K)*C
JD = JR
GO TC 500
END

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SUBROUTINE PCDEL (CT,XA,XC,XM,XN,U,F,JDIAG,ID,M,NE,NDF,AFR,CFR,DFR,...01406
,EFR,EER) ...01407
IMPLICIT REAL*8 (A-T,C-Z) ...01408
...01409
C..... FIRST ORDER ORDINARY DIFFERENTIAL EQUATION SOLVER ...01410
COMMON /CCATA/ O,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR ...01411
LOGICAL FCCMP,EER,AFR,CFR,DFR,EFR ...01412
DIMENSION F(1),U(1),JDIAG(1),M(1),XA(1),XC(1),XM(1),XN(1),ID(1) ...01413
1 ,CT(1),WD(1) ...01414
DATA WC /4HINIT,4HLINE,4TQUAD/ ...01415
NNEQ = NCF*NUMNP ...01416
REWIND 5 ...01417
WRITE (5) (M(I),I=1,NE) ...01418
END FILE 5 ...01419
DO 100 J=1,3 ...01420
IF (FCCMP(CT(1),WD(J))) GO TO 110 ...01421
CONTINUE ...01422
EER = .TRUE. ...01423
RETURN ...01424
EER = .FALSE. ...01425
GO TO (1,2,3),J ...01426
DO 10 I=1,NNEQ ...01427
M(I) = IC(I) ...01428
M1 = NNEQ + 1 ...01429
M1 = M1 + MOD((M1-1),IPR) ...01430
CALL INIT (XA,XC,XM,XN,F,M,M(M1),U,JDIAG,NDF,NNEQ) ...01431
GO TO 3 ...01432
CALL PLINE (XA,XC,XM,XN,F,M,U,JDIAG,AFR,CFR,DFR,EFR) ...01433
CALL TERM (XA,XC,XM,XN,F,M,U,JDIAG,AFR,CFR,DFR,EFR) ...01434
GO TO 3 ...01435
M2 = 1 + NNEQ*IPR ...01436
M2 = M2 + MOD((M2-1),IPR) ...01437
CALL PGLAC (XA,XC,XM,XN,F,M,M(M2),U,JDIAG,AFR,CFR,DFR,EFR) ...01438
CALL TERM (XA,XC,XM,XN,F,M(M2),U,JDIAG,AFR,CFR,DFR,EFR) ...01439
REWIND 5 ...01440
READ (5) (M(I),I=1,NE) ...01441
RETURN ...01442
END ...01443
SUBROUTINE INIT (XA,XC,XM,XN,F,ID,UP1,U,JDIAG,NDF,NNEQ) ...01444
IMPLICIT REAL*8 (A-T,C-Z) ...01445
C..... INPUT INITIAL CONDITIONS ...01446
COMMON /CCATA/ O,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR ...01447
COMMON /TCATA/ TIME,CT,C1,C2,C3,C4,C5 ...01448
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COMMON /LCATA/ DUMAX,DUMIN,CCNTR
DIMENSION UI(3),XA(1),XC(1),XM(1),XN(1),F(1),UP1(1),U(1),ID(1),
1 JCIAG(1)
LOGICAL ERR,INIC,INPLT,AFR,CFR,DFR,EFR,CCNTR
DATA UI/4F INI,4HT CI,4HSPL /
NN = JCIAG(NEC)
READ (5,1000) C5,C1,C2,DCMAX,DUMIN
IF(C5.LE.0.0D0) C5 = 2.0C0/3.0D0
IF(C1.LE.C.CD0) C1 = 1.5C0
IF(C2.LE.C.CD0) C2 = .8DC
WRITE (6,1002) C5,C1,C2
CONTR = .TRUE.
IF(DUMAX.LE.0.0D0.OR.DUMIN.LE.0.0D0) CONTR = .FALSE.
IF(CCNTR) WRITE (6,1001) DUMAX,DUMIN
DO 9 I=1,ANEQ
UP1(I) = 0.0C0
CALL GENVEC (NDF,UP1,UI,.TRUE.,ERR,.FALSE.)
INPUT = .TRUE.
DC 10 I=1,ANEQ
K = IABS(IC(I))
IF(K.GT.0) U(K) = UP1(I)
CONTINUE 10
WRITE (10) (U(J),J=1,NEQ)
END FILE 10
RETURN
1000 FORMAT(5F1C.0)
1001 FORMAT(5X,8HDUMAX = ,1PG13.5,8HDUMIN = ,1PG13.5)
1002 FORMAT(5X,8FTHETA = ,1PG13.5,8HGAMMA = ,1PG13.5,7HBETA = ,1PG13.5)
END

SUBROUTINE PLINE (XA,XC,XM,XN,F,UP1,U,JCIAG,AFR,CFR,DFR,EFR)
IMPLICIT REAL*8 (A-F,G-Z)

C..... TWO PCINT SCHEME
C
COMMON /CADATA/ 0,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /TCATA/ TIME,CT,C1,C2,C3,C4,C5
DIMENSION XA(1),XC(1),XM(1),XN(1),F(1),UP1(1),U(1),JDIAG(1)
LOGICAL AFR,CFR,DFR,EFR
NN = JCIAG(NEQ)
REWIND 10
WRITE (10) (XA(J),J=1,NA)
END FILE 10
DO 11 I=1,ANEQ
UP1(I) = CT*F(I)
CC = 1.0C0/(C5*D1)
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...
IF(DFR) CALL PROMUL (XM,U,UP1,JDIAG,NEQ)
IF(EFR) CALL PRODIA (XN,U,UP1,NEQ)
CALL PFCAL (LPI,CC,NEQ)
CALL PRCMLL (XA,U,UP1,JDIAG,NEQ)
IF(DFR) CALL PADDM (XA,XN,CC,NN)
IF(EFR) CALL PADDD (XA,XN,JDIAG,CC,NEQ)
CALL ACTCOL (XA,UP1,JDIAG,NEQ,.TRUE.,.TRUE.)
READ (10)
REWIND 10
READ (10) (XA(J),J=1,NN)
RETURN
END

SUBROUTINE PQUAD (XA,XC,XM,XN,F,UM1,UP1,U,JDIAG,AFR,CFR,EFR)
IMPLICIT REAL*8 (A-F,C-Z)

C.....
C      THREE PCINT SCHEME

COMMON /CDATA/ O,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /TEATA/ TIME,CT,C1,C2,C3,C4,C5
DIMENSION XA(1),XC(1),XM(1),XN(1),F(1),UM1(1),U(1),UP1(1),JDIAG(1)
LOGICAL AFR,CFR,EFR,CONTR
REWIND 10
READ (10) (UM1(J),J=1,NEQ)
NN = JDIAG(NEQ)
REWIND 10
WRITE (10) (XA(J),J=1,NN),(F(J),J=1,NEQ)
END FILE 10
C3 = C2*CT
C4 = C1/C3
DO 10 I=1,NEQ
  LPI(I) = 1.0D0/C2*F(I)
  F(I) = ((2.0D0*C1-1.0D0)/C3)*U(I)
  F(I) = F(I) + ((1.0D0-C1)/C3)*UM1(I)
CONTINUE
IF(DFR) CALL PROMUL (XM,F,UP1,JDIAG,NEQ)
IF(EFR) CALL PRODIA (XN,F,UP1,NEQ)
DO 11 I=1,NEQ
  F(I) = (2.0D0+(-.5D0-C1)/C2)*U(I)
  F(I) = F(I) + ((C1-.5D0)/C2-1.0D0)*UM1(I)
CONTINUE
CALL PFCMLL (XA,F,UP1,JDIAG,NEQ)
IF(DFR) CALL PADDM (XA,XN,C4,NN)
IF(EFR) CALL PADDD (XA,XN,JDIAG,C4,NEQ)
CALL ACTCOL (XA,UP1,JDIAG,NEQ,.TRUE.,.TRUE.)
REWIND 10
READ (10) (XA(J),J=1,NN),(F(J),J=1,NEQ)
RETURN

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END

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SUBROUTINE TERM (XA,XC,XM,XN,F,UP1,U,JDIAG,AFR,CFR,DFR,EFR)
IMPLICIT REAL*8 (A-H,C-Z)
COMMON /CCATA/ O,HEAD(20),NUMNP,NUMEL,NLMMAT,NEN,NEQ,IPR
COMMON /TCATA/ TIME,LT,C1,C2,C3,C4,C5
DIMENSION XA(1),XC(1),XM(1),XN(1),F(1),U(1),UP1(1),JDIAG(1)
COMMON /UCATA/ DUMAX,DUMIN,CONTR
LOGICAL AFR,CFR,DFR,EFR,CCNTR
IF(.NOT.CCNTR) GO TO 3
LN = 0.000
DO 11 JJ = 1,NEQ
UN = UN + (UP1(JJ)-U(JJ))*(UP1(JJ)-U(JJ))
UN = DSQRT (UN)
K = 1
IF(UN.GT.DUMAX) K = 1
IF(UN.LT.DUMIN) K = 2
GO TO (1,2,3),K
LT = LT/2.000
CALL PLINE (XA,XC,XM,XN,F,UP1,U,JDIAG,AFR,CFR,DFR,EFR)
GO TO 10
DT = DT*2.000
CALL PLINE (XA,XC,XM,XN,F,UP1,U,JDIAG,AFR,CFR,DFR,EFR)
REWINC 10
WRITE (IC) (L(J),J=1,NEQ)
END FILE 10
DO 14 I=1,NEQ
L(I) = UP1(I)
WRITE(6,1000) DT
FORMAT(1CX,4FDT =,1FG13.6)
RETURN
END

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SUBROUTINE ELMT02(D,LL,XL,IX,TL,S,P,NDF,NCM,NST,ISW)
IMPLICIT REAL*8 (A-H,C-Z)
C.....
C      TWO DIMENSIONAL HEAT TRANSFER ELEMENT
C.....
COMMON /CCATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON/ELCATA/ DM,N,MA,MCT,IEL,NEL
DIMENSION C(1),UL(1),XL(NCM,1),IX(1),TL(1),S(NST,1),P(1),
1SHP(3,1),WLAB(4)
DATA WLAB/4H PLA,4HNE ,4HAXIS,4HYM /
C.....
TRANSFER TO CORRECT PROCESSOR
GO TO (1,2,3,2,5,3),ISW
C.....
C      INPUT MATERIALS PROPERTIES
C.....
NLBC = 0
READ(5,1000) D(1),D(2),D(3),D(4),D(5),NGP,KAT,NLBC,INOL
IF(NCF.LE.0.OR.NGP.GT.6) NGP = 4
IF(KAT.NE.2) KAT = 1
WRITE(6,2000) D(1),D(2),D(3),D(4),D(5),NGP,NLBC
1C(3) = L(3)*C(4)
IF(KAT.EQ.2) WRITE(6,2001) WLAB(3),WLAB(4)
IF(KAT.EQ.1) WRITE(6,2001) WLAB(1),WLAB(2)
IKX = 0
IKY = C
IRCC = C
IQ = 0
IF(INOL.EQ.0) GO TO 11
READ(5,1001) IKX,IKY,IRCC,IQ
IF(IKX.NE.0) CALL PKX (T,D(1),1)
IF(IKY.NE.0) CALL PKY (T,D(2),1)
IF(IRCC.NE.0) CALL FRCC (T,D(3),1)
IF(IQ.NE.0) CALL PQ (T,D(5),1)
CONTINUE
IF(NLBC.NE.0) CALL ECCND2 (UL,XL,IX,S,P,NDF,ADM,NST,NLBC,NGP,1)
RETURN
C.....
C      INSERT CHECK OF MESH IF DESIRED
C.....
RETURN
C.....
C      COMPLETE CONDUCTIVITY (STIFFNESS) MATRIX
C.....
DO 102 I=1,NGP
CALL FGALSS(SG,WS,NGF,I1)
DO 102 JJ=1,NGP
CALL FGALSS(TG,WT,NGF,JJ)
WG = WS*WT
CALL SHAPE(SG,TG,XL,SHP,XSJ,NDM,NEL,IX,.FALSE.)
IF(IKX.EQ.0.AND.IKY.EQ.0.AND.IQ.EQ.0) GO TO 113

```



```

111 T = C*CO
    TO 111 KK=1,NEL
    IF(SHP(3,KK)*UL(KK)
    IF(IKX.NE.O) CALL PKX(T,C(1),2)
    IF(IKY.NE.C) CALL PKY(T,C(2),2)
    IF(IQ.NE.C) CALL PQ(T,D(5),2)
113 CONTINUE
    IF(KAT.NE.2) GO TO 101
    RR = C*CC
    CO 100 J = 1,NEL
    RR = RR + SHP(3,I)*XL(1,I)
100 XSJ = XSJ*RR
101 CV = XSJ*KG
    DO 102 J = 1,NEL
    SHJ = SFF(3,J)*DV
    A1 = D(1)*SHP(1,J)*CV
    A2 = D(2)*SHP(2,J)*DV
    P(J) = D(5)*SFJ
    DO 102 I = 1,NEL
    S(I,J) = S(I,J) + A1*SHP(1,I) + A2*SHP(2,I)
102 IF(NLBC.NE.O) CALL ECCND2 (UL,XL,IX,S,P,NCF,NDM,NST,NLBC,NGP,2)
    CO 106 I = 1,NEL
    CO 106 J = 1,NEL
    P(I) = P(I) - S(I,J) * UL(J)
    IF(ISW.EQ.3) RETURN
106 CO 116 I = 1,NEL
    CO 116 J = 1,NEL
    S(I,J) = C*ODO
116 C.....
    C.....
    5 COMPUTE HEAT CAPACITY (MASS) MATRIX
    CO 105 II = 1,NGP
    CALL PGALSS(SG,WS,NGF,II)
    CO 105 JJ = 1,NGP
    CALL PGALSS(TG,WT,NGF,JJ)
    WG = WS*WT
    CALL STAPE(SG,TG,XL,SF-P,XSJ,NDM,NEL,IX,.FALSE.)
    IF(IRCC.EQ.O) GO TO 114
    T = C*CCC
    DO 112 KK=1,NEL
    T = T + SHP(3,KK)*UL(KK)
    CALL PRCC(T,D(3),2)
112 CONTINUE
114 IF(KAT.NE.2) GO TO 104
    RR = O*OCC
    CO 103 I = 1,NEL
    RR = RR + SHP(3,I)*XL(1,I)
103 XSJ = XSJ*RR
104 DV = XSJ*KG

```



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...01675
...01680
...01681
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...01726

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```

105  CO 1 C5 J = 1,NEL
      SHJ = C(2)*SF-P(3,J)*CV
      IF(I SW .EC.5) P(J) = P(J) + SHJ
      DO 105 I = 1,NEL
      S(I,J) = S(I,J) + SF-J*SHF(3,I)
      IF(I SW .EC.5) RETURN
      DO 601 I = 1,NEL
      DO 601 J = 1,NEL
      P(I) = P(I) - S(I,J)*TL(J)
      RETURN
C
C
1000  FORMAT(5F10,C,4I5)
1001  FORMAT(5X,26HHEAT CONDUCTION ELEMENT // 7X,
2000  12HFCONDUCTIVITY ,5X,4HKX = ,1PG15.7, 5X,4HKY = ,1PG15.7,
      1 7X,19HSPCIFIC HEAT ,1PG15.7,7X,
      2 19HHEAT GENER/UNIT VOL,1PG15.7,7X,
      3 19HHEAT GAUSS PTS/CIR,7X,
      4 I3,15H LINES WITH SPECIFIED BOUNDARY CONCTIONS)
      5 I3,42H
      6 FORMAT(6X,2A4,9H ANALYSIS )
2001  END

SUBROUTINE SHAPE(SS,TT,X,SHP,XSJ,NDM,NEL,IX,FLG)
IMPLICIT REAL*8 (A-H,C-Z)

C.....
C      SHAPE FUNCTION ROUTINE FOR TWO DIMENSIONAL ELEMENTS

LOGICAL FLG
COMMON /JAC2L/ XS
DIMENSION SHP(3,1),X(NDM,1),S(4),T(4),XS(2,2),SX(2,2),IX(1)
DATA S/-C.500,0.500,0.500,-0.500/,T/-0.500,-0.500,0.500,0.500/
FORM 4-ACDE QUADRILATERAL SHAPE FUNCTIONS
CO 100 I = 1,4
SHP(1,I) = (0.500+S(I)*SS)*(0.500+T(I)*TT)
SHP(2,I) = S(I)*(0.500+T(I)*TT)
SHP(3,I) = T(I)*(0.500+S(I)*SS)
IF(NEL .GE. 4) GO TO 120
FORM 1-ACDE QUADRILATERAL SHAPE FUNCTIONS
CO 100 I = 1,3
SHP(1,I) = SHP(I,3)+SHP(I,4)
SHP(2,I) = SHP(I,3)
SHP(3,I) = SHP(I,3)
ADD QUADRATIC TERMS IF NECESSARY
IF(NEL .GT. 4) CALL SHAPE2(SS,TT,SHP,IX,NEL)
CONSTRUCT JACOBIAN AND ITS INVERSE
CO 120 I = 1,NDM
DO 120 J = 1,2
DO 120 J = 1,2

```



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...017772
...017773
...017774

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```

130 XS(I,J) = C.ODO
    CO I=0 K = 1,NEL
    XS(I,J) = XS(I,J) + X(J,K)*SHP(I,K)
    XSJ = XS(1,1)*XS(2,2)-XS(1,2)*XS(2,1)
    IF(FLG) RETURN
    SX(1,1) = XS(2,2)/XSJ
    SX(2,2) = XS(1,1)/XSJ
    SX(1,2) = -XS(1,2)/XSJ
    SX(2,1) = -XS(2,1)/XSJ
    FORM GLREAL DERIVATIVES
    CO I=0 J = 1,NEL
    TP SHP(2,1) = SHP(1,1)*SX(1,1)+SHP(2,1)*SX(1,2)
    SHP(1,1) = TP
    SHP(1,1) = TP
    RETURN
END
140
C.....

```

```

SUBROUTINE SHAP2(S,T,SPP,IX,NEL)
IMPLICIT REAL*8 (A-H,C-Z)
C.....
C      ADD QUADRATIC FUNCTIONS AS NECESSARY

```

```

    DIMENSION IX(1),SHP(3,1)
    S2 = (1.ODO-S*S)/2.ODO
    T2 = (1.ODO-T*T)/2.ODO
    CO I=0 J = 1,3
    CO I=0 J = 1,3
    SHP(J,I) = 0.ODO
    NICESICE ACCES (SERENCIPITY)
    IF(IX(1)) = EQ.0) GO TO 101
    SHP(1,1) = -S*(1.ODO-T)
    SHP(2,1) = -S2*(1.ODO-T)
    IF(NEL:LT.6) GO TO 107
    IF(IX(2)) = EC.0) GO TO 102
    SHP(1,6) = -T*(1.ODO+S)
    SHP(2,6) = T2*(1.ODO+S)
    IF(NEL:LT.7) GO TO 107
    IF(IX(7)) = EC.0) GO TO 103
    SHP(1,7) = -S*(1.ODO+T)
    SHP(2,7) = S2*(1.ODO+T)
    IF(NEL:LT.8) GO TO 107
    IF(IX(8)) = EC.0) GO TO 104
    SHP(1,8) = -T2
    SHP(2,8) = -T*(1.ODO-S)

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C.....
104 SHP(3,8) = T2*(1.0DC-S)
    INTERIOR NODE (LAGRANGIAN)
    IF(IX(9).EQ.0) GO TC 107
    SHP(1,9) = -S*T2
    SHP(2,9) = -T*S2
    SHP(3,9) = 4.0DO*S2*T2
    CORRECT EDGE NODES FOR INTERIOR NODE (LAGRANGIAN)
    DO 106 J = 1,4
    CO 105 I = 1,4
    SHP(J,I) = SHP(J,I) - 0.25DO*SHP(J,9)
    CO 106 I = 5,8
    IF(IX(I).NE.0) SHP(J,I) = SHP(J,I) - 0.5DO*SHP(J,9)
    CORRECT CORNER NODES FOR PRESENCE OF MIDSIDE NODES
    K = 8
    CO 109 I = 1,4
    L = I + 4
    CO 108 J = 1,3
    SHP(J,I) = SHP(J,I) - 0.5E0*(SHP(J,K)+SHP(J,L))
    K = L
    RETURN
    END

SUBROUTINE BCOND2 (UL,XL,IX,S,P,NDF,NCM,NST,NLBC,NGP,IFLAG)
IMPLICIT REAL*8 (A-H,O-Z)
COMMON /CCATA/ O,HEAC(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /ELDATA/ DM,N,MA,MCT,IEL,NEL
DIMENSION K(1),XL(NCM,1),IX(1),S(NST,1),P(1),SHP(3,9)
DIMENSION K(50),KBCCND(50),KLINE(50),PROPE(50,2)
GO TC (1,2),IFLAG

C.....
1 INPUT LINE BOUNDARY CONDITIONS
WRITE(6,2000) O,HEAC
KBMAX = C
DO 10 I = 1,NLBC
    READ(5,1000) KEL(I),KBCOND(I),KLINE(I),PROPB(I,1),PROPB(I,2)
    KBMA = MAXC(KBMAX,KBCCND(I))
    WRITE(6,2001) KEL(I),KBCCND(I),KLINE(I),PROPE(I,1),PROPB(I,2)
    IF(KBMAX.GE.4) CALL CCNV(T,COEFF,I)
    RETURN

C.....
2 ADD LINE BOUNDARY CONDITIONS CONTRIBUTIONS
    TO CONDUCTIVITY MATRIX AND LOAD VECTOR
    CO 30 LL=1,NLBC
    IF(KEL(LL).NE.N) GC TC 30
    KBT = KBCCND(LL)
    KBL = KLINE(LL)

```



```

C.....
CONST = PROPB(LL,1)
TEMP = FFCPE(LL,2)
KBLABS = IABS(KBL)
C2 = CFLCAT(KBL/KBLABS)
DO 30 I=1,NGP
CALL PGALSS (C1,W1,NGP,I)
WG = W1
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C.....
SELECT CORRECT INTEGRATION POINTS
GO TC (21,22),KBLABS
SG = C2
TG = C1
GO TC 24
SG = C1
TG = C2
GO TC 24
CALL SHAPE (SG,TG,XL,SHP,DUM,NDM,NEL,IX,.TRUE.)
CALL JACEE2 (XSJ,KELAES)
DS = WG*XSJ
CCEFF = CCNST
GO TC (25,26,27,28),KET
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C.....
FLUX BCLNARY CONDITION
DO 251 L=1,NEL
P(L) = F(L) - CCEFF*SHP(3,L)*DS
GO TC 30
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C.....
RADIATION BOUNDARY CCNDITION
T = 0.000
DO 271 I=1,NEL
T = T + SHP(3,I)*UL(I)
CCEFF = CCEFF*(T*T + TEMP*TEMP)*(T + TEMP)
GO TC 26
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C.....
CONVECTION BOUNDARY CCNDITION (TEMP. DEPEDENT)
T = C.CC
DO 281 I=1,NEL
T = T + SHP(3,I)*UL(I)
CALL CCNV(T,CCEFF,2)
GO TC 26
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C.....
CONVECTION BOUNDARY CCNDITION
DO 261 J=1,NEL
A = SHP(3,J)*DS
P(J) = P(J) + CCEFF*TEMP*A
DO 261 I=1,NEL
S(I,J) = S(I,J) + CCEFF*SHP(3,I)*A
CONTINUE
RETURN
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IF(IFLAG.EQ.1) WRITE (6,2000) C,HEAD
CALL TABLE (TEMP,COEFF,T,C,NDATA,IFLAG)
RETURN
2000 1 5X,30F TEMPERATURE
1 END
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C.....
C
SUBROUTINE PROC (T,C,IFLAG)
IMPLICIT REAL*8 (A-F,O-Z)
CALCULATE HEAT CAPACITY
COMMON /CDATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
DIMENSION TEMP(50),CCEF(50)
IF(IFLAG.EQ.1) WRITE (6,2000) O,HEAD
CALL TABLE (TEMP,CCEF,T,C,NDATA,IFLAG)
RETURN
2000 1 5X,30F TEMPERATURE
1 END
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C.....
C
SUBROUTINE PC (T,C,IFLAG)
IMPLICIT REAL*8 (A-H,C-Z)
CALCULATE INTERNAL HEAT GENERATION
COMMON /CDATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
DIMENSION TEMP(50),CCEF(50)
IF(IFLAG.EQ.1) WRITE (6,2000) O,HEAD
CALL TABLE (TEMP,CCEF,T,C,NDATA,IFLAG)
RETURN
2000 1 5X,30F TEMPERATURE
1 END
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C.....
C
SUBROUTINE CONV (T,C,IFLAG)
IMPLICIT REAL*8 (A-F,C-Z)
CALCULATE HEAT TRANSFER COEFFICIENT
COMMON /CDATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
DIMENSION TEMP(50),CCEF(50)
IF(IFLAG.EQ.1) WRITE (6,2000) O,HEAD
CALL TABLE (TEMP,CCEF,T,C,NDATA,IFLAG)
RETURN
2000 1 5X,30F TEMPERATURE
1 END
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2000  FORMAT(A1,20A4,/,5X,24HHEAT TRANSFER COEF. DATA//,
1      5X,30F TEMPERATURE
      END
      SUBROUTINE TABLE (XX,YY,X,Y,N,K)
      IMPLICIT REAL*8 (A-F,O-Z)
      C..... CALCULATE Y =TABLE(X) BY LINEAR INTERPOLATION
      C
      DIMENSION XX(1),YY(1)
      GO TO (1,2),K
      READ (5,1000) N
      DO 1 I=1,N
      READ (5,1001) XX(I),YY(I)
      WRITE(6,2001) XX(I),YY(I)
      RETURN
      IF (X.LT.XX(N).AND.X.GT.XX(1)) GO TO 21
      IF (X.GE.XX(N)) Y = YY(N)
      IF (X.LE.XX(1)) Y = YY(1)
      RETURN
      DO 2 I=1,N
      IF (X.LE.XX(I)) GO TO 23
      CONTINUE
      X1 = XX(I) - X
      X2 = XX(I-1) - X
      Y1 = YY(I)
      Y2 = YY(I-1)
      Y = (X1*Y2 - X2*Y1)/(X1 - X2)
      RETURN
      FORMAT(I5)
      1000  FORMAT(2F10.0)
      1001  FORMAT(5X,2(1PG15.7,2X))
      2001  END

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SUBROUTINE ELMT03 (C,UL,XL,IX,TL,S,P,NCF,NDM,NST,ISW)
IMPLICIT REAL*8 (A-F,C-Z)

C.....
C      THREE DIMENSIONAL HEAT TRANSFER ELEMENT
C
C      MATERIAL PROPERTIES
C      D(1) CONDUCTIVITY IN X DIRECTION
C      D(2) CONDUCTIVITY IN Y DIRECTION
C      D(3) CONDUCTIVITY IN Z DIRECTION
C      D(4) SPECIFIC HEAT
C      D(5) MASS DENSITY
C      D(6) HEAT GENERATION PER UNIT VOLUME
C
C      NGP      NUMBER OF INTEGRATION POINTS
C      WGT      INTEGRATION WEIGHT
C      KAT      =2 FOR AXISYMMETRY
C              =1 FOR PLANE GEOMETRY
C      NSBC     NUMBER OF SURFACES WITH SPECIFIED
C              BOUNDARY CONDITIONS
C
COMMON /CADATA/ O,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
COMMON /ELDATA/ OM,N,MA,NCT,IEL,NEL
DIMENSION C(1),UL(1),XL(NCM,1),IX(1),TL(1),S(NST,1),P(1),
1 SHP(4,21),WLAB(4)
C      CATA WLAB /4/ PLA,4FNE ,4HAXIS,4HVM /

C.....
C      TRANSFER TO CORRECT PROCESSOR
C      GO TC (1,2,3,2,5,3), ISW

C.....
C      INPUT MATERIAL PROPERTIES
C      NSBC = 0
C      READ(5,1000) D(1),D(2),D(3),D(4),D(5),D(6),NGP,KAT,NSBC,INOL
C      IF(NGP.LE.0.OR.NGP.GT.6) NGP = 4
C      WRITE(6,2000) D(1),D(2),D(3),D(4),D(5),D(6),NGP,NSBC
C      D(4) = C(4)*D(5)
C      IF(KAT.NE.2) KAT = 1
C      IF(KAT.EQ.1) WRITE(6,2001) WLAB(1),WLAB(2)
C      IF(KAT.EQ.2) WRITE(6,2001) WLAB(3),WLAB(4)
C      IF(INCL.EQ.0) GO TC 11
C      READ(5,1001) IKX,IKY,IKZ,IROC,IQ
C      IF(IKX.NE.0) CALL PKX (T,D(1),1)
C      IF(IKY.NE.0) CALL PKY (T,D(2),1)
C      IF(IKZ.NE.0) CALL PKZ(T,C(3),1)
C      IF(IRGC.NE.0) CALL PRCC (T,D(4),1)
C      IF(IC.NE.0) CALL PC (T,D(6),1)
C      CONTINUE
C      IF(NSBC.NE.0) CALL BCCND3 (UL,XL,IX,S,P,NDF,NDM,NST,NSBC,NGP,1)
C      RETURN
C
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C

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C.....
2      INSERT CHECK CF MESH IF DESIRED
C      RETURN
C.....
3      COMPUTE CCNDUCTIVITY (STIFFNESS) MATRIX
      CO 1C2 II=1,NGP
      CALL FGALSS(RG,WR,NGF,II)
      CO 1C2 JJ=1,NGP
      CALL FGALSS(SG,WS,NGF,JJ)
      CO 1C2 KK=1,NGP
      CALL FGALSS(TG,WT,NGF,KK)
      WG=WF*WS*WT
      CALL SHAPED(RG,SG,TG,XL,SHP,XSJ,NDM,NEL,IX,.FALSE.)
      IF(IKX.EQ.0.AND.IKY.EQ.0.AND.IKZ.EQ.0) GO TO 112
      T = C.CCC
      CO 111 L=1,NEL
      T = T + SHP(4,L)*UL(L)
      IF(IKX.NE.0) CALL PKX(T,C(1),2)
      IF(IKY.NE.0) CALL PKY(T,C(2),2)
      IF(IKZ.NE.0) CALL PKZ(T,C(3),2)
      IF(IQ.NE.C) CALL PC(T,D(6),2)
      CONTINUE
      IF(KAT.NE.2) GO TO 101
      RR=KAT.QCO
      CO 100 I=1,NEL
      RR = RR + SHP(4,I)*XL(1,I)
      XSJ = XSJ*WG
      DV = XSJ*WG
      D11 = L(1)*DV
      D12 = L(2)*DV
      D13 = L(3)*DV
      CO 1C2 JJ=1,NEL
      CB11 = C11*SHP(1,J)
      CB21 = C12*SHP(2,J)
      CB31 = C13*SHP(3,J)
      P(J) = F(J) + SHP(4,J)*C(6)*DV
      CO 1C2 I = 1,NEL
      S(I,J) = S(I,J) + SHP(1,I)*DB11 + SHP(2,I)*DB21 + SHP(3,I)*DB31
      IF(NSB.C.NE.0) CALL BCCNDE(UL,XL,IX,S,P,NCF,ADM,NST,NSBC,NGP,2)
      CO 1C3 I=1,NEL
      P(I) = F(I) - S(I,J)*UL(J)
      RETURN
C.....
5      COMPUTE HEAT CAPACITY (MASS) MATRIX
      CO 5C2 II=1,NGP
      CALL FGALSS(RG,WR,NGF,II)
      CO 5C2 JJ=1,NGP
      CALL FGALSS(SG,WS,NGF,JJ)

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C... C... X(1,I) X NODAL COORDINATES
C... C... X(2,I) Y NCDAL COORDINATES
C... C... X(3,I) Z NCDAL COORDINATES

LOGICAL FLAG XS
COMMON /JAC/ SHP(4,1), X(NDM,1), R(8), S(8), T(8), XS(3,3), SX(3,3), IX(1)
DIMENSION SHP(4,1), X(1), R(8), S(8), T(8), XS(3,3), SX(3,3), IX(1)
DATA R/C.5D0,0.5D0,-0.5D0,-0.5D0,0.5D0,0.5D0,0.5D0,-0.5D0,0.5D0,-0.5D0/,
      T/C.5D0,0.5D0,0.5D0,0.5D0,0.5D0,-0.5D0,-0.5D0,-0.5D0,-0.5D0,-0.5D0/,
      IX/1,2

C... C... COMPUTE 8-NCDE RECTANGULAR PRISM SHAPE FUNCTIONS AND DERIVATIVES
C... C... IN NATURAL COORDINATES
DO 100 I = 1,8
  SHP(4,I) = (0.5D0+R(I)*RR)*(0.5D0+S(I)*SS)*(0.5D0+T(I)*TT)
  SHP(1,I) = R(I)*(0.5D0+S(I)*SS)*(0.5D0+T(I)*TT)
  SHP(2,I) = S(I)*(0.5D0+R(I)*RR)*(0.5D0+T(I)*TT)
  SHP(3,I) = T(I)*(0.5D0+R(I)*RR)*(0.5D0+S(I)*SS)
  IF(NEL.GE.8) GO TO 200
DEGENERATE FC RMS
C... C...
C... C... ADC QUADRATIC TERMS IF NECESSARY
C... C... IF(NEL.GT.8) CALL SHAP32(RR,SS,TT,SHP,IX,NEL)

CONSTRUCT JACOBIAN AND ITS INVERSE
DO 300 I = 1,NDM
  DO 300 J = 1,3
    XS(I,J) = C.0D0
    XS(I,J) = XS(I,J) + X(J,K)*SHP(I,K)
  IF(FLAG) RETURN
  XSJ = XS(1,1)*XS(2,2)*XS(3,3)
        +XS(1,2)*XS(2,1)*XS(3,3)
        -XS(1,3)*XS(2,2)*XS(3,1)
        -XS(1,1)*XS(2,3)*XS(3,2)
        -XS(1,2)*XS(2,3)*XS(3,1)
        +XS(1,3)*XS(2,1)*XS(3,2)
  IF(XSJ.GT.1.0D-13) GO TO 400
  WRITE(6,1000)
  STOP
1000 1 2 3 4 5
      XS(1,1) = (XS(2,2)*XS(3,3)-XS(3,3)*XS(2,2))/XSJ
      XS(2,1) = -(XS(1,2)*XS(3,3)+XS(3,3)*XS(1,2))/XSJ
      XS(3,1) = -(XS(1,3)*XS(2,2)+XS(2,2)*XS(1,3))/XSJ
      XS(1,2) = -(XS(1,1)*XS(3,3)-XS(3,3)*XS(1,1))/XSJ
      XS(2,2) = -(XS(1,2)*XS(3,1)+XS(3,1)*XS(1,2))/XSJ
      XS(3,2) = -(XS(1,3)*XS(2,1)+XS(2,1)*XS(1,3))/XSJ
      XS(1,3) = -(XS(1,1)*XS(2,3)+XS(2,3)*XS(1,1))/XSJ
      XS(2,3) = -(XS(1,2)*XS(3,2)+XS(3,2)*XS(1,2))/XSJ
      XS(3,3) = -(XS(1,3)*XS(2,3)+XS(2,3)*XS(1,3))/XSJ

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C.....
SX(2,2) = -(XS(1,1)*XS(2,2)-XS(2,1)*XS(1,2))/XSJ
SX(3,3) = (XS(1,1)*XS(2,2)-XS(2,1)*XS(1,2))/XSJ
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C.....
FORM GLCBAL DERIVATIVES
DO 500 I=1,NEL
TP1 = SHF(1,1)*SX(1,1) + SHP(2,1)*SX(1,2) + SHP(3,1)*SX(1,3)
TP2 = SHF(1,1)*SX(2,1) + SHP(2,1)*SX(2,2) + SHP(3,1)*SX(2,3)
SHP(3,1) = SHP(1,1)*SX(3,1) + SHP(2,1)*SX(3,2) + SHP(3,1)*SX(3,3)
SHP(1,1) = TP1
SHP(2,1) = TP2
RETURN
1000 FORMAT(5X,**FATAL ERROR ** NEGATIVE OR ZERO JACOBIAN DETERMINAN
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2000 FORMAT (A1,2CA4, //5X,12HSURFACE B.C.//VALUE TEMPERATURE)
2001 15X,52H ELEM B.C. SURF. PROPERTY VALUE
      FORMAT (5X,3(I5,2X),2(1PG15.7,2X))
      END
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C.....
SUBROUTINE PKZ (T,C,IFLAG)
IMPLICIT REAL*8 (A-F,C-Z)
C..... CALCULATE CCNDUCTIVITY (Z DIRECTION)
COMMON /CCATA/ D,HEAD(20),NUMNP,NUMEL,NUMMAT,NEN,NEQ,IPR
DIMENSION TEMP(50),COEF(50)
IF (IFLAG.EC.1) WRITE (6,2000) C,HEAD
CALL TABLE (TEMP,COEF,T,C,NDATA,IFLAG)
RETURN
2000 FORMAT(A1,2CA4, //5X,26HCNDUCTIVITY DATA (Z CIR.)//,
      15X,30F TEMPERATURE
      1 END
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C.....
SUBROUTINE JACBB3 (XSJ,KS)
IMPLICIT REAL*8 (A-H,C-Z)
C..... COMPUTE SURFACE JACCRIAN DETERMINANT
COMMON /JAC3S/ XS
DIMENSION XS(3,3)
GO TC 1,2,3),KS
C1 = XS(2,2)*XS(3,3) - XS(2,3)*XS(3,2)
C2 = XS(2,3)*XS(3,1) - XS(2,1)*XS(3,3)
C3 = XS(2,1)*XS(3,2) - XS(2,2)*XS(3,1)
GO TC 1,2,3)
D1 = XS(1,1)*XS(3,3) - XS(1,3)*XS(3,1)
D2 = XS(1,3)*XS(3,2) - XS(1,2)*XS(3,3)
D3 = XS(1,2)*XS(3,1) - XS(1,1)*XS(3,2)
GO TC 1,2,3)
C1 = XS(1,1)*XS(2,2) - XS(1,2)*XS(2,1)
C2 = XS(1,2)*XS(2,3) - XS(1,3)*XS(2,2)
C3 = XS(1,3)*XS(2,1) - XS(1,1)*XS(2,3)
XSJ = C3C3C1(D1*D1 + C2*D2 + D3*D3)
RETURN
10 END
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